

**PERFORMANCE AND LIFE PREDICTION MODEL FOR  
PHOTOVOLTAIC MODULE: EFFECT OF ENCAPSULANT  
CONSTITUTIVE BEHAVIOR**

BY  
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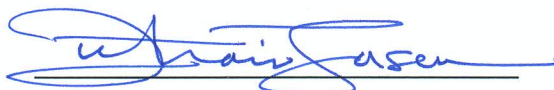
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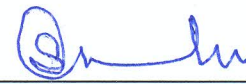
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*Dedicated to my parents, my brothers and my sister*

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*“Say, my prayer, my offering, my life and my death are for Allah, the Lord of all the world”* (Surat Al-‘An’am, verse 162)

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# TABLE OF CONTENTS

<b>LIST OF TABLES .....</b>	<b>xii</b>
<b>LIST OF FIGURES .....</b>	<b>xv</b>
<b>THESIS ABSTRACT (ENGLISH) .....</b>	<b>xx</b>
<b>THESIS ABSTRACT (ARABIC).....</b>	<b>xxi</b>
<b>CHAPTER 1</b>	
<b>INTRODUCTION.....</b>	<b>1</b>
1.1 Performance of a PV Module .....	3
1.2 Structure of a PV Module .....	4
1.3 Manufacturing of PV Modules .....	5
1.3.1 Silicon Purification .....	5
1.3.2 Types of Silicon Cells.....	5
1.3.3 The Czochralski Process .....	6
1.3.4 Doping.....	7
1.3.5 Anti-Reflective Coating, Back Surface Field and Busbar Pads.....	7
1.3.6 Assembling Process at BP-Solar, Riyadh .....	8
1.3.7 Summary of Pre-Stress within the Module.....	9
1.4 Loads During Operation .....	11
1.5 Failure Modes .....	12
1.5.1 Cracks in Cells .....	12
1.5.2 Breakage of Interconnects and Solder-bond Failures .....	13

1.5.3 Detachment of the Frame and Delamination .....	13
1.5.4 Glass Breakage.....	14
1.5.5 Encapsulant Degradation .....	15
1.5.6 Dominant Mode of Failure.....	16
1.6 Role of Encapsulant in PV Module Performance .....	17
1.7 Qualifying Standards .....	19
1.7.1 ASTM E1171-09 Temperature Cycle .....	19
1.7.2 Modified Standards .....	20
1.8 Motivation.....	21
1.9 Objectives of Proposed Work .....	22
<b>CHAPTER 2</b>	
<b>LITERATURE REVIEW .....</b>	<b>23</b>
2.1 Structural Modeling .....	23
2.2 Radiation and Optical Modeling .....	28
2.3 Thermal Modeling .....	28
2.4 Electrical Modeling.....	29
<b>CHAPTER 3</b>	
<b>COMPUTATIONAL MODEL FOR PV MODULES .....</b>	<b>31</b>
3.1 Basic Thermo-Mechanical Principles .....	31
3.2 Evaluation of Materials .....	35
3.2.1 Silicon .....	35
3.2.2 Glass.....	37

3.2.3 Back-Sheet .....	37
3.2.4 Interconnects .....	38
3.2.5 Encapsulant .....	40
3.3 Viscoelasticity .....	40
3.3.1 Maxwell's Model .....	41
3.3.2 DMA and Relaxation Tests.....	45
3.3.3 Time-Temperature-Superposition.....	47
3.4 Structural Modeling .....	49
3.4.1 Silicon .....	49
3.4.2 Copper.....	50
3.4.3 Glass and PVF/PET/PVF Composite .....	52
3.4.4 Ethylene-Vinyl Acetate.....	53
3.5 Radiation and Optical Modeling .....	56
3.5.1 Optical Model .....	56
3.5.2 Radiation Model.....	57
3.6 Thermal Modeling .....	58
3.7 Electrical Modeling.....	61
3.8 Geometric Modeling .....	62
3.9 Meshing.....	66



## **CHAPTER 4**

### **MODEL VALIDATION ..... 68**

4.1 Cell-gap displacement experiment.....	69
4.2 Solid Model.....	71
4.2.1 Linear Elastic Model for EVA.....	72
4.2.2 Viscoelastic Model for EVA.....	73
4.3 Shell Model.....	74
4.4 Conclusions.....	77

## **CHAPTER 5**

### **QUALIFICATION TEST USING FINITE-ELEMENT MODEL ..... 79**

5.1 Loads and Boundary Conditions.....	79
5.2 Glass.....	80
5.3 Backsheet .....	81
5.4 Cells .....	82
5.5 Interconnects .....	83
5.6 Parametric study.....	84
5.7 Conclusion .....	85

## **CHAPTER 6**

### **LIFE PREDICTION OF PV MODULES..... 87**

6.1 Fatigue.....	88
6.2 Basquin-Coffin-Manson Model.....	90
6.3 Representative Days.....	92
6.4 Loads and Boundary Conditions.....	94

6.4.1 Lamination Process .....	94
6.4.2 Sequentially Coupled Analysis .....	94
6.5 Results and Discussion .....	96
6.5.1 Lamination Process .....	96
6.5.2 Stress Analysis on the Basis of Worst Day Conditions .....	98
6.6 Life Prediction .....	105
6.7 Conclusions.....	107
<b>CHAPTER 7</b>	
<b>COMPARATIVE STUDY OF PV MODULE ENCAPSULANTS .....</b>	<b>109</b>
7.1 Significance and Desirable Properties .....	111
7.2 Material Modeling Of Encapsulants .....	112
7.3 Results and Discussion .....	119
7.4 Comparison of Other Properties .....	131
7.5 Weighting and Rating of Encapsulants.....	133
7.6 Conclusions.....	135
<b>CHAPTER 8</b>	
<b>SENSITIVITY ANALYSIS.....</b>	<b>140</b>
8.1 Basic Principles of Sensitivity Analysis .....	141
8.2 Selection of Input and Output Parameters .....	143
8.3 Results and Discussion .....	148
8.4 Conclusions.....	152

## **CHAPTER 9**

<b>CONCLUSIONS AND RECOMMENDATIONS.....</b>	<b>153</b>
<b>NOMENCLATURE.....</b>	<b>157</b>
<b>REFERENCES.....</b>	<b>162</b>
<b>VITAE.....</b>	<b>171</b>

## LIST OF TABLES

Table 1.1: Silicon cell types.....	6
Table 1.2: Temperatures of PV components during manufacturing.....	10
Table 1.3: Desired properties of encapsulant and their link with PV module performance .....	18
Table 1.4: Link between no. of temperature cycles and life of a PV module used by BP Solar [31] .....	21
Table 2.1: Summary of the material models used for the components of a PV module in the literature. <i>T.I stands for temperature independent, T.D stands for temperature dependent and BISO stands for bilinear isotropic hardening model</i> .....	27
Table 3.1: Comparison of the material modeling used in the literature along with the material model used in the current work. <i>T.D stands for Temperature Dependent; T.I stands for Temperature Independent and BISO stands for Bilinear Isotropic</i> .....	34
Table 3.2: Elastic constants for the compliance of Silicon [42] .....	36
Table 3.3: Bilinear elastic-plastic model for Copper [26] .....	51
Table 3.4: Viscoelastic properties for EVA .....	55
Table 3.5: Material properties of module components. <i>T. dep.</i> stands for temperature dependent, <i>BISO</i> stands for bilinear isotropic.....	60
Table 3.6: Thickness of layers within the PV laminate .....	63

Table 3.7: Mesh convergence with respect to maximum von-Mises stress in all layers ..	67
Table 4.1: Thicknesses of the components of the PV module specimen.....	70
Table 6.1: Fatigue parameters for copper [64].....	92
Table 6.2: Maximum and minimum principal stress and stress amplitude at a point on the interconnect for all four representative days.....	105
Table 6.3: Maximum and minimum total strain, no. of cycles to crack initiation, life and respective weights for the four representative days.....	107
Table 7.1: Thermo-mechanical properties of polymers used in simulation.....	118
Table 7.2: Table comparing von-Mises stress, first principal stress and total first principal strain at four points (Fig. 7.4) using different encapsulants .....	122
Table 7.3: Von-Mises stress, first principal stress and total strain comparison of encapsulants at minimum and maximum temperature for the month of October .....	123
Table 7.4: Maximum power point voltage and current along with the efficiency of the PV module at 10:30 am during Day 3 for all encapsulants .....	126
Table 7.5: Life of PV module using different encapsulants .....	130
Table 7.6: Comparison of encapsulant properties in the literature .....	137
Table 7.7: Weighting matrix for properties of encapsulant and its outcomes .....	138
Table 7.8: Rating matrix for encapsulants by scaling properties on a scale of 1 to 5 and the weights from Table 7.7 .....	139
Table 8.1: List of variables with their nominal values selected for sensitivity analysis.	147

Table 8.2: Outputs and NSCs with respect to varying input properties of EVA

encapsulant.....	149
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## LIST OF FIGURES

Fig. 1.1: Cross-section of a PV module .....	4
Fig. 1.2: The Czochralski process [8] .....	7
Fig. 1.3. Untabbed PV cells at BP Solar, Riyadh .....	9
Fig. 1.4: Images of crack in cells, broken interconnects and solder bond failures captured by Jeong et al. [18].....	13
Fig. 1.5: (a) Delamination [13] (b) Glass breakage [19].....	14
Fig. 1.6: Hemispherical transmittance vs. wavelength of different aged samples of EVA [19].....	16
Fig. 1.7: Failure modes of PV modules [24].....	17
Fig. 1.8: One temperature cycle of ASTM E1171-09 [4] .....	20
Fig. 3.1: Structure of Silicon crystal .....	36
Fig. 3.2: (a) Tensile tests on copper interconnects performed on Zwick and (b) Temperature dependence of Young's modulus of copper interconnects found by DMA [11].....	39
Fig. 3.3: (a) Maxwell's model and (b) Generalized Maxwell or Maxwell–Wiechert model .....	45
Fig. 3.4: (a) DMA experiment result for EVA and (b) Isothermal relaxation curves for EVA [41].....	47
Fig. 3.5: CTE of Silicon vs. Temperature [51,52] .....	50
Fig. 3.6: CTE of Copper vs. Temperature [54].....	52

Fig. 3.7: Prony series fit of the Master curve in [41] .....	56
Fig. 3.8: Modes of energy transfer in a PV module.....	59
Fig. 3.9: Equivalent circuit of an actual PV cell .....	62
Fig. 3.10: Dimensions of the shell model .....	64
Fig. 3.11: Layered configuration of areas along transverse direction.....	65
Fig. 3.12: Interconnection approximation in shell model .....	65
Fig. 3.13: FE mesh of the geometric model for PV module .....	66
Fig. 4.1: Flowchart of the validation process.....	69
Fig. 4.2: Digital image correlation experimental setup to measure cell-gap displacement [38] .....	70
Fig. 4.3: The temperature history of the cell gap displacement experiment [37] .....	71
Fig. 4.4: Solid model representing cells without glass, encapsulant and backsheet .....	72
Fig. 4.5: Comparison of the two linear elastic models with experimental data.....	73
Fig. 4.6: Comparison of the viscoelastic model of EVA with the experimental results...	74
Fig. 4.7: Von-Mises stress contour of backsheet using (a) Solid Model and (b) Shell Model .....	75
Fig. 4.8: Von-Mises stress contour of glass cover using (a) Solid Model and (b) Shell Model .....	75
Fig. 4.9: Von-Mises stress contour of cells using (a) Solid Model and (b) Shell Model .	76
Fig. 4.10: Comparison of the von-Mises stress along the thickness of the module by using solid and shell models .....	77
Fig. 5.1: Simulated temperature profile .....	80



Fig. 5.2: First principle stress on glass (left) and third principle stress on glass (right) at -40°C .....	81
Fig. 5.3: First principle stress within the back-sheet at -40°C .....	82
Fig. 5.4: $x$ -direction stress in cells (left) and third principle stress in cells (right) at -40°C .....	83
Fig. 5.5: First principle stress on the interconnects (left) and on the connection between the interconnects (right) .....	84
Fig. 5.6: Parametric study showing max. third principle stress on cells by varying encapsulant thickness.....	85
Fig. 6.1: Flow chart of the modeling process.....	88
Fig. 6.2: The irradiation and ambient temperatures of the four representative days. ....	93
Fig. 6.3: Von-Mises stress (left) and von-Mises plastic strain (right) at room temperature after the lamination process at the interconnect region between two adjacent cells (shaded region A) .....	97
Fig. 6.4: Max von-Mises stress and first principal stress through the thickness of the module at lowest temperature on Day 4 .....	98
Fig. 6.5: First principal stress contours of (a): Glass at region A, (b): Backsheet at region A, (c): Cells at region B and (d): Encapsulant at region A at lowest temperature on Day 4.....	101
Fig. 6.6: First principal stress contours of (a): Interconnect region over the cells at region A and (b): Interconnect region between two adjacent cells at region B at lowest temperature on Day 4.....	102

Fig. 6.7: Stress variation on module along (A): Longitudinal path <i>AB</i> and (B): Transverse path <i>CD</i> at lowest temperature on Day 4 .....	103
Fig. 6.8: Transient change in von-Mises stress and first principal stress for Day 3 on copper interconnect. A represents the time of max. stress, min. temperature and B represents the time of min. stress, max. temperature .....	104
Fig. 7.1: Measured data and master-curve for the loss modulus of PDMS in reduced frequency domain [73] .....	115
Fig. 7.2: Master-curve for PVB obtained by shifting the results of stress relaxation experiments at different temperatures in time domain [74] .....	116
Fig. 7.3: Prony fit of the master-curve of TPU in time domain [75] .....	117
Fig. 7.4: Location of points A, B, C and D for Table 7.2 .....	121
Fig. 7.5: Stress variation on module along (a): Longitudinal path <i>AB</i> and (b): Transverse path <i>CD</i> at lowest temperature on Day 3 using EVA as encapsulant .....	124
Fig. 7.6: Von-Mises stress variation along thickness of the PV module using EVA for Day 3 .....	126
Fig. 7.7: Cell efficiency for the five encapsulants during Day 3 .....	127
Fig. 7.8: Total first principal strain variation for all encapsulants during Day 3. Line A shows the time of maximum strain and Line B shows the time of minimum strain .....	127
Fig. 8.1: Nominal system [90] .....	143
Fig. 8.2: Perturbed system [90] .....	143

Fig. 8.3: Division of relaxation curve of EVA on time scale where A represents the glassy region, B represents the viscoelastic region and C is the rubbery region. ....	144
Fig. 8.4: Relaxation modulus of EVA at -20°C varying by one order .....	145
Fig. 8.5: Relaxation modulus of EVA at -20°C with varying slope by 10% .....	145
Fig. 8.6: NSC evaluated through life output varying encapsulant parameters .....	150
Fig. 8.7: NSC evaluated through power output varying encapsulant parameters.....	150

## **ABSTRACT (ENGLISH)**

**NAME:** Osama Hasan  
**TITLE:** Performance and Life Prediction Model for Photovoltaic  
Module: Effect of Encapsulant Constitutive Behavior  
**MAJOR FIELD:** MECHANICAL ENGINEERING  
**DATE OF DEGREE:** APRIL 2013

A Photovoltaic (PV) module consists of layers of different materials constrained together through an encapsulant polymer. During operation, it experiences thermal loads due to temperature variations and humidity, which cause breakage of interconnects owing to fatigue, corrosion and laminate warpage. The encapsulant protects the silicon cells and interconnects from moisture, heat and mechanical damage. The lifetime of today's PV module is expected to be 25 years, but the problem is that it is not convenient to wait and assess its durability. The objective of the current work is to develop a comprehensive Finite-Element (FE) model capable of capturing the actual behavior of PV module under operation. Viscoelasticity of the encapsulant polymer was taken into account and the silicon cells were modeled as orthotropic. It was found that the copper interconnects undergo plastic deformation just after the lamination process. The developed model is sequentially-coupled to a transient thermal model. By using meteorological data, average life of PV module is predicted considering thermal fatigue life of copper interconnects. Finally, an encapsulant based comparative study is performed to determine an optimal option with respect to various parameters affecting PV module performance and life.

## ABSTRACT (ARABIC)

### ملخص الرسالة

الاسم: اسامه حسن

عنوان الرسالة: تقويم الأداء و العمر الافتراضي لنموذج الخلية الكهروضوئية : تأثير السلوك

الميكانيكي للغلاف البوليمري

التخصص العام : الهندسة الميكانيكية

تأريخ التخرج: 1434 هـ - (أبريل 2013 م)

الخلايا الكهروضوئية تتألف من طبقات من مواد مختلفة مترابطة معا من خلال غلاف بوليمري . فمن خلال عملية التغليف تواجه الخلايا الكهروضوئية أحمالا ميكانيكية وحرارية نظرا لاختلاف درجة الحرارة الموسمية واليومية والتغير في رطوبة الهواء والتي بدورها تسبب في كسر الروابط بسبب تكرار التأثير الحراري والتآكل في المواد وانفصالها. وكما هو معروف فالهدف من الغلاف البوليمري هو حماية خلايا السيلكون و المواد المترابطة معها من الرطوبة و الحرارة و الأضرار الميكانيكية الأخرى. لقد قدر العمر الافتراضي للخلايا الكهروضوئية الحالية بـ 25 عاما لكن المشكلة أنه ليس من المعقول الانتظار طول هذه المدة لتقييم الخلية الضوئية. لذا تهدف هذه الدراسة في تطوير نموذج شامل بواسطة نظام العناصر المحددة قادر على تمثيل سلوك الخلية الكهروضوئية تحت الظروف العملية آخذة بعين الاعتبار اللزوجة المطاطية للغلاف البوليمري لتكون متعامدة مع خلايا السيلكون. وقد وجد أن الروابط النحاسية تخضع لتشنوه بلاستيكي بعد عملية التغليف. وقد تم اختبار النموذج المطور وذلك بعرضه لنظام حراري متتابع وعابر باستخدام بيانات الأرصاد الجوية ومن خلال ذلك أمكن حساب العمر الافتراضي لنموذج الخلية الكهروضوئية آخذا في الاعتبار عمر العبء الحراري للنحاس والمواد المرتبطة به. وأخيرا أختبر أداء التغليف بالاعتماد على بحوث مماثلة لإيجاد أفضل الخيارات بالنسبة للعوامل المؤثرة على عمر وأداء نموذج الخلية الكهروضوئية.

# **CHAPTER 1**

## **INTRODUCTION**

In the near future, the demand for electric energy is expected to increase rapidly due to the global population growth and industrialization. This increase in the energy demand requires electric utilities to increase their generation. Recent studies predict that the world's net electricity generation is expected to rise from 20,261 terawatt-hours in 2008 to 24,400 terawatt-hours (an increase of 20.4%) in 2015 and 33,300 terawatt-hours (an increase of 64.4%) in 2030 [1]. Currently, a large share of electricity is generated from fossil fuels, especially coal, due to their low prices. However, the increasing use of fossil fuels accounts for a significant portion of environmental pollution and greenhouse gas emissions, which are considered as the main reason behind the global warming. For example, the emissions of carbon dioxide and mercury are expected to increase by 35% and 8%, respectively, by the year 2020 due to the expected increase in electricity generation [2]. Moreover, possible depletion of fossil fuel reserves and unstable price of oil are two main concerns for industrialized countries.

To overcome the problems associated with generation of electricity from fossil fuels, renewable energy sources can be participated in the energy mix. One of the renewable

energy sources that can be used for this purpose is the light received from the sun. This light can be converted to clean electricity through the photovoltaic (PV) process. The use of PV systems for electricity generation started in the seventies of the 20th century and is currently growing rapidly worldwide. The PV industry is growing even in times of economic crisis. The global solar electricity market is currently more than \$10 billion/year and the industry is rising at a rate of greater than 30% per annum [3].

But questions arise that whether PV modules are reliable enough? Can this technology be considered to reach grid parity? The lifetime of today's PV module is expected to be 25 years with 20% reduction in its power output over this period, and this is usually guarantee of the manufacturer. In accordance with such requirements, the PV module must withstand mechanical loads reliably. Its high reliability will help it to reach grid parity. But the problem is that it is not convenient to wait and assess its durability. Qualification standards such as ASTM E1171-09 and ASTM E1830-09 [4,5] are useful in predicting failures during the infant mortality period of operation but cannot foresee long-term failures.

A PV module consists of layers of different materials (glass, interconnects, cells and back sheet) constrained together through an encapsulant polymer. During operation, it experiences mechanical and thermal loads due to seasonal and temperature variations ultimately stressing the each component of the module. This is due to the fact that there is a coefficient of thermal expansion (CTE) mismatch because of the presence of unlike materials within the laminate.

## 1.1 PERFORMANCE OF A PV MODULE

Performance is actually the measure of output derived by a system. In the case of Photovoltaic (PV) modules, the total performance can be categorized into 3 types:

- (i) Electrical Performance: It is estimated through the power output ( $V_{mp} \times I_{mp}$ ) and the electrical efficiency.  $V_{mp}$  and  $I_{mp}$  are the voltage and current at maximum power point respectively.
- (ii) Thermal Performance: Thermal performance is basically measured through the electrical performance except that it is can be quantified through cell temperature ( $T_c$ ). In the case where a thermal collector is attached to a PV module, it is also a measured through outlet fluid temperature ( $T_{out}$ ) of the thermal collector. It actually gives the heat removal from cells. As described in [6], electrical performance of a PV module is adversely affected by increasing  $T_c$ .
- (iii) Structural Performance: The current work is based on this part of performance and it is estimated through working life of a PV module. Life can be defined as the time span within which a module delivers performance up to its specification. For example, the statement “*25-year life with 20% loss in power output*” given by most manufacturers specifies the structural performance of the PV module.



## 1.2 STRUCTURE OF A PV MODULE

As shown in Fig. 1.1, a PV module consists of layers of different materials (glass, interconnects, cells and back sheet) that are bound together through an encapsulant polymer. This single laminate of various materials is formed by the lamination process, in which the encapsulant is placed between each layer and melted at its curing temperature. Polymer chains are cross-linked after curing and the whole laminate is cooled to room temperature. Upon cooling, each material tends to contract but all of them are restricted to one another due to adhesion of the encapsulant. The differences in the coefficients of thermal expansion (CTE) of all components induce thermo-mechanical stresses within them. Hence, a PV module is pre-stressed before its service. During operation, it experiences temperature cycles of day and night due to which each component is further stressed within the laminate, which may lead to failure.

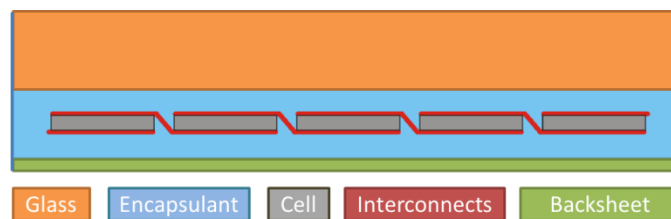


Fig. 1.1: Cross-section of a PV module

## **1.3 MANUFACTURING OF PV MODULES**

During manufacturing of a PV module, the components face different mechanical and thermal stresses. The pre-stress during manufacturing adds up to the stresses generated during operating loads, thus it is important to study in order assess the structural performance of PV module.

### **1.3.1 Silicon Purification**

Silicon is purified in the first step. Silicon dioxide is used as a raw material to develop pure silicon. Purification is done by dissociating silicon dioxide into pure molten silicon and carbon dioxide by an electric arc. The process is done in an electric arc furnace. Silicon obtained by this method is almost 1% impure. Even this percentage of impurity is removed through the Floating Zone technique. The general procedure is to drag the silicon rod through a heated zone, in the same direction, several times. The impurities are dragged to one end with each pass. After a certain time and number of passes, impurities are removed and pure silicon is obtained [7].

### **1.3.2 Types of Silicon Cells**

General silicon cell types are listed in Table 1.1. A thin film solar cell is produced by depositing PV material on a ceramic substrate. These PV materials are mentioned in the second column of Table 1.1. Thin-Film solar cells are less efficient as compared to crystalline cells. Throughout the literature, only monocrystalline cells will be considered.

Monocrystalline silicon cells, among the mentioned types, are the most efficient cells. It comprises of a single crystal of silicon grown and doped, as discussed in the next articles.

Table 1.1: Silicon cell types

<b>Crystalline Solar Cells</b>	<b>Thin-Film</b>
Monocrystalline	CdTe
Polycrystalline	CIGS
	CIS
	a-Si

### 1.3.3 The Czochralski Process

The growing of monocrystalline silicon into a pseudo-square wafer cut from column ingots is done through the Czochralski (CZ) process as shown in Fig. 1.2 [8]. The technique involves lowering a seed crystal of silicon into silo of melted purified silicon. As the dipped seed crystal is removed from the vat and rotated, a long cylindrical boule of silicon is formed. This ingot or boule is extremely pure, as the impurities are likely to remain in the molten liquid. Now the ingot is sliced to make wafers of silicon by a circular diamond saw. A multiwire saw may also be used to cut many wafers at a time. About half of the silicon is wasted from the ingot to the finished circular wafer. In order to increase the area of solar cells over the front surface of the module, the wafers are given pseudo-square shape as they require less space and get more room for additional silicon in a limited area.

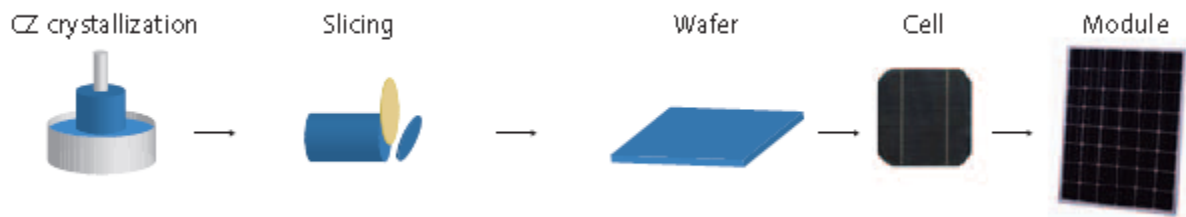


Fig. 1.2: The Czochralski process [8]

#### 1.3.4 Doping

Doping is done to make pure silicon able to conduct electricity. Basically, dopant elements are added in order to produce excess of electrons in one region and deficiency of electrons in the other. Usually, silicon is p-doped by adding boron in the Cz process described previously. Gallium, Indium or their combination with Boron can also suffice as a p-dopant. Now, the n-dopant is applied whose conductivity is opposite as of the first dopant. This forms a p-n junction in the wafer. This is done by depositing the n-dopant onto the surface of the wafer and heating the wafer in order to allow the n-dopant to penetrate within. Phosphorus is a preferred n-dopant and applied through Phosphorus oxychloride. The wafer is heated to a temperature of 700°C to 850°C [9].

#### 1.3.5 Anti-Reflective Coating, Back Surface Field and Busbar Pads

Pure silicon is shiny and it reflects up to 35% of light. To decrease reflection, anti-reflective coatings (ARC) are deposited on the front surface of the cells. The coatings comprise of di-electric such as titanium oxide, silicon dioxide or silicon nitride. There are several methods of depositing the anti-reflective layer, one of which is Low Pressure

Chemical Vapor Deposition (LPCVD). The wafer is kept in an environment of a silicon compound and ammonia at an elevated temperature of 750°C to 850°C [9]. Then the surface coating, preferably silicon nitride is deposited.

The wafer also receives a coating of the Back-Surface Field (BSF) to increase the efficiency of the cell. It comprises of depositing of a heavily doped p++ layer at the back of the cell usually aluminum. The aluminum containing paste is applied at the back of the cell by screen printing technique and then it transferred to a furnace where the cell is heated to a temperature of 200°C, so that newly applied paste can diffuse.

Busbar pads are used to solder interconnects onto the cells thereby connecting one cell to other. It is applied by using the same technique as of the BSF, except silver paste is applied instead of Aluminum.

### **1.3.6 Assembling Process at BP-Solar, Riyadh**

The assembling process was investigated at a BP Solar facility in Riyadh. The steps are summarized as follows:

- (i) As depicted in Fig. 1.3, pure copper strips are soldered over the busbar pads from the top (n-doped) of the cell to the bottom (p-doped) of the next cell to connect them in series. The soldering temperature is about 280°C.
- (ii) Next the soldered cells are kept in between two encapsulant layers (Ethylene-Vinyl Acetate) and placed over the top of a white tedlar back-sheet. A float glass is then positioned over the top of the laminate and the whole assembly is

kept in a laminator. The assembly is heated to 150°C for about 12 minutes. The heating is done so that the encapsulant layers melt and cover the empty spaces plus adds adhesion in between so that the assembly converts into a single laminate after cooling to room temperature.

A junction box (J-box) is fixed below the panel and the panel is fixed with an aluminum frame. The frame is sealed by butyl rubber.

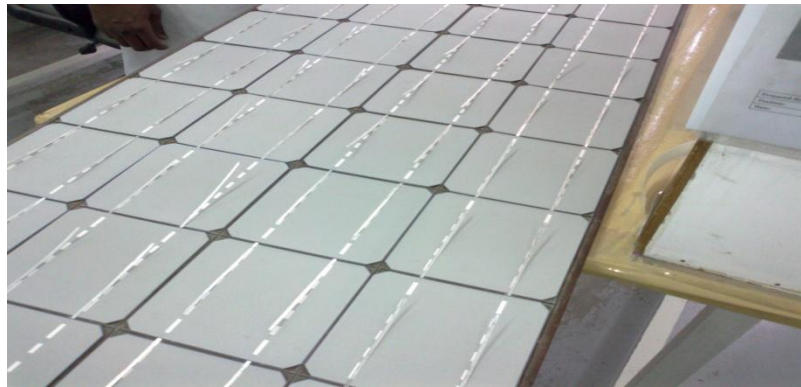


Fig. 1.3. Untabbed PV cells at BP Solar, Riyadh

### **1.3.7 Summary of Pre-Stress within the Module**

In the described manufacturing process of solar cell, it faces a number of temperature changes. When silicon is solidified after purification, it may result in internal tension. However, due to slow cooling rate, internal stresses might be negligibly small. Then there are the stresses caused by the sawing process after the growth to form an ingot. Here, it is assumed that purely elastic deformations occur in the wafer [10]. The pure Si wafer, under these assumptions, is regarded as a stress-free. But, stresses may arise due to the screen printing and coating processes as they are done at high temperatures. Due to the

difference in thermal expansion of aluminum at the back and silver on the top, it may induce significant stresses within the cell. Next, the cell is heated to temperatures between 230°C and 300°C depending on the soldering alloy. To reduce the temperature difference, the cells are usually preheated to a temperature of 120°C. During the lamination process heating takes place to a temperature of 150°C (curing temperature of EVA). In this step, all of the components of a solar module expand, and are unbound to one another at that instant. The subsequent cooling causes the encapsulant to bind all the components. Due to different CTEs of components, each restrains one another inducing thermo-mechanical stresses. As a simple assumption it can be stated that all the components of the laminate are stress free at 150°C. Table 1.2 gives the summary of the processes described.

Table 1.2: Temperatures of PV components during manufacturing

<b>Process</b>	<b>Components</b>	<b>Condition</b>
Cz process	Si wafer	Sawing after growing crystals
Doping	Si wafer	Heated from 700°C to 850°C
Deposition of the ARC	Cell	Heated from 750°C to 850°C
Deposition of BSF & Busbar pads	Cell	Heated to 200°C
Soldering	Cell, Interconnects	Heated to 280°C
Lamination	All	Heated to 150°C

## 1.4 LOADS DURING OPERATION

While at service, a PV module may be subjected to four types of loading [11].

(i) Mechanical Loads

Wind is a source of mechanical load on the module. Heavy winds may be a reliability concern for a PV module. Similarly, snow accumulation over the top of glass surface for long periods and its intrinsic weight induce creeping effects on the structure.

(ii) Thermal Loads

As stated earlier, a PV module is aged due to consecutive alternation of day and night and seasonal temperature variations. Therefore, thermo-mechanical stresses are induced and may cause failures due to fatigue in certain components.

(iii) Radiation

UV rays from sun decreases the functionality of the polymeric sheet deployed as encapsulant material by either yellowing the medium (decreases the transparency of encapsulant) or by destruction of chemical bonds (decreases the adhesion).

(iv) Chemical Loads

Moisture in the form of damp heat is the main cause of corrosion. Moisture reacts with the polymeric sheet which introduces chemicals within the interface. These chemicals corrode the components of PV module.



## **1.5 FAILURE MODES**

Failure is defined as the change in properties of a structure, machine or machine part that makes it inept to perform its intended functions. The occurrence of such failure is through physical means which are known as failure modes [12]. In the case of PV modules, failure may be stated as when the module is not capable of producing power as per its specification due to degradation caused by failure modes. While operating at the field, a PV module is subjected to various loading conditions. A number of failures have been reported during the course of its operation. Such failures have been adequately described in [13,14] which are summarized below.

### **1.5.1 Cracks in Cells**

During the last decade, PV industry has constantly been trying to decrease the thickness of silicon cells as this will help to decrease the manufacturing cost and to increase the cell's performance [15]. It is reported that in the last few years, thickness of silicon wafers has decreased from 300 $\mu\text{m}$  to 200 $\mu\text{m}$  today [16] and is expected to decrease to 100 $\mu\text{m}$  by the year 2020 [17]. But with the reduction in thickness, cell area has increased up to 210 mm  $\times$  210 mm. These factors have made the cells more susceptible to warpage and ultimately cracking. These problems arise because of the thermally induced stresses during soldering and lamination which eventually lead to cracking during operation. Direct mechanical stressing due to snow and ice accumulation over the top surface has also been regarded as the reason for such failure. Fig. 1.4 [18] gives the image captured for cracks in cells for an aged PV module.

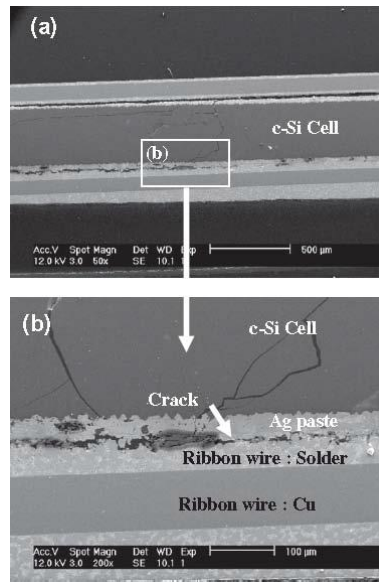


Fig. 1.4: Images of crack in cells, broken interconnects and solder bond failures  
captured by Jeong et al. [18]

### 1.5.2 Breakage of Interconnects and Solder-bond Failures

The thickness of interconnects range from 100μm to 150μm whereas for the solder bond is only about 15μm to 20μm. Such small thicknesses make them easily vulnerable to damage. These components are already pre-stressed as they have gone through the soldering process. Furthermore, during operation they undergo temperature cycles of day and night causing fatigue which leads to breakage. This failure mechanism is shown in Fig. 1.4.

### 1.5.3 Detachment of the Frame and Delamination

Detachment is basically the separation of the frame whereas delamination is the loss of adherence of the encapsulant material as shown in Fig. 1.5 (a). Both the failures are

originated when PV modules operate under hot and humid climates. When such a phenomenon occurs, it may cause foreign impurities to enter and react with the constituents of the module producing gases which may be seen in the form of bubbles over the surface. The damp heat causes moisture ingress which corrodes the components of the module. Another reason is the use of stiff adhesive which loses its adhesion under thermocycling.

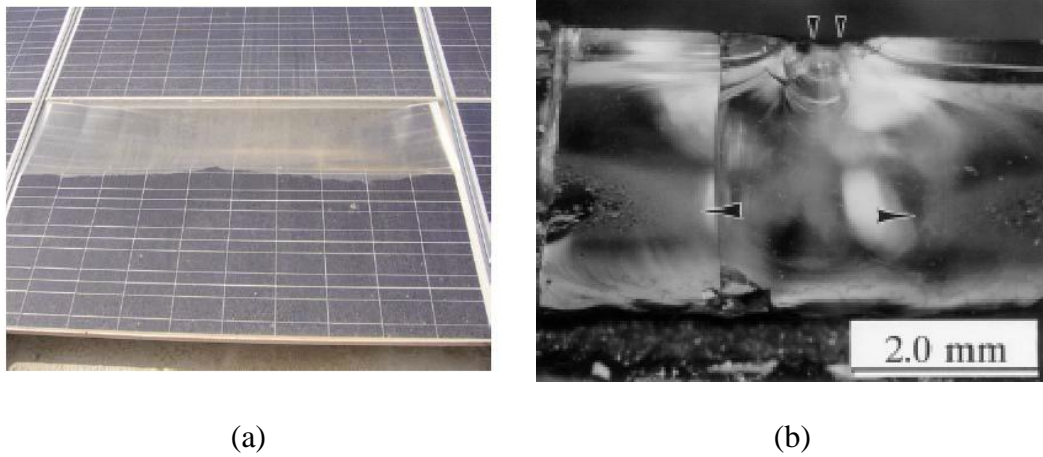


Fig. 1.5: (a) Delamination [13] (b) Glass breakage [19]

#### 1.5.4 Glass Breakage

Glass provides mechanical rigidity to the module being the thickest layer within the laminate. If the glass is not strong enough, cracks and fractures may induce due to direct mechanical loading on the top surface. Such cracks are harmful as it provides open invasion of environmental impurity to react with the inner components of the laminate. It is generally caused due to stresses induced by hail, storm, and snow and ice accumulation. Fig. 1.5 (b) shows the damage in glass due to windblown roofing gravel [19].

### **1.5.5 Encapsulant Degradation**

For more than 25 years, Ethylene-Vinyl Acetate (EVA) has been used as an encapsulant in PV module industry. It is prioritized over other encapsulants due to its low cost. However, EVA encapsulants have shown problems in the field and its failures have been studied through accelerated aging tests. In [20], by performing experiments, it was seen that the EVA produced acetic acid in moisture which could catalyze the corrosion process. It was also seen that the glass transition of EVA starts at temperatures less than -15°C, which decreases the compliancy of EVA and may cause mechanical problems due to wind and snow in regions with the cold climate. In [21,22], accelerated aging tests are performed to photovoltaic modules to find out the effect of ultraviolet (UV) radiation which the disruption of bonds of EVA prescribed by its yellowing. Some of the aged samples of EVA were tested and compared in Sandia labs as shown in Fig. 1.6 [19]. In [23], UV exposure, damp heat and 30x concentration tests were performed encapsulant samples which were the variants of EVA and silicones. It was found that EVA had a great tendency to react with moisture.

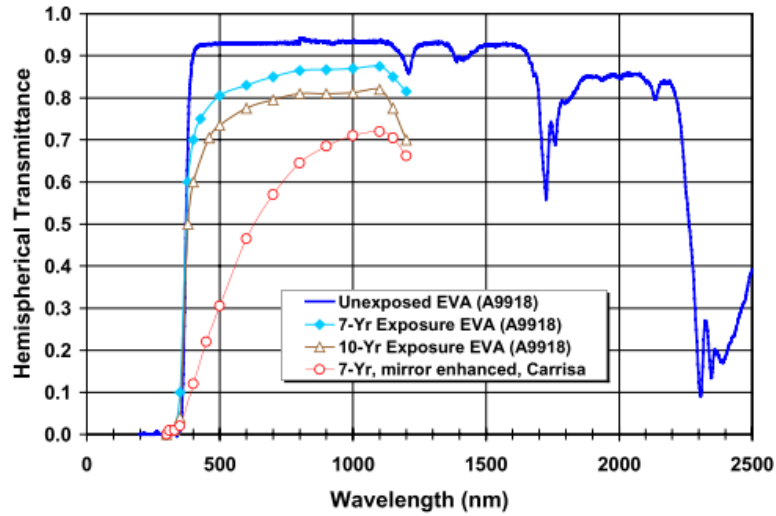


Fig. 1.6: Hemispherical transmittance vs. wavelength of different aged samples of EVA [19]

### 1.5.6 Dominant Mode of Failure

A number of failures have been reported during the course of its operation. Wohlgemuth et al. [24] have gathered commercial PV module returns under warranty of BP Solar/Solarex from 1994 to 2005. Each product was examined and the cause of failure was found which is summarized by Fig. 1.7. It is seen that corrosion and cell/interconnect breakage have the highest part in failure. Wohlgemuth et al. [25] have concluded that cell breakage during operation is due to pre-damaged cells during soldering. Wiese et al. [11,26] have attributed interconnect breakage to fatigue as a result of thermo-cycling. Such failures deteriorate PV module performance, ultimately affecting its life.

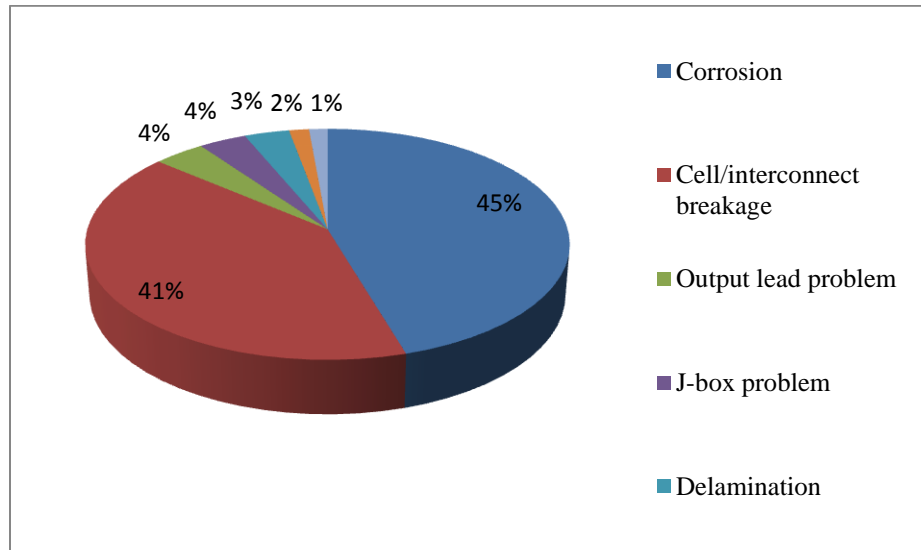


Fig. 1.7: Failure modes of PV modules [24]

## 1.6 ROLE OF ENCAPSULANT IN PV MODULE PERFORMANCE

In [27], the encapsulant is defined as a protective material to completely wrap up and segregate the silicon cells from moisture, heat and mechanical damage in addition to good optical contact between the surface of PV cells and the outer coating. In addition to this, good thermal conduction has also been defined as a property of encapsulant in [28]. In [20,21], the encapsulant defined as a polymeric material that is used to provide electrical insulation and protection from mechanical damage and environmental corrosion. In addition to this, there are certain desirable properties which come with an encapsulant and link to the performance of a PV module. Table 1.3 provides a list of desired properties and the dependence of PV module performance to them. A good transparent encapsulant and such which prevents its yellowing through resistance of UV

radiation provides a means of more transmittance of light and thus more electrical power. Higher the thermal conductivity of the encapsulant polymer, more will be the heat removed absorbed by cells through natural convection or by attaching an auxiliary thermal collector. An encapsulant should also be in its solid state within the operating range of the module and its glass transition temperature should be far from this range, as the sudden change of properties might decrease the compliancy or adhesiveness of the encapsulant material.

Table 1.3: Desired properties of encapsulant and their link with PV module performance

<b>Properties</b>	<b>Performance</b>
Good transmittance of light	Electrical Performance
Good thermal conduction	Thermal performance
Long operational temperature range	Structural performance
UV radiation resistant	Electrical performance
Good compliancy	Structural performance
Low glass transition temperature	Structural performance
Long life	Structural performance

## **1.7 QUALIFYING STANDARDS**

Studies on the reliability and failure observed during its operation began in the 1980s [29]. It has led to the design of certain certification methods for PV modules such as ASTM E1171-09 [4], ASTM E1830-09 [5] etc. These standards are generally test methods to determine the ability of a PV module to withstand thermo-mechanical stresses under heat, humidity, static and dynamic loads etc. ASTM E1171-09 deals with accelerated aging test using temperature cycles to simulate effect of day and night.

### **1.7.1 ASTM E1171-09 Temperature Cycle**

The temperature cycle test of ASTM E1171-09 (Fig. 1.8) is used to predict the effect of thermo-mechanical stresses. The module is run under accelerated aging test from 50 to 200 cycles between temperatures from -40°C to 85°C. Similarly, according to ASTM E1830-09 for the mechanical testing of wind effects, load of 2400 Pa is applied, both on the front side (wind pressure) and at the back (suction) of the module. This corresponds to a wind speed of 130 km/h with a safety factor of 3 for gusty winds. The burden on surface is maintained for 1 hour. To simulate the effect of snow and ice deposits, a load of 5400 Pa is applied to the front of module. The implementation of the temperature test is carried out in a climatic chamber with automatic temperature control and air circulation. The total cycle time must not be greater than 6 hours. The minimum time for isothermal condition is about 30 minutes and the heating rate can be up to a maximum of 100°C/h.



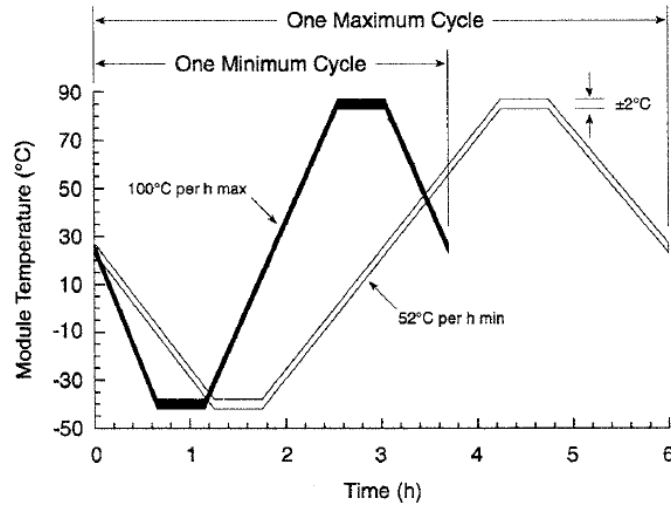


Fig. 1.8: One temperature cycle of ASTM E1171-09 [4]

### 1.7.2 Modified Standards

Qualification tests are generally used by manufacturers to determine the ability of a module to work under environmental conditions. It is also used by customers, to qualify it for purchase. But such standards are only beneficial to find design failures or failures which may occur during the infant mortality period for a module under test. This has been proven by evaluating qualified module failures under service and has been discussed in [30]. Thus, different manufacturers have developed their own reliability testing procedures by altering the conditions of tests mentioned in the standards. In the case of temperature cycles this is usually done by either increasing the test duration (no. of cycles) or by using higher stress levels (increasing the upper temperature limit and decreasing the lower one). The level of alteration and its relation to module life is determined by equalizing the failures caused by long-term field testing and reliability assessment. In [31], BP Solar has estimated the link between reliability testing and the

number of years of life through thorough experimentation by altering the no. of temperature cycles of the standard IEC61215, which is summarized in Table 1.4.

Table 1.4: Link between no. of temperature cycles and life of a PV module used by BP Solar [31]

<b>No. of Temperature Cycles</b>	<b>Life (years)</b>
50	2 to 2.5
200	10
400	20
500	25

## 1.8 MOTIVATION

As mentioned earlier, the usual warranty given by a PV module manufacturer is “25 years of life with 20% reduction in power output”. This has been found out by accelerated aging tests where their extremity has been linked with the no. of years of life. Thus, it can be said that it is quite a rough estimate of PV module life. It is also not convenient to wait assess the durability of a PV module for such long times as technology changes are rapid these days. Furthermore, accelerated aging tests account for a fixed temperature cycle and neglect the effect of different operating environments of PV modules. The long life of modules may help them to reach grid parity. Thus, a numerical model is required which accounts of life as well as the operating conditions of PV module. This model will be helpful in determining the pros and cons of design changes.

In chapter 2, it will be seen that FE modeling is a good tool to assess the durability of PV module. But some have modeled only cell strings while others have idealized the material behavior to temperature independent linear elastic. Hence, there is need of a comprehensive model which captures the actual behavior of PV modules. This comprehensive model should be able to be used with thermal and life prediction models to incorporate the effect of operating conditions and estimate reliability respectively.

## **1.9 OBJECTIVES OF PROPOSED WORK**

By viewing the voids within this field, the following objectives are set to fill them.

- (i) To develop a comprehensive FE structural model. The model will be validated and will be used to find the reason of dominant mode of failure.
- (ii) Use previously developed transient thermal model for PV modules and couple it to the structural model. This will include the effect of operating conditions.
- (iii) Couple a life prediction model using the outcomes of (ii). Life will be predicted through the meteorological data of representative days of a year.
- (iv) Selection of an optimum encapsulant polymer will be done by comparing those available in the market through the model developed on the basis of overall performance of PV module and its life.

## **CHAPTER 2**

### **LITERATURE REVIEW**

In the later sections, first a structural model for PV module was developed. Then it was coupled to a thermal model where the inputs of the thermal model were determined through optical modeling. The results of the coupled model were used to determine efficiency of the module using an electrical model. The outcomes were also used to predict PV module life. The following sections are dedicated to the previous work done on them.

#### **2.1 STRUCTURAL MODELING**

Literature shows some recent efforts in modeling and studying the structural behavior of PV module through Finite Element (FE) Analysis. Chen et al. in [32] developed a structural model which was used estimate cell warpage and residual stress induced during soldering. In [11,33], the pre-stress in the cell during assembly was studied. The variation of stress in cells was also studied by changing Young's modulus, yield strength and geometry of copper ribbon. The impact on stress in cell due to the composition of solder alloy was also viewed. Eitner et al. [34] have applied uniform temperature loads from

150°C to -40°C to the model and have calculated thermal stresses in a static time-independent analysis. The variation of stress and electrical loss is also compared for different interconnect designs. Dietrich et al. [35] have assessed thermo-mechanical behavior of a three cell interconnected string using sub-modeling procedure. PV module warpage and cell gap displacements were calculated during the lamination procedure. A parametric study was also performed with respect to encapsulant thickness to see its effect on deflection and stress in cells. Gonzalez et al. [36] have used their structural model to study the effect of encapsulant and cell dimensions on the thermal stress developing in cells and interconnects. This was done by performing a parametric study to view the consequence of encapsulant material, encapsulant thickness and cell thickness on the thermal stresses in cells. The conclusion of the work was that the stresses in cells are less for soft and thin encapsulant and thick PV cells. Again, Eitner et al. [37] have performed FE simulations for a 60 cell module during thermal cycling after validating their model with the experiment they have performed in [38]. In [39,40], Siddiqui and Arif have developed a model to determine the thermal stresses in cell during temperature variations of a day.

One of the most important aspects of modeling is the material model. The closer it gets to real life situation of a material, the better are outcomes of a numerical simulation. Chen et al. in [32] developed a 2D shell model of a single cell with aluminum back surface field (BSF), copper and solder. All materials were modeled as temperature independent elastic perfectly plastic. Wiese et al. in [11,33] have modeled copper, silver (busbar) and aluminum BSF as bilinear and silicon as linear elastic. Eitner et al. [34] have

modeled a 3D nine cell string with 2D plane stress back contacts for the string. Dietrich et al. [35] have assessed thermo-mechanical behavior of a three cell interconnected string using a 3D FE model and have included the effect of metallization paste used in soldering using sub-modeling procedure. Gonzalez et al. [36] have modeled a back contact PV module using 3D elements by idealizing copper as a layer in between silicon and the encapsulant. Eitner et al. [37] have modeled a 60 cell module as 3D without incorporating the interconnects. They have used temperature dependent properties for silicon and modeled it as an orthotropic material. They also have performed DMA and relaxation tests on Ethylene-Vinyl Acetate (EVA) to develop a viscoelastic model in [41]. The viscoelastic model was compared with linear and temperature dependent model for EVA. It was found that the viscoelastic model matched close to the experimental results of cell gap displacement. In [39,40], Siddiqui and Arif developed a PV model using 3D layered shell elements by defining layers of different materials within. They have also used a 3D solid model to couple an auxiliary heat exchanger to the PV module. Interconnects were not modeled and all panel materials were assumed to be temperature independent linear elastic. The model was used to determine the effect of changing operating conditions on the stressing of PV module. The material modeling in the literature is summarized in Table 2.1.

In almost all the previous work discussed either the lamination process was studied or IEC 61215 standard thermal test cycle (80°C to -40°C) was simulated or both of them were studied. Lamination process actually comprises the curing of the encapsulant material which for EVA is heating at 150°C and cooling to room temperature. The

selection of strain-free temperature was also different for them. It gives the initial state or the stress-free state of the model and has significant impact on the results of simulation. Eitner et al. [34,37] have assumed a strain-free temperature of 150°C to study lamination and temperature cycle of IEC 61215. Gonzalez et al. [36] have used a strain-free temperature of 100°C. Dietrich et al. [35] used a strain-free temperature of 150°C to simulate the lamination process. The study of the IEC 61215 temperature cycle comprised 20°C as the strain-free temperature. Siddiqui and Arif [39,40] have coupled a three dimensional numerical thermal model to the structural model of PV module. Thermal stresses were evaluated during the temperature variations of the day. Thus, their transient analysis was capable to simulate the effect of real-life environmental conditions.

Table 2.1: Summary of the material models used for the components of a PV module in the literature. *T.I* stands for *temperature independent*, *T.D* stands for *temperature dependent* and *BISO* stands for *bilinear isotropic hardening model*

References	Glass	EVA Encapsulant	Copper Interconnects	Silver Paste	Silicon Cells	Aluminum BSF	Backsheet
[32]	N.A	N.A	Perf. Plastic	Perf. Plastic	Perf. Plastic	Perf. Plastic	N.A
[11]	N.A	N.A	T.D, BISO	T.D, BISO	T.I	Non-Linear	N.A
[33]	N.A	N.A	T.D, BISO	T.D, BISO	T.D	T.D, BISO	N.A
[34]	T.I	T.I	N.A	N.A	T.I	N.A	T.I
[35]	T.D	T.D	T.D	N.A	T.D	N.A	N.M
[36]	T.I	T.I	Perf. Plastic	N.A	T.I	N.A	T.I
[37]	T.I	Viscoelastic	N.A	N.A	T.D	N.A	T.I
[39]	T.I	T.I	N.A	N.A	T.I	N.A	T.I



## **2.2 RADIATION AND OPTICAL MODELING**

Both models are used in conjunction to calculate absorbed solar radiation. This factor is used in thermal and electrical models to determine cell temperature and efficiency of the PV module respectively. Various radiation models have been discussed in [40] and implemented and a comparative study was performed to evaluate the prediction performance. The compared models included isotropic, Hay & Davies, Hay-Davies-Klucher-Riendl (HDKR) and Perez models. Experimental data for the representative simulated days was used to find that the Isotropic radiation model is the simplest and the most conservative whereas the Perez model had the highest over-prediction. HDKR model showed minimum mean bias error. Thus, in [40], the HDKR model was implemented as its prediction was closest to the actual values.

## **2.3 THERMAL MODELING**

Thermal models of PV modules are developed to evaluate the temperature field. Most of the previous work deals with the one-dimensional model which calculates temperature variations along thickness of the module. In [40], a three dimensional numerical model was developed to predict temperature field in a PV module, with and without cooling. The absorbed solar radiation from the HDKR model was used as an input to the model. This absorbed radiation was divided into two portions. One dealt with the generation of electrical energy, whereas the other one was consumed in temperature rise of the components of PV module. It was applied in the form of internal heat generation in cells.

Convection was applied to the top and bottom of the module using a correlation incorporating wind speed. Layered-shell elements were used to define the layered structure of PV module without cooling. Interconnects were not modeled. The model was validated using experimentally measured data and against the normal operating conditions temperature (NOCT) reported in the simulated module datasheet. A parametric study was performed under varying atmospheric and operating conditions. In the current work, the mentioned thermal model is used to couple it to the structural model developed in Chapter 3.

## **2.4 ELECTRICAL MODELING**

The performance of a PV module is estimated through its electrical efficiency. To estimate electrical performance, various electrical models have been developed. The inputs of such models are generally the absorbed radiation and the operating temperature of PV cells. In [40] several electrical models have been discussed. These include models in which temperature and radiation scaling of reference parameters, interpolation of I-V curves, empirical derivation of correlations and electrical circuit modeling are done. Three electrical models were selected for comparison with the five parameter model. The parameter estimation was done through multi-variable optimization technique known as the Nelder-Mead simplex search algorithm. The three performance models were the four parameter electric circuit model [48], Sandia labs model [49] and Villalva et al. [50] electric circuit model. These models were compared by simulating the performance of a total of six PV modules including three crystalline and three thin film cell modules. It

was seen that the new parameter estimation methodology provided comparable results to other models and better than the five parameter model when the parameter estimation methodology of Villalva et al. [50] was used. Next, a sensitivity analysis was performed to find the relative importance of five model parameters which are light current ( $I_L$ ), diode reverse saturation current ( $I_o$ ), modified diode ideality factor ( $a$ ), series resistance ( $R_s$ ) and shunt resistance ( $R_{sh}$ ). It was found that  $I_L$  and  $a$  are more sensitive than the other three parameters by several orders. By viewing the impact of these parameters, a seven parameter model was proposed. Two new parameters were introduced which actually are the irradiance dependence for  $I_L(m)$  and temperature dependence for  $a(n)$ . These were estimated using a secondary optimization routine by minimizing the objective function. This model showed improvement in the electrical performance prediction accuracy. In Chapter 3, the implementation of the seven parameter model is discussed.

## **CHAPTER 3**

### **COMPUTATIONAL MODEL FOR PV MODULES**

Chapter 2 deals with the literature review of structural modeling. By grasping the different ways of modeling, the best was chosen, shown by Table 3.1, which is intended to be close to the actual behavior of materials in the PV module. This chapter is about the implementation of the structural and thermal models into the Finite-Element (FE) package, ANSYS.

#### **3.1 BASIC THERMO-MECHANICAL PRINCIPLES**

For three dimensional temperature distribution, a thermal model was utilized as described by Siddiqui and Arif [39]. The energy equation of heat transfer for each layer is given by Eq. (3.1).

$$\rho_i C_{pi} \frac{\partial T_i(x, y, z)}{\partial t} = \nabla \cdot (\mathbf{q}_i) + Q_i, \quad i=1,2,\dots,n \quad (3.1)$$

where  $i$  represents the number of layers.

In [10], the classical linear elasticity theory has been discussed. Total strain is given as the sum of elastic strain and thermal strain as in Eq. (3.2). These strains are the result of mechanical loading and thermal expansion due to temperature change.

$$\{\boldsymbol{\varepsilon}\} = \{\boldsymbol{\varepsilon}^{el} + \boldsymbol{\varepsilon}^{th}\} \quad (3.2)$$

Here,  $\{\boldsymbol{\varepsilon}\} = \{\varepsilon_x \ \varepsilon_y \ \varepsilon_z \ \gamma_{xy} \ \gamma_{yz} \ \gamma_{xz}\}^T$  is the total strain vector,  $\{\boldsymbol{\varepsilon}^{el}\}$  is the elastic strain vector and  $\{\boldsymbol{\varepsilon}^{th}\}$  is the thermal strain vector. In a three dimensional case, thermal strain vector can be given as Eq. (3.3).

$$\{\boldsymbol{\varepsilon}^{th}\} = \{\alpha_x \ \alpha_y \ \alpha_z \ 0 \ 0 \ 0\}^T \cdot (T - T_{ref}) \quad (3.3)$$

where,  $\alpha_i$  is the linear Coefficient of Thermal Expansion (CTE) in  $i^{th}$  direction ( $i=x, y, z$ ),  $T$  is the current temperature and  $T_{ref}$  is the initial temperature. From Hooke's law, elastic strain is given as Eq. (3.4).

$$\{\boldsymbol{\varepsilon}^{el}\} = [S]\{\boldsymbol{\sigma}\} \quad (3.4)$$

where,  $[S]$  is the compliance matrix given in Eq. (3.8) for an isotropic material and  $\{\boldsymbol{\sigma}\} = \{\sigma_x \ \sigma_y \ \sigma_z \ \sigma_{xy} \ \sigma_{yz} \ \sigma_{xz}\}^T$  is the stress vector.

$$\boldsymbol{\sigma} = [D](\{\boldsymbol{\varepsilon}\} - \{\boldsymbol{\varepsilon}^{th}\}) \quad (3.5)$$

$$\boldsymbol{\sigma} = [D](\{\boldsymbol{\varepsilon}\} - \{\alpha_x \ \alpha_y \ \alpha_z \ 0 \ 0 \ 0\}^T \cdot (T - T_{ref})) \quad (3.6)$$

The Material properties matrix  $[D]$  in Eq. (3.5) is the inverse of compliance matrix. Eq. (3.6) gives the constitutive relation for thermo-mechanical induced stresses.

When temperature is increased in a material, the amplitude of vibration of the atoms increases with respect to their equilibrium position and leads to larger inter-atomic distances. This results in an increase in geometric dimensions of a material under subject. The thermal expansion coefficient describes the relative change in length of a body at a temperature change of 1 K. The coefficient depends on the strength of the inter-atomic bonds. Materials with strong bonds have a lower CTE, i.e., their expansion or contraction due to temperature change is lesser than those materials having a higher CTE.

The encapsulant material used in PV modules is a polymer. In polymers there are strong covalent bonds along the chain molecules, while the secondary bonds between the chains are weak. This leads to relatively large coefficient of expansion. The CTE of polymers is temperature dependent and it changes its behavior significantly above the glass transition temperature. The glass transition is the reversible transition in polymer materials from a hard and relatively brittle state into a molten or rubber-like state. The CTE is defined by Eq. (3.7) where  $L_o$  is the original length of the heated specimen.

$$\alpha(T_o, T) = \frac{L_F - L_o}{L_o(T - T_o)} = \frac{1}{L_o} \cdot \frac{\Delta L}{\Delta T} \quad (3.7)$$

Table 3.1: Comparison of the material modeling used in the literature along with the material model used in the current work. *T.D* stands for *Temperature Dependent*; *T.I* stands for *Temperature Independent* and *BISO* stands for *Bilinear Isotropic*

Ref.	Float Glass	EVA Encap.	Cu Connects	Ag Paste	Si Cells	Al Paste	Tedlar Backsheet
[35]	T.D	T.D	T.D	N.A	T.D	N.A	N.M
[36]	T.I	T.I	Perf. Plastic	N.A	T.I	N.A	T.I
[37]	T.I	Viscoelastic	N.A	N.A	T.D	N.A	T.I
[34]	T.I	T.I	N.A	N.A	T.I	N.A	T.I
[33]	N.A	N.A	T.D, BISO	T.D, BISO	T.D	T.D, BISO	N.A
[11]	N.A	N.A	T.D, BISO	T.D, BISO	T.I	Non-Linear	N.A
[32]	N.A	N.A	Perf. Plastic	Perf. Plastic	Perf. Plastic	Perf. Plastic	N.A
*	T.I	Viscoelastic	T.D, BISO	N.A	T.D	N.A	T.I

## 3.2 EVALUATION OF MATERIALS

Keeping in view of the material models discussed in the literature, the following topics are intended to understand the actual behavior of materials.

### 3.2.1 Silicon

Today, the element silicon is the most important component of most semiconductors and micro- electronic components. The wafer production procedure for making solar cells from both monocrystalline and polycrystalline silicon has been discussed earlier. The atoms of silicon are arranged in a diamond structure. This arises from two face-centered cubic (FCC) lattices as shown in Fig. 3.1. It is seen that each silicon atom is linked with four other atoms via covalent bond. The constellation of atoms and bonds of silicon are different in different directions therefore, silicon exhibits anisotropy. The silicon lattice also exhibits cubic symmetry so mutually perpendicular directions and planes within the lattice are equivalent. In case of anisotropy, the compliance tensor has 81 constants in Eq. (3.8). If the material properties of silicon wafers are described in a fixed arbitrary direction, the cubic symmetry of silicon lattice may be exploited to give compliance in matrix form as shown in Eq.(3.8). The experimentally determined values of elastic constants of the compliance of silicon oriented in the  $\langle 100 \rangle$  directions is given in the literature [42] and shown in Table 3.2. An orthotropic material has two or three planes of symmetry, thus compliance of silicon may be described by the expansion of Eq. (3.8) as



shown, which will enable us to determine the Young's modulus ( $E$ ), shear modulus ( $G$ ) and Poisson's ratio ( $\nu$ ).

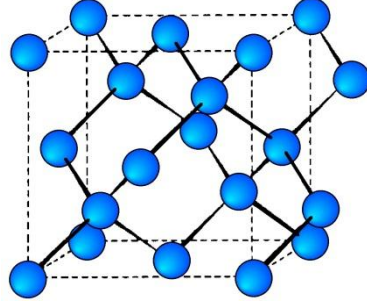


Fig. 3.1: Structure of Silicon crystal

Table 3.2: Elastic constants for the compliance of Silicon [42]

Compliance (S)	$s_{11}$	$s_{12}$	$s_{44}$
$\times 10^{12} \text{ Pa}$	7.68	-2.14	12.6

$$\{\epsilon^{el}\} = [S]\{\sigma\} \quad (3.8)$$

where,

$$[S] = \begin{bmatrix} s_{11} & s_{12} & s_{12} & 0 & 0 & 0 \\ s_{12} & s_{11} & s_{12} & 0 & 0 & 0 \\ s_{12} & s_{12} & s_{11} & 0 & 0 & 0 \\ 0 & 0 & 0 & s_{44} & 0 & 0 \\ 0 & 0 & 0 & 0 & s_{44} & 0 \\ 0 & 0 & 0 & 0 & 0 & s_{44} \end{bmatrix}$$

It can be further expanded to,

$$[S] = \begin{bmatrix} \frac{1}{E_x} & -\frac{\nu_{xy}}{E_x} & -\frac{\nu_{xz}}{E_x} & 0 & 0 & 0 \\ -\frac{\nu_{yx}}{E_y} & \frac{1}{E_y} & -\frac{\nu_{yz}}{E_y} & 0 & 0 & 0 \\ -\frac{\nu_{zx}}{E_z} & -\frac{\nu_{zy}}{E_z} & \frac{1}{E_z} & 0 & 0 & 0 \\ 0 & 0 & 0 & \frac{1}{G_{xy}} & 0 & 0 \\ 0 & 0 & 0 & 0 & \frac{1}{G_{yz}} & 0 \\ 0 & 0 & 0 & 0 & 0 & \frac{1}{G_{xz}} \end{bmatrix}$$

### 3.2.2 Glass

Glass structure used in PV modules is well transparent due to low iron content, so that a high transmission in usable range of the light spectrum (between 380 and 1200 nm) is achieved. The transmittance is almost 95%. Glass behaves as a completely elastic material and does not undergo plastic deformation within the module. It is also characterized by its excellent weather resistance, so that it prevents the effect of oxygen, water vapor and atmospheric pollution over the entire demanded period of 25 years. The thermal expansion coefficient is depends on the chemical composition of the glass being used.

### 3.2.3 Back-Sheet

For the back side isolation of crystalline solar modules there are a number of industrial solutions and within these solutions an important role is played by polyvinyl fluoride

(PVF) [43]. PVF in the early sixties was introduced by the plastic manufacturer DuPont<sup>TM</sup> under the trade name Tedlar [44]. PVF is characterized by low water absorption (<0.5%), good weather resistance, resistance to acids and alkalis, and has been tested over long periods of time in the outdoors. The film is designed for use between -72 ° C and 107 ° C [44]. It has a high electrical resistance and very good insulating properties. PVF is one of the few polymers that can be easily colored, with mostly white color is selected to achieve a high reflection [43].

Most commonly, the solar module manufacturers produce a PVF / PET / PVF three-layer composite used that combines the best qualities of both materials. Additional Polyethylene-terephthalate (PET) layer improves the electrical insulation. It also helps to reduce cost as PVF is expensive [43].

#### **3.2.4 Interconnects**

Steffen Wiese et al. [36] conducted experiments on copper ribbons used for interconnections between cells. The experiments helped them to provide the yield stress and Young's modulus of copper. First they performed tensile tests on Zwick (Fig. 3.2 (a)), which gave the stress-strain curve at room temperature. When DMA was carried out, it was found that the Young's modulus of copper changed with temperature as shown in Fig. 3.2 (b). Thus, they combined both the results to get the effective stress-strain curve of copper for a range of temperature between -40 to 125°C.

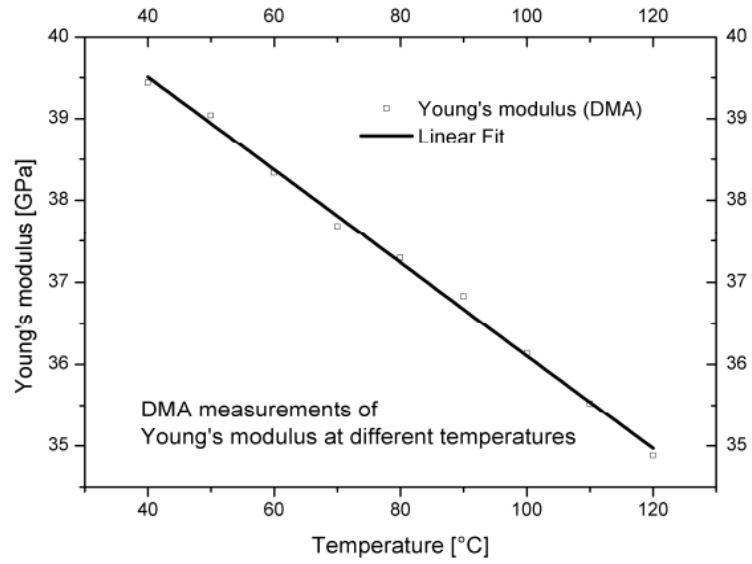
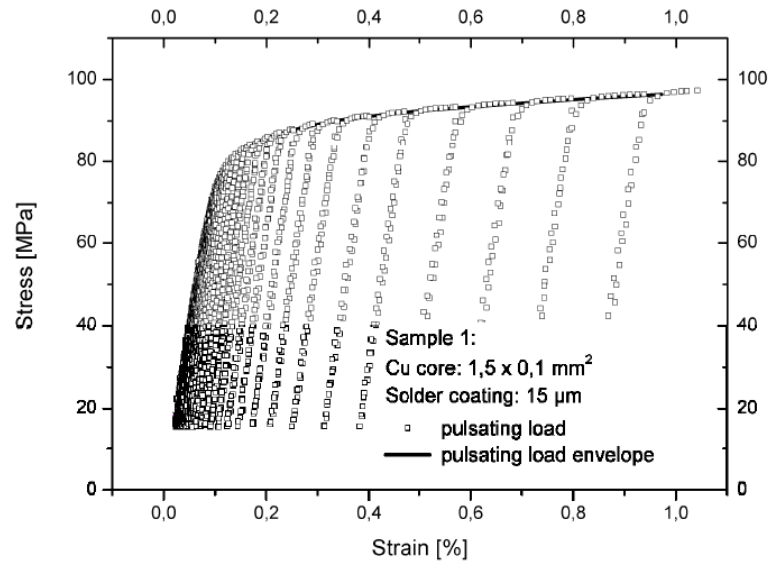


Fig. 3.2: (a) Tensile tests on copper interconnects performed on Zwick and (b) Temperature dependence of Young's modulus of copper interconnects found by DMA

[11]

### **3.2.5 Encapsulant**

The Encapsulant widely used in PV industry is Ethylene-vinyl acetate (EVA) copolymer. The properties of EVA are largely dependent on vinyl acetate content within the polymer. By increasing the vinyl acetate content, crystallization is hampered, thus increasing the elasticity and weather resistance, stress crack resistance, stickiness and flexibility. Due to its amorphous structure EVA is transparent to provide hurdle-free transmittance of light. Its curing takes place at about 150°C. At this temperature the EVA-melted and polymeric chains are cross-linked so that polymer network forms. After cooling, it results in a permanent association with the module, to protect cells from environmental influences. Furthermore, additives are provided to prevent yellowing and aging caused by UV radiation, light and heat.

## **3.3 VISCOELASTICITY**

Viscoelastic materials are such that exhibit the behavior of both elastic and viscous materials. It is a property of viscous materials to strain linearly with time when load is applied. In contrast to this, elastic materials strain on the spot when stressed and return to their original state when load is removed. A hysteresis can be observed in the stress strain curve of such material as energy is lost while returning to its initial state. Hence, stress relaxation phenomenon is observed when a viscoelastic material is kept under constant strain. Similarly, creep occurs when constant stress is applied (strain increases). The time taken for the molecular rearrangements in a viscoelastic material, after being stressed occur on a time-scale comparable to that of the experiment performed on it. Therefore,

the relation between stress and strains cannot be just described by material constants as it is in the case of purely elastic or viscous materials [45]. The behavior of viscoelastic materials is characterized by its time and temperature dependency.

Hence, the constitutive equation for a viscoelastic material in simple shear is given as [45],

$$\sigma = \int_0^t R(t-\tau) \frac{d\varepsilon}{d\tau} d\tau \quad (3.9)$$

$\sigma$  = Cauchy stress

$\varepsilon$  = Deviatoric strain

$R(t)$  = Shear relaxation modulus

$t$  = Current time

$\tau$  = Pseudo time

### 3.3.1 Maxwell's Model

The shear relaxation modulus of viscoelasticity can be approximated by Maxwell's model through spring and dashpot configuration as shown in Fig. 3.3 (a). The spring depicts elastic part of a viscoelastic material whereas the dashpot represents the viscous part. Eq. (3.10) and Eq. (3.11) give the constitutive relation (disregarding the tensorial character of stress and strain) in shear for spring and dashpot respectively.

$$\sigma_1(t) = G\varepsilon_1(t) \quad (3.10)$$

$$\sigma_2(t) = \eta\dot{\varepsilon}_2(t) \quad (3.11)$$

where,

$\eta$  = viscosity of the dashpot

$\dot{\varepsilon}_2(t)$  = rate of strain

For the elements attached in series as shown in Fig. 3.3 (a),

$$\sigma = \sigma_1 = \sigma_2$$

$$\dot{\varepsilon} = \dot{\varepsilon}_1 + \dot{\varepsilon}_2 \quad (3.12)$$

It is understood that the stress and strains in the formulation are functions of time hence; it is not shown in some parts of the derivation for convenience. Here  $\sigma$  and  $\varepsilon$  represent the total stress and strain of the system. Taking time rate of change of Eq. (3.10) and substituting the value of  $\dot{\varepsilon}_1$  in Eq. (3.12). Also, substituting  $\dot{\varepsilon}_2$  from Eq. (3.11) to Eq. (3.12). This gives [46],

$$\dot{\varepsilon}(t) = \frac{\dot{\sigma}(t)}{G} + \frac{\sigma(t)}{\eta} \quad (3.13)$$

Eq. (3.13) gives the constitutive relation for Maxwell's model.

Now, consider a cylindrical block of viscoelastic material loaded in uniaxial direction such that the strain is held constant. Then relaxation modulus of the spring can be given as,

$$R_1(t) = \frac{\sigma_1(t)}{\varepsilon_1} = \frac{G\varepsilon_1}{\varepsilon_1} = G(t)$$

And for the dashpot,

$$\sigma_2(t) = \eta \frac{d\varepsilon_2}{dt} = 0$$

$$R_2(t) = \frac{\sigma_2(t)}{\varepsilon_2} = 0$$

Substituting constant strain  $\varepsilon_o$  in Eq. (3.13),

$$0 = \frac{\dot{\sigma}}{G} + \frac{\sigma}{\eta}$$

Rearranging,

$$\frac{d\sigma}{dt} = -\frac{G\sigma}{\eta}$$



Integrating on both sides,

$$\int_{\sigma_o}^{\sigma} \frac{d\sigma}{\sigma} = - \int_0^t \frac{dG}{\eta} dt$$

$$\ln(\sigma) - \ln(\sigma_o) = - \frac{G}{\eta} t$$

$$\sigma(t) = \sigma_o e^{-\frac{G}{\eta} t} \quad (3.14)$$

Now, relaxation modulus for the whole system can be given as,

$$R(t) = \frac{\sigma(t)}{\epsilon_o}$$

Substituting  $\sigma(t)$  from Eq. (3.14),

$$R(t) = \frac{\sigma_o e^{-\frac{G}{\eta} t}}{\epsilon_o} = G e^{-\frac{G}{\eta} t}$$

But Relaxation time  $= \tau = \frac{\eta}{G}$  [45], therefore,

$$R(t) = G e^{-\frac{t}{\tau}}$$

For a generalized case with  $n$  number of springs and dashpots are connected in parallel with a single spring element (shown in Fig. 3.3 (b)), the relaxation modulus is expressed by Eq. (3.15). Here  $G_\infty$  is the shear modulus of the single spring element.

$$R(t) = G_\infty + \sum_{i=1}^n G_i e^{-\frac{t}{\tau_i}} \quad (3.15)$$

Eq. (3.15) takes the form of prony series. In our problem bulk modulus ( $K$ ) is assumed to be constant with time as in most polymers, viscoelastic effects are much stronger in shear. The number of spring-dashpot elements is represented by  $n$  in Eq. (3.15).

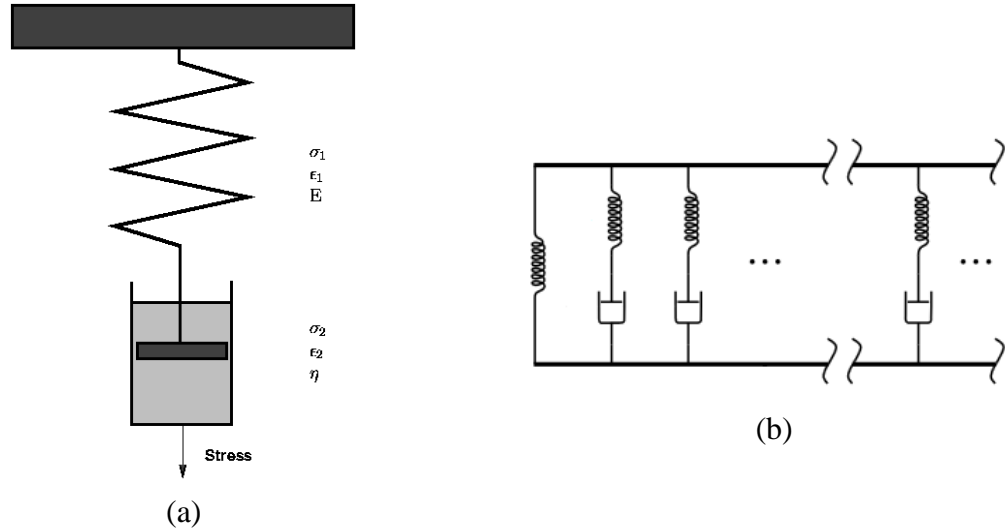


Fig. 3.3: (a) Maxwell's model and (b) Generalized Maxwell or Maxwell–Wiechert model

### 3.3.2 DMA and Relaxation Tests

Dynamic mechanical analysis (DMA) is used to characterize a material's response with respect to temperature and frequency by applying small cyclic deformations. It is useful

in the study of viscoelasticity in polymers. The outcomes of this procedure is the storage modulus ( $E'$ ) and the loss modulus ( $E''$ ). The cumulative effect of both the moduli is called the complex modulus ( $E$ ). The storage modulus gives the energy stored during deformation or the elastic part of viscoelasticity whereas, the loss modulus is the energy lost or converted to heat during deformation. Due to this phenomenon, a phase lag ( $\delta$ ) is observed:

$$\text{Storage modulus ( } E' \text{ )} = \frac{\sigma}{\varepsilon} \cos \delta$$

$$\text{Loss modulus ( } E'' \text{ )} = \frac{\sigma}{\varepsilon} \sin \delta$$

Similarly we also define shear storage and shear loss moduli,  $G'$  and  $G''$  .

Complex modulus is then expressed as:

$$E = E' + iE''$$

$$G = G' + iG''$$

The temperature or frequency of the sample is varied in order to identify their effect or to find the glass transition temperature. Inverse Fourier transform is used to find time dependence. Eitner et al. [41] performed DMA to investigate the dependency of EVA on the temperature. Their result is shown in Fig. 3.4 (a).

To perform a relaxation test a material specimen is simply held for a prolonged period of time by applying constant strain. The procedure is repeated at different temperatures to

develop a master curve for relaxation of the polymer. During the experiment, the time dependency of a viscoelastic polymer is analyzed. Results of relaxation tests performed by Eitner et al. [41] are given in Fig. 3.4 (b).

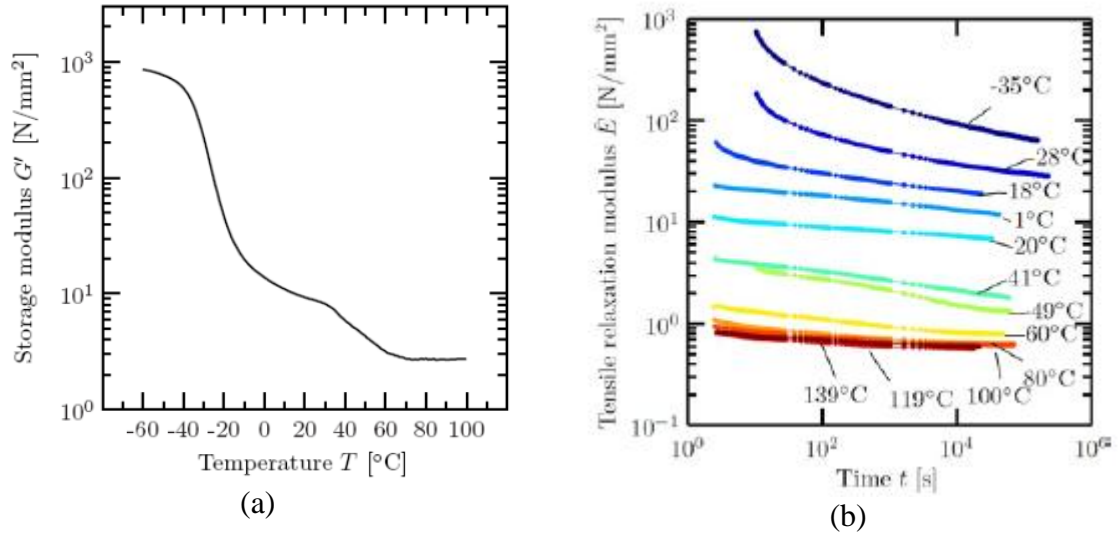


Fig. 3.4: (a) DMA experiment result for EVA and (b) Isothermal relaxation curves for EVA [41]

### 3.3.3 Time-Temperature-Superposition

With Time-Temperature Superposition (TTS) one can add temperature effects in the viscoelastic model. With the help of relaxation experiment at different temperatures, a single master curve may be formed by shifting others over the time scale. This process is done through shift function. In TTS, it is assumed that at higher temperature, relaxation occurs faster. This assumption is called “Thermorheologically simple”. So the shift function basically scales time to get pseudo time. If  $A[T(t)]$  is a shift function, then,

$$A[T(t)] = \tau / t$$

where,

$t$  = current time

$\tau$  = pseudo time

$$\log[A\{T(t)\}] = \log(\tau) - \log(t)$$

$$\log(\tau) = \log(t) + \log[A\{T(t)\}] \quad (3.16)$$

As depicted by Eq. (3.16), log of shift function represents the horizontal shifting of the master curve. The William-Landel-Ferry (WLF) shift function is widely used and is given by Eq. (3.17).

$$\log A[T(\tau)] = \frac{-C_1(T - T_r)}{C_2 + T - T_r} \quad (3.17)$$

where,

$T_r$  = Reference temperature at which the master curve is obtained

$C_1, C_2$  = Material constants

When the reference temperature is chosen to be equal to glass transition temperature for the polymer under consideration, then in most cases,  $C_1 = 17.44$  and  $C_2 = 51.6K$ . It should be noted that all temperatures must be Kelvin. Temperatures less than  $C_2 - T_r$  should be avoided as it is the limit below which response of the material is fully elastic [47].

### 3.4 STRUCTURAL MODELING

Table 3.1 gives the comparison of the material modeling used in the literature. It can be seen that when only cells are modeled, details such as silver and aluminum paste are also provided to the FE package. But as in the current work the whole PV module is modeled, therefore, silver and aluminum are not included considering their effect to be negligible. The later sections provide the reasons for model selection for each material.

#### 3.4.1 Silicon

By using Table 3.2, one can easily find values of Young's modulus, Poisson's ratio and shear modulus described by Eq. (3.18), Eq. (3.19) and Eq. (3.20).

$$E_{(100)} = \frac{1}{s_{11}} = 130GPa \quad (3.18)$$

$$\nu_{(100),(010)} = \nu_{(100),(001)} = \frac{-s_{12}}{s_{11}} = 0.28 \quad (3.19)$$

$$G_{(100),(010)} = \frac{1}{s_{44}} = 79.5 GPa \quad (3.20)$$

Regarding, the CTE of silicon, it varies with temperature as shown in Fig. 17.

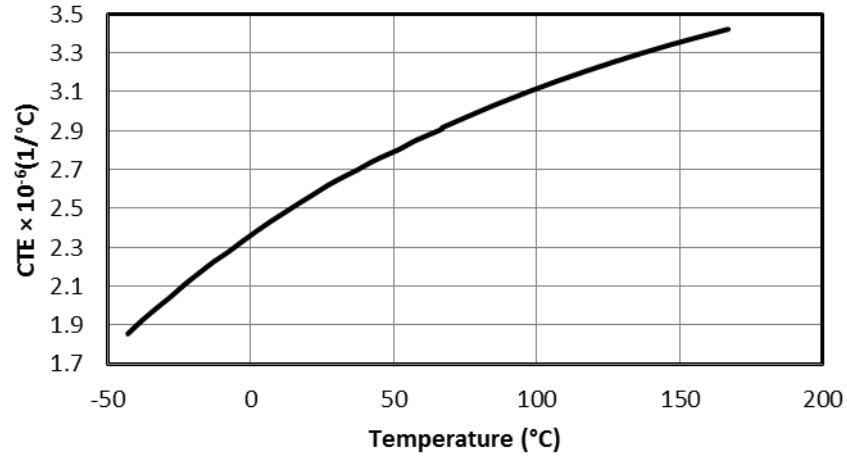


Fig. 3.5: CTE of Silicon vs. Temperature [51,52]

The CTE values in Fig. 3.5 have obtained through experiments performed by K.G. Lyon et al. [51] and R.B. Roberts [52]. Hence, its dependence on temperature cannot be ignored. The density is taken as 2329 kg/m<sup>3</sup> [37].

### 3.4.2 Copper

Dietrich et al. [53] concluded that interconnects undergo plastic strain when cooled from lamination temperature. Wiese et al. [11,26] found that the stress-strain curve of copper can be approximated by bilinear model so that computation can be simplified. By including the temperature dependence of Young's modulus through DMA and the stress-

strain curve at room temperature, they developed a model shown in Table 3.3, which can be used for FE simulation.

CTE of copper also depends on temperature (shown in Fig. 3.6) and its temperature dependence is provided in the literature [54]. Density of copper is given as  $8890 \text{ kg/m}^3$ .

Table 3.3: Bilinear elastic-plastic model for Copper [26]

Temperature	Young's Modulus	Yield Stress	Tangent Modulus
(°C)	(GPa)	(MPa)	(MPa)
-40	91.5	116.2	1000
25	85.7	95.1	1000
125	82	62.6	1000
225	79.2	30	1000



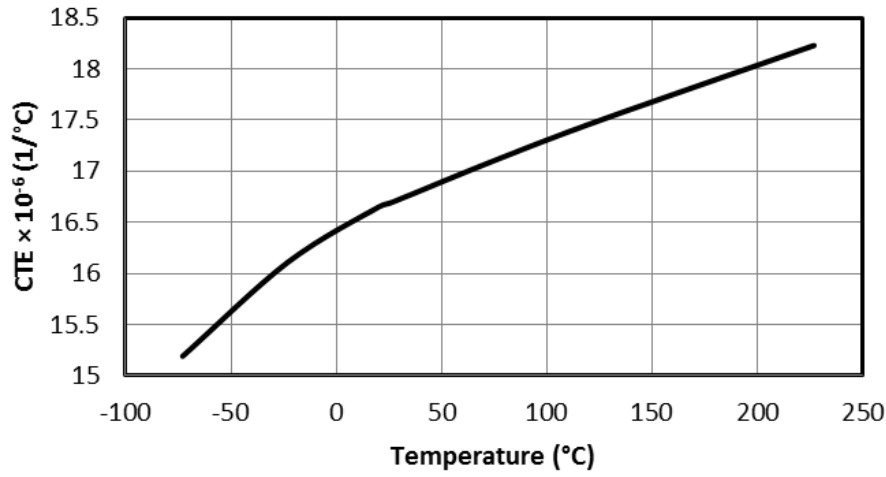


Fig. 3.6: CTE of Copper vs. Temperature [54]

### 3.4.3 Glass and PVF/PET/PVF Composite

The CTE for soda-lime glass was looked into the literature [55] and was found to be almost constant throughout the temperature range of -40 to 150°C. It is preferred to use a constant CTE for glass which is equal to  $8 \times 10^{-6}$  (1/°C). It is modeled as isotropic linear elastic with its Young's modulus equal to 73 GPa and Poisson's ratio equal to 0.23. The density is provided as 2500 kg/m<sup>3</sup> [8].

Tensile tests were performed by Eitner et al. [8] in order to determine the mechanical properties at different temperatures for PVF / PET / PVF. The back-sheet used was Isovolt Icosolar 2442 and it found that the Young's modulus did not show significant changes with the change in temperature thus its value was taken as 3.5 GPa with

Poisson's ratio equal to 0.29. The density and the CTE is given as  $2520 \text{ kg/m}^3$  and  $50.4 \times 10^{-6} (1/^\circ\text{C})$  respectively.

### 3.4.4 Ethylene-Vinyl Acetate

Viscoelasticity of EVA has been discussed in detail in the earlier section where Eq. (3.15) was found to be as the relaxation modulus given by prony series. ANSYS interprets relaxation modulus in terms of relative modulus ( $\alpha_i$ ) where,

$$\alpha_i = \frac{G_i}{G_o} \quad (3.21)$$

and,

$$G_o = G_\infty + \sum_{i=1}^n G_i \quad (3.22)$$

By substituting, Eq. (3.21) and Eq. (3.22) in Eq. (3.15), we get,

$$R(t) = G_o \left[ \alpha_\infty + \sum_{i=1}^n \alpha_i e^{-\frac{t}{\tau_i}} \right] \quad (3.23)$$

where,

$$\alpha_\infty = \frac{G_\infty}{G_o}$$

To fit the prony series mentioned in Eq. (3.23), experimental results can be used. A value of  $n$  (number of spring-dashpot elements) is chosen with guess values for  $\alpha_i$ , and  $\tau_i$ . Residuals are calculated and brought to minimum by a number of iterations. For this process, a master curve plotted by relaxation experiments and WLF shift function can be

used. In this case, the master curve in Fig. 3.7 was used. Table 3.4 provides all the viscoelastic properties used for EVA.

The application of the procedure gives the Prony series fit for the Maxwell's model of 25 arms. The relative shear moduli and the pseudo time found can be directly fed into ANSYS. Now, the issue is to find the instantaneous shear modulus ( $G_0$ ). From  $\alpha_i$  obtained from the curve fitting procedure, it is given that,

$$\alpha_{\infty} = 1 - \sum_{i=1}^n \alpha_i \quad (3.24)$$

For each data point of the master curve, the value of instantaneous shear modulus can be calculated. For a proper curve fit, this value will be almost same for each data point or an average may be taken. The density of EVA is taken as  $960 \text{ kg/m}^3$ .

Table 3.4: Viscoelastic properties for EVA

$C_1$	48.44	$\phi_6$	0.0136822	$\phi_{13}$	0.0015563	$\phi_{20}$	0.0005415
$C_2$	172.55 K	$\tau_6$	15.7945	$\tau_{13}$	$10^9$	$\tau_{20}$	$10^{16}$
$T_r$	253 K	$\phi_7$	0.0105574	$\phi_{14}$	0.0023002	$\phi_{21}$	0.0005575
$E_o$	1.3 GPa	$\tau_7$	235.0052	$\tau_{14}$	$10^{10}$	$\tau_{21}$	$10^{17}$
$\phi_1$	0.5467175	$\phi_8$	0.0037958	$\phi_{15}$	0.0008377	$\phi_{22}$	0.0003087
$\tau_1$	0.0001219	$\tau_8$	10333.19	$\tau_{15}$	$10^{11}$	$\tau_{22}$	$10^{18}$
$\phi_2$	0.2222377	$\phi_9$	0.002486	$\phi_{16}$	0.0013597	$\phi_{23}$	0.000177
$\tau_2$	0.0007823	$\tau_9$	99967.33	$\tau_{16}$	$10^{12}$	$\tau_{23}$	$10^{19}$
$\phi_3$	0.0992664	$\phi_{10}$	0.0010978	$\phi_{17}$	0.0013157	$\phi_{24}$	1.122E-05
$\tau_3$	0.0063471	$\tau_{10}$	1000001	$\tau_{17}$	$10^{13}$	$\tau_{24}$	$10^{20}$
$\phi_4$	0.0590673	$\phi_{11}$	0.0021774	$\phi_{18}$	0.00076	$\phi_{25}$	1.536E-08
$\tau_4$	0.075255	$\tau_{11}$	$10^7$	$\tau_{18}$	$10^{14}$	$\tau_{25}$	$10^{21}$
$\phi_5$	0.0265249	$\phi_{12}$	0.0010461	$\phi_{19}$	0.0011088	$\phi_{26}$	6.337E-06
$\tau_5$	1.261626	$\tau_{12}$	$10^8$	$\tau_{19}$	$10^{15}$	$\tau_{26}$	$10^{22}$

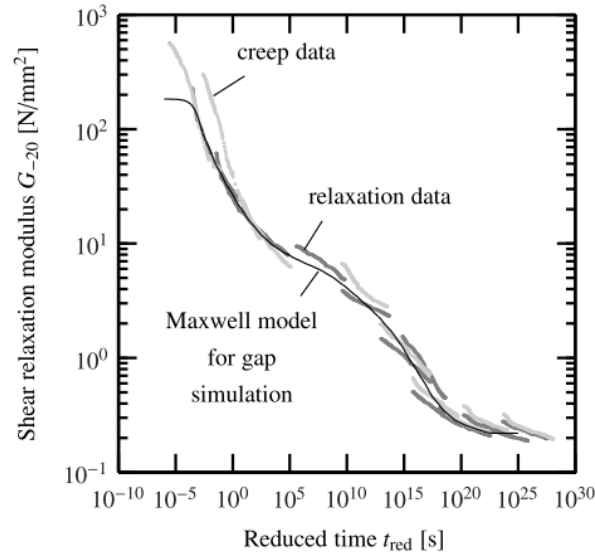


Fig. 3.7: Prony series fit of the Master curve in [41]

## 3.5 RADIATION AND OPTICAL MODELING

As the name indicates, it is a combination of two models. The radiation model is used to calculate the plane of array irradiance from the measured horizontal solar irradiance. The optical model is used to estimate the amount of plane of array irradiance absorbed on a respective surface.

### 3.5.1 Optical Model

Eq. (3.25) is used to calculate the transmittance-absorptance product ( $\tau\alpha$ ). Here  $\theta$  and  $\theta_r$  are the incidence and refraction angles,  $K$  is the extinction coefficient and  $L$  is the thickness of the glass cover [56]. Eq. (3.26) is used to calculate the incidence angle modifiers ( $K_{\tau\alpha}$ ) by using the ( $\tau\alpha$ ) product. It should be noted that separate incidence angle

modifiers are required for beam, diffuse and ground reflected components of the incident solar radiation.

$$\tau\alpha(\theta) = e^{-(KL/\cos\theta_r)} \left[ 1 - \frac{1}{2} \left( \frac{\sin^2(\theta_r - \theta)}{\sin^2(\theta_r + \theta)} + \frac{\tan^2(\theta_r - \theta)}{\tan^2(\theta_r + \theta)} \right) \right] \quad (3.25)$$

$$K_{\tau\alpha}(\theta) = \frac{\tau\alpha(\theta)}{\tau\alpha(0)} \quad (3.26)$$

### 3.5.2 Radiation Model

The radiation model used in the current work is the Hay-Davies-Reindl-Klutcher (HDKR) model and is given by Eq. (3.27). Here,  $S$  is the absorbed solar radiation,  $G$  is the horizontal plane solar radiation,  $R_{beam}$  is the ratio of beam radiation on tilted plane to that on horizontal plane,  $\rho$  is the ground reflectivity,  $\beta$  is the tilt angle of PV module,  $A_i$  is the anisotropy index (given by Eq. (3.28)) and  $M$  is the air mass modifier. The subscripts  $b$ ,  $d$ ,  $g$  and  $ref$  are for the beam, diffuse, ground reflected and reference solar radiations respectively.

$$\begin{aligned} \frac{S}{S_{ref}} = & M \frac{G_b + A_i G_d}{G_{ref}} R_{beam} K_{\tau\alpha,b} \\ & + M \frac{(1 - A_i) G_d}{G_{ref}} K_{\tau\alpha,d} \left( \frac{1 + \cos \beta}{2} \right) \left( 1 + f \sin^3 \left( \frac{\beta}{2} \right) \right) \\ & + M \frac{G}{G_{ref}} \rho K_{\tau\alpha,g} \left( \frac{1 - \cos \beta}{2} \right) \end{aligned} \quad (3.27)$$

where the factor  $f$  and  $S_{ref}$  are given by Eq. (3.29).and Eq.(3.30) respectively.

$$A_i = \frac{G_b}{G_o} \quad (3.28)$$

$$f = \sqrt{\frac{G_b}{G}} \quad (3.29)$$

$$S_{ref} = (\tau\alpha)_n G_{ref} \quad (3.30)$$

### 3.6 THERMAL MODELING

Thermal modeling is done in ANSYS by providing various modes of energy transfer, as shown in Fig. 3.8. The application of these modes are explained in the loads and boundary conditions section. The PV module gains energy by absorbing the incoming solar radiation. Some of the energy is lost due to convection by wind on the top and bottom surfaces. Some of it is lost through radiation to the environment. The energy is also used to convert thermal energy to electrical energy by cells and the rest of the energy is used up in the heating of the module. The combined thermal and structural properties for the components of the PV module are summarized by Table 3.5.

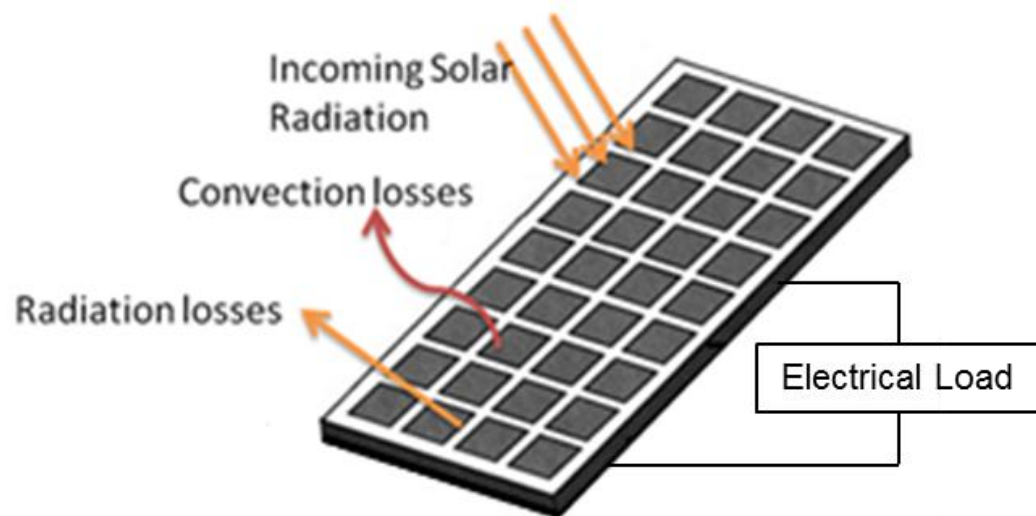


Fig. 3.8: Modes of energy transfer in a PV module



Table 3.5: Material properties of module components. *T. dep.* stands for temperature dependent, *BISO* stands for bilinear isotropic

<b>Component</b>	<b>Density</b>	<b>Elastic</b>	<b>Poisson's</b>	<b>CTE</b>	<b>Specific</b>	<b>Thermal</b>
		<b>Modulus</b>	<b>ratio</b>		<b>heat</b>	<b>conductivity</b>
	$\rho$ (kg/m <sup>3</sup> )	$E$ (GPa)	$\nu$	$\alpha$ (10 <sup>-6</sup> 1/K)	$C$ (J/kg K)	$k$ (W/m K)
<b>Silicon</b>	2329	<i>Stiffness matrix</i>		<i>T. dep.</i>	677	130
<b>Backsheet</b>	2520	3.5	0.29	50.4	1010	0.36
<b>Glass</b>	2500	73	0.23	8	913	0.937
<b>EVA</b>	960	<i>Viscoelastic model</i>		270	2090	0.311
<b>Copper</b>	8890	<i>BISO</i>		<i>T. dep.</i>	386	401

### 3.7 ELECTRICAL MODELING

The electrical model used was developed by Siddiqui et al. (2013) in which a PV device is represented by an equivalent electric circuit of Fig. 3.9 (Duffie and Beckman, 1991). The governing equation for current-voltage relationship for the PV devices is given by Eq. (3.31).

$$I = I_L - I_o \left( \exp \left( \frac{V + I \cdot R_s}{a} \right) - 1 \right) - \frac{V + I \cdot R_s}{R_{sh}} \quad (3.31)$$

The model is used by determining the parameters  $I_L$ ,  $I_o$ ,  $a$ ,  $R_s$  and  $R_{sh}$  at a reference condition. Then these are translated to the operating condition using the translation equations (3.32)-(3.36). The parameters  $m$  and  $n$  are determined using two additional maximum power values at a higher temperature and a lower irradiance.

$$a = a_{ref} \left( \frac{T_{cell}}{T_{cell,ref}} \right)^n \quad (3.32)$$

$$I_L = \left( \frac{S}{S_{ref}} \right)^m \left( I_{L,ref} + \mu_{isc} (T_{cell} - T_{cell,ref}) \right) \quad (3.33)$$

$$I_o = I_{o,ref} \left( \frac{T_{cell}}{T_{cell,ref}} \right)^3 e^{\left( \frac{NCS \cdot T_{cell,ref}}{a_{ref}} \left( \frac{E_{g,ref}}{T_{cell,ref}} - \frac{E_g}{T_{cell}} \right) \right)} \quad (3.34)$$

$$R_{sh} = \frac{S_{ref}}{S} R_{sh,ref} \quad (3.35)$$

$$R_s = R_{s,ref} \quad (3.36)$$

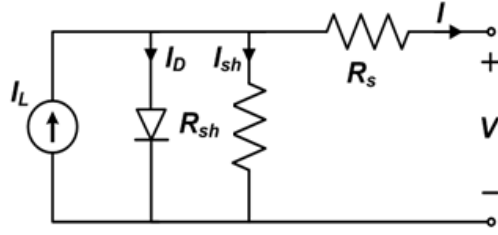


Fig. 3.9: Equivalent circuit of an actual PV cell

### 3.8 GEOMETRIC MODELING

The layers of a PV module are very thin as compared to their lengths. Therefore, solving a 3D problem of such nature over the whole module would take time in days which is inappropriate. To resolve this issue, shell modeling was a perfect option as it idealizes the problem to 2D. Shell elements in ANSYS have the ability to solve problems from thin to moderately thick structures [59]. The multilayer definition ability in shell helped to provide dimensions and material properties along the thickness of the laminate. It evaluates the results in one plane and interpolates them along the thickness. Thus, stresses and strains can be viewed in each layer.

Fig. 3.10 shows the overall module area ( $0.546 \text{ m} \times 1.181 \text{ m}$ ) along with its dimensions. The area of a single cell is  $125 \text{ mm} \times 125 \text{ mm}$ . The gap between two cells is 2 mm and 20 mm from the edge of the module. The space between two strips of interconnection is 77 mm. It is seen that the whole module area has been constructed by smaller sections (separated by lines) and are merely of four types.

- (i) Areas representing the cell region,
- (ii) Areas representing the interconnect region along the cells,
- (iii) Areas representing the interconnect region within the cell gap and
- (iv) Rest of the module area.

Different layered configuration, along the thickness of the module, is defined for each section. Some of them are shown in Fig. 3.11. The section type (iii) modeling consists of interconnects. The interconnects, within the cell gaps, have a curved slanted profile as shown in Fig. 3.11. To approximate such profile along the thickness, the cell gap region is further divided into 14 sections. These sections are defined to constitute a layer of copper which is positioned in adjacent layers in such a way that it produced almost the same profile. This is done by varying the thickness of encapsulant layers within these sections as depicted in Fig. 3.12. The thickness of each layer within the PV laminate is mentioned in Table 3.6.

Table 3.6: Thickness of layers within the PV laminate

<b>Layer</b>	<b>Thickness (<math>\mu\text{m}</math>)</b>
Glass	4000
Cell	200
Encapsulant	1200
Back-Sheet	350
Interconnector	129

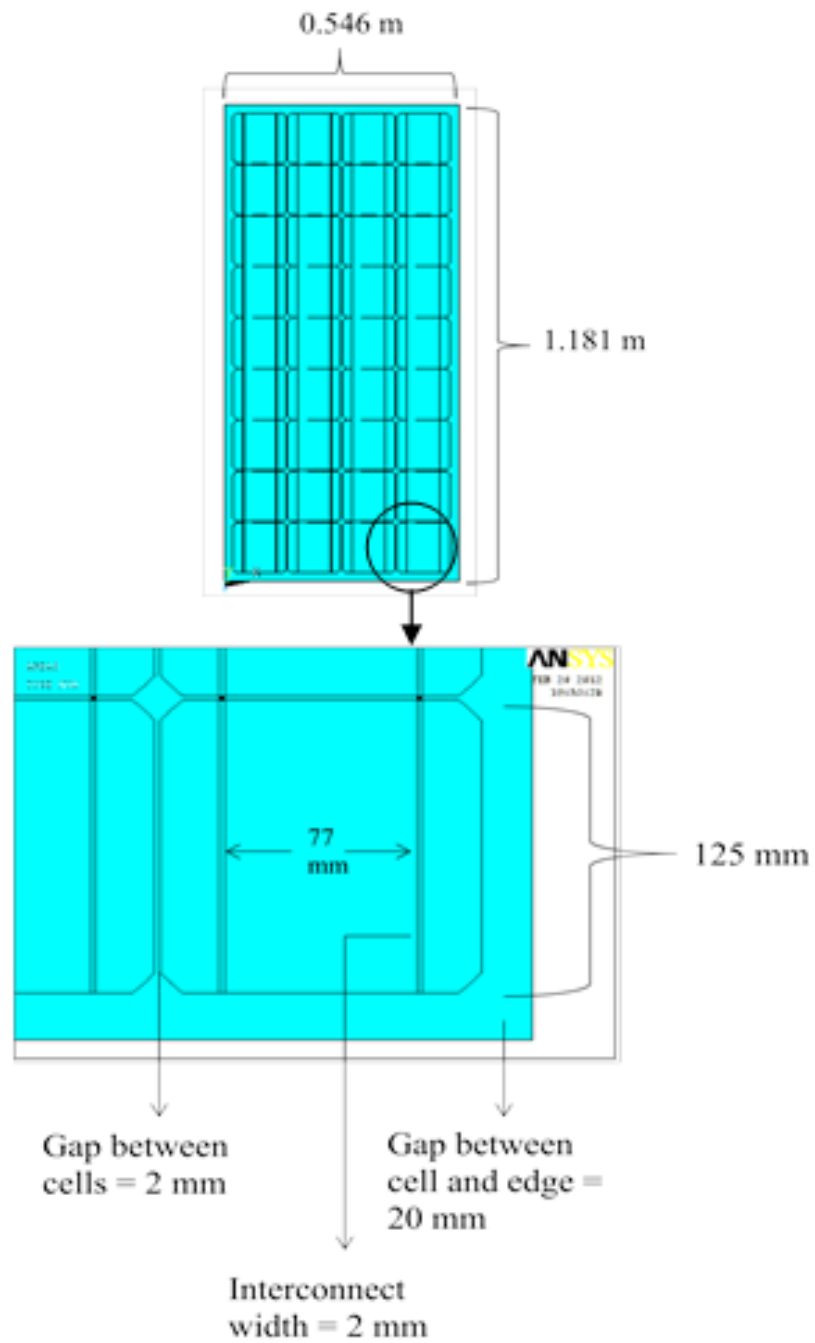


Fig. 3.10: Dimensions of the shell model

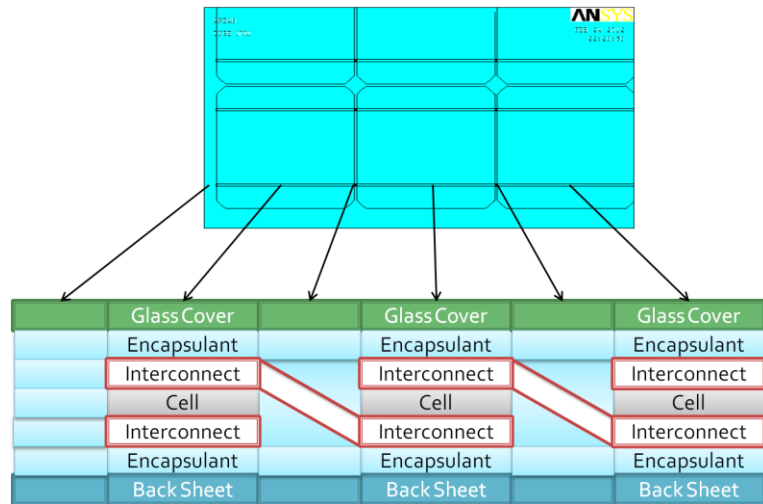


Fig. 3.11: Layered configuration of areas along transverse direction

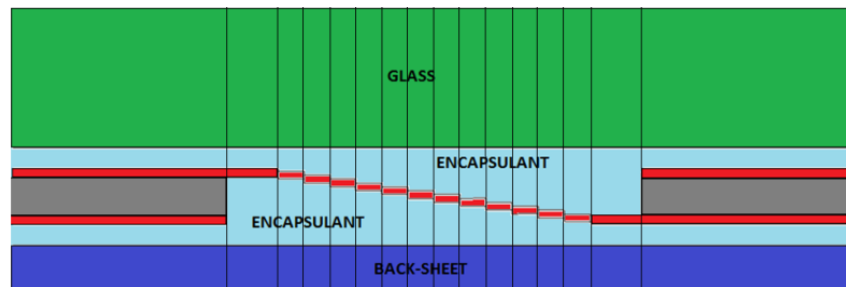


Fig. 3.12: Interconnection approximation in shell model

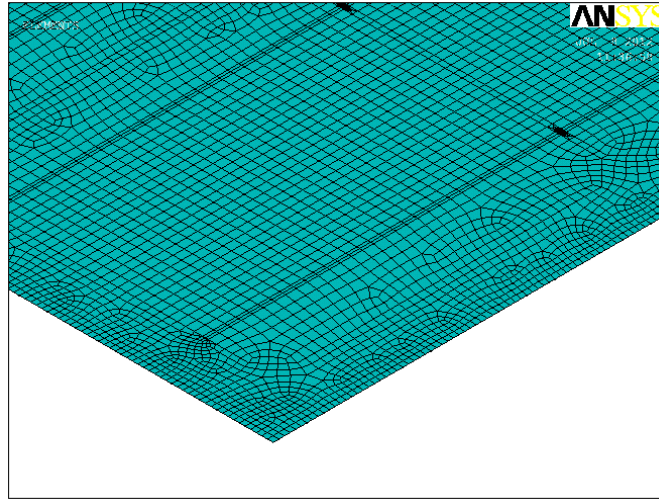


Fig. 3.13: FE mesh of the geometric model for PV module

### 3.9 MESHING

As shown in Fig. 3.13, the FE mesh consists of four node shell elements. SHELL131 was used for thermal analysis and its counterpart SHELL181 was used to solve the structural problem. Mesh convergence test was performed with respect to maximum von-Mises stress within each component of the PV module. The converged mesh had 83,351 elements as shown in Table 3.7. It is evident that the same geometry and mesh was used to couple thermal model with structural model.

Table 3.7: Mesh convergence with respect to maximum von-Mises stress in all layers

<b>No. of Elements</b>	<b>Max. von-Mises stress (MPa)</b>			
	<b>Glass cover</b>	<b>Backsheet</b>	<b>Cell</b>	<b>Interconnects</b>
8,394	6.41	41.2	212	160
16,356	9.31	41.2	219	162
32,627	9.23	41.1	217	162
59,162	9.24	41.2	218	162
83,351	9.24	41.2	218	162



## **CHAPTER 4**

### **MODEL VALIDATION**

The structural model developed in Chapter 3 is validated in this chapter. The validation is done in two steps. The first step deals with the development of a 3D FE solid model. This model is then given the inputs of an experiment in the literature. The results were found to be in good agreement. In the second step, the stresses in the solid model were compared to that of the shell model (the model to be used in the next chapter) to find similarity between them. This is depicted in Fig. 4.1. Although it is mentioned in Chapter 3 that shell elements ANSYS will be used for analysis to save computational time, but displacement of each component cannot be viewed using shell elements. These elements instead compute the cumulative displacements and estimate stresses on each layer by forming an equivalent stiffness matrix.

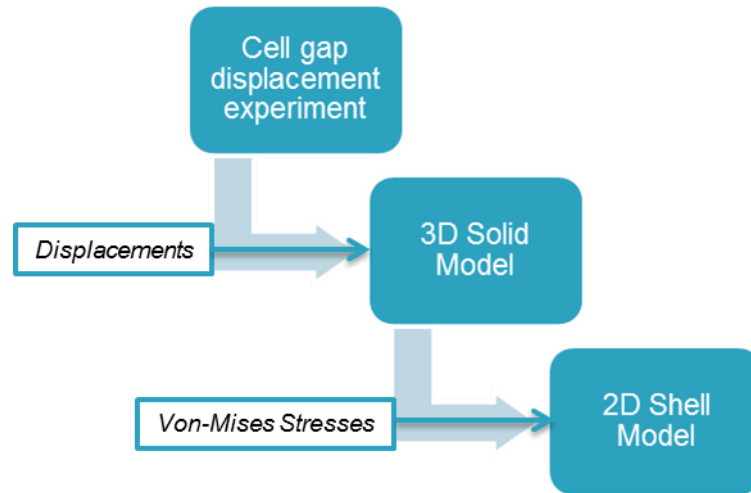


Fig. 4.1: Flowchart of the validation process

## 4.1 CELL-GAP DISPLACEMENT EXPERIMENT

The cell-gap displacement is basically the change of cell gap due to temperature change. Eitner et al. in [38] applied digital image correlation technique to measure the thermo-mechanical displacements in cells due to heating and cooling of a module. Fig. 4.2 gives a schematic diagram to understand the setup. As shown, two cameras are attached to take pictures of the specimen. The measurement of displacement is done by comparing the pictures in reference state and in the loaded state through a computer algorithm. A three cell sample is prepared with speckled surface for measurement purposes. The three cells are then laminated by EVA, attaching glass and backsheet. The lamination is done by heating the assembly to 150°C for 13 minutes. The cells were not interconnected. The laminate is 40 cm × 15 cm. The cells had a dimension of 125 mm ×

125 mm and were placed at a distance of 2 mm from each other. The thicknesses of each component are given in Table 4.1.

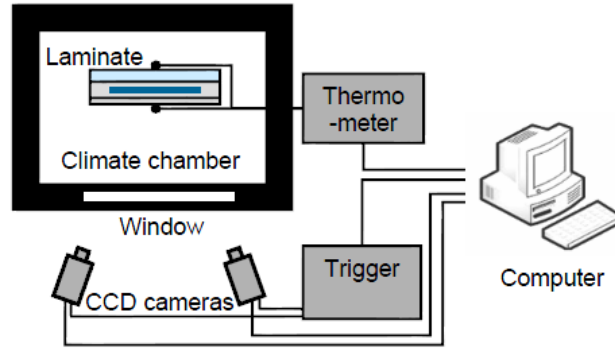


Fig. 4.2: Digital image correlation experimental setup to measure cell-gap displacement

[38]

Table 4.1: Thicknesses of the components of the PV module specimen

Component	Thickness
	( $\mu\text{m}$ )
Glass	4000
Cell	200
Encapsulant	1100
Backsheet	100

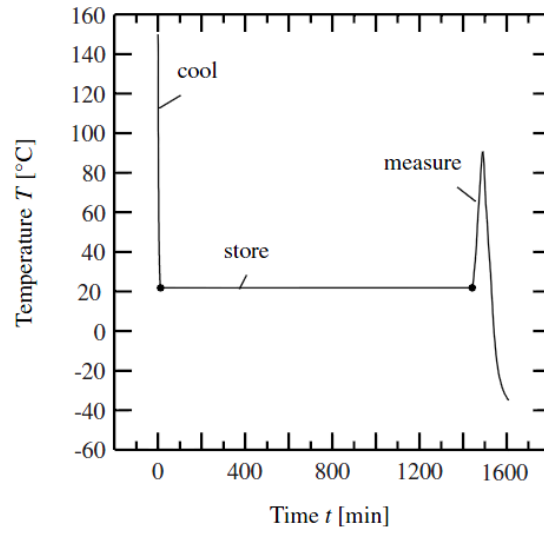


Fig. 4.3: The temperature history of the cell gap displacement experiment [37]

Fig. 4.3 gives the complete temperature history under which the 3 cell module was experimented. As depicted by the figure, first the curing of the encapsulant was done by cooling it from 150°C to room temperature of 23°C. Then the specimen was stored for 24 hours under constant room temperature. After that the module was first heated to 85°C and then cooled to -40°C. The cell gap displacements were recorded.

## 4.2 SOLID MODEL

In [37], Eitner et al. evaluate different material models for EVA by comparing the results for cell gap displacements in experiment and simulation. In the current work, the same methodology has been adopted to reproduce similar validation results. As seen in Fig. 4.4, the shape of cells is square instead of pseudo square. It was done in order to create assistance in mapped meshing of the components of the module.

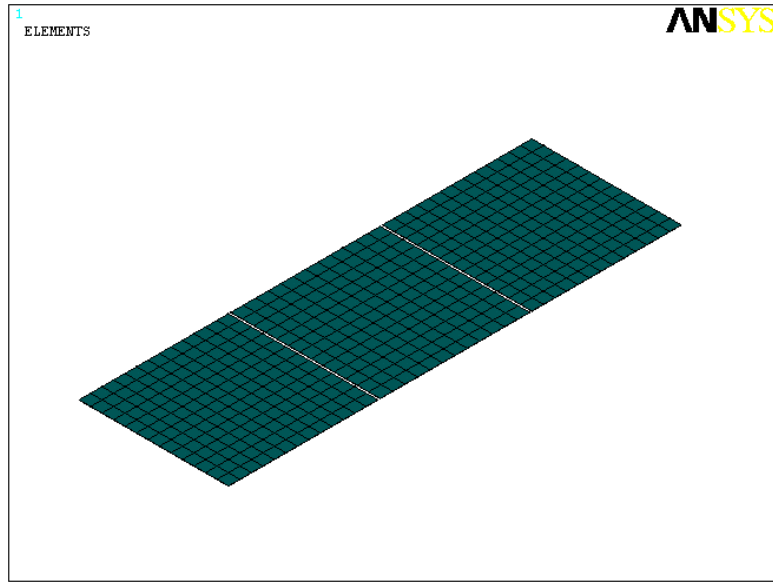


Fig. 4.4: Solid model representing cells without glass, encapsulant and backsheet

#### 4.2.1 Linear Elastic Model for EVA

As described in [37], the first part was done by modeling the EVA encapsulant as linear elastic. Fig. 4.5 shows the comparison of the two linear elastic models used for EVA with experimental data of cell gap displacement. Eitner et al. [37] determined that the Young's modulus of elasticity ( $E$ ) is not the same for EVA and changes with time and temperature. Therefore, highest and the lowest values of  $E$  were chosen for comparison which were 2.1 GPa and 6.5 MPa respectively. The same was done in the current work and a great difference was found between the actual and the simulated results as shown in Fig. 4.5. The reference temperature was set as 150°C i.e. the temperature of zero strain.

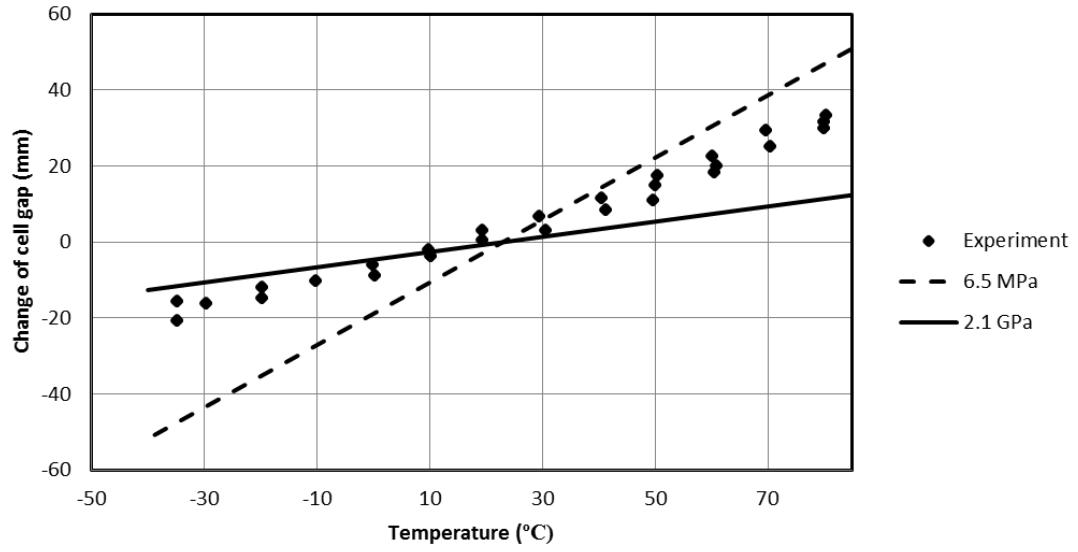


Fig. 4.5: Comparison of the two linear elastic models with experimental data

#### 4.2.2 Viscoelastic Model for EVA

The results mentioned in the previous section confirmed the viscoelastic behavior of EVA, therefore EVA was modeled and implemented with the procedure mentioned in Chapter 3. The cell gap displacements were calculated (Fig. 4.6) and compared with the experimental results performed in [38]. It is actually the difference of the average displacement of the nodes on the center of the edge along the thickness between the two adjacent cells. A good agreement is seen between both experimental and the simulation outcomes.

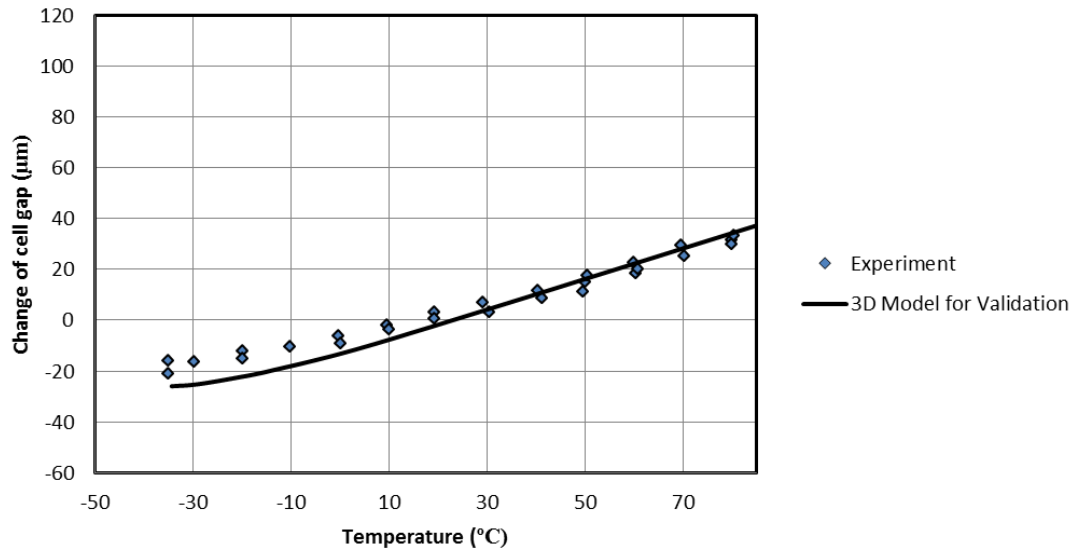


Fig. 4.6: Comparison of the viscoelastic model of EVA with the experimental results

### 4.3 SHELL MODEL

The purpose of the work in this chapter is to validate the shell model so that it can be used for FE simulation of a whole PV module. As discussed earlier, the shell model calculates a cumulative displacement of all layers and therefore, single layer displacements cannot be viewed. But the experiment for validation in [38] require displacements to be compared with the model and for this reason, a solid model was developed. Now the idea is to compare the solid model with the shell model under the same load and having the same geometric and material properties. To remove the effect of boundary condition, the boundaries of glass, encapsulant and backsheet were moved further from the cells causing an increase in the size of the specimen of both the solid and the shell model.

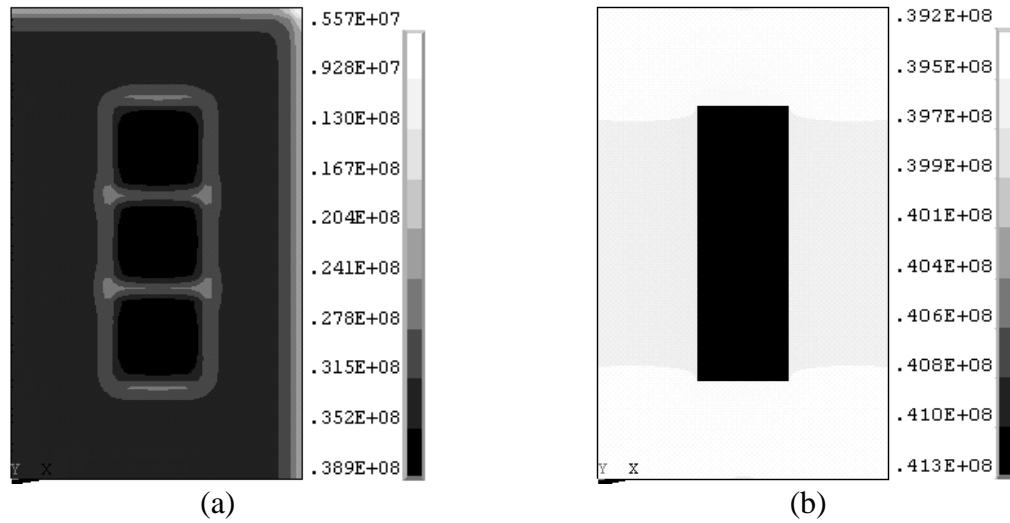


Fig. 4.7: Von-Mises stress contour of backsheet using (a) Solid Model and (b) Shell

Model

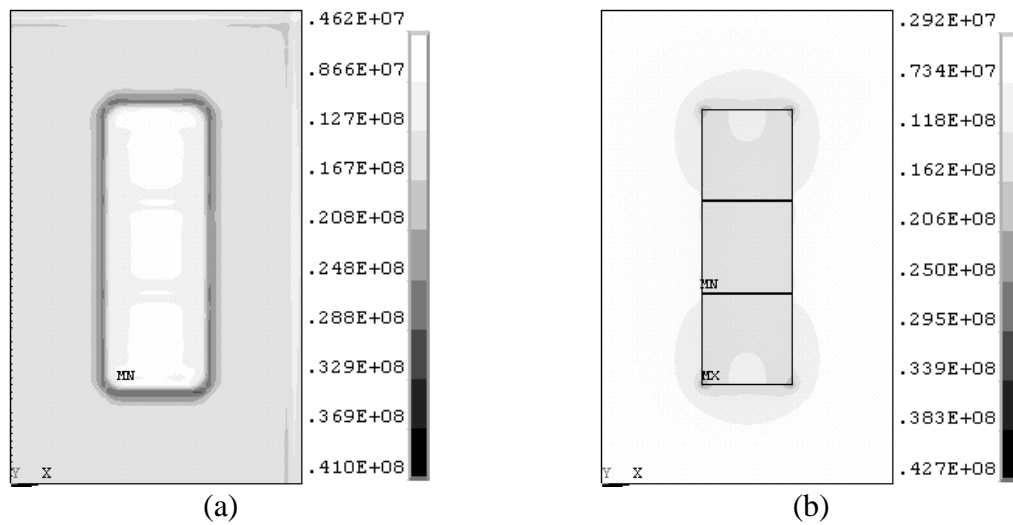


Fig. 4.8: Von-Mises stress contour of glass cover using (a) Solid Model and (b) Shell

Model



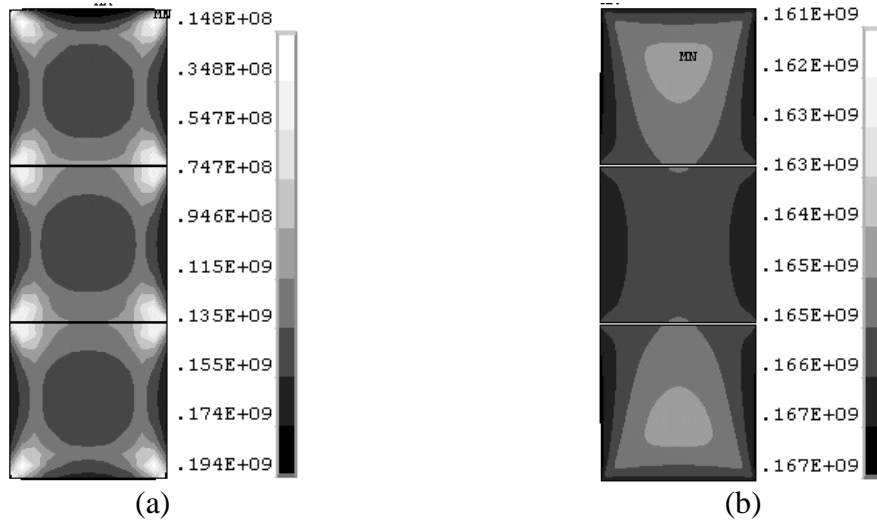


Fig. 4.9: Von-Mises stress contour of cells using (a) Solid Model and (b) Shell Model

Fig. 4.7 (a) and (b) are the von-Mises stress contours of solid and shell models for backsheet respectively. It is seen that the maximum stress is below the cell region and which are quite close for models. The error between the two increases towards the boundary of the model. It is attributed to boundary condition effect. The same can implied for glass cover and cells respectively in Fig. 4.8 and Fig. 4.9 respectively. A larger difference can be seen in the case of cells but this difference is subject to normalize for a 36 cell PV module model used in the later chapters. It can also be seen that the shell model gives a conservative estimate of stresses, which is beneficial if used for design purposes. Fig. 4.10 gives a comparison along the thickness of the module for both models over a point on the top surface away from the boundaries of the specimen. A good agreement can be seen for both the cases with maximum difference between them to be 6%.

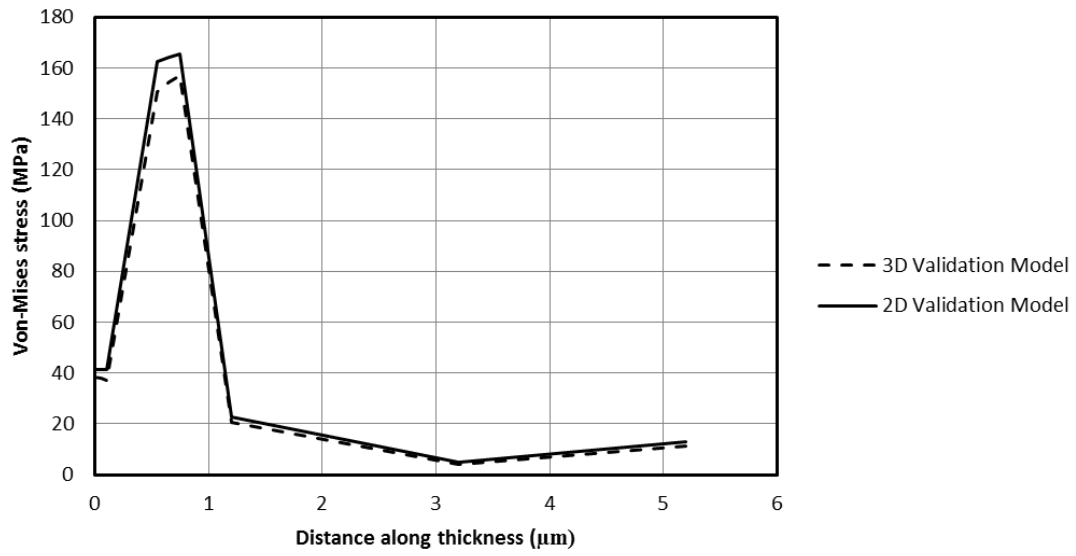


Fig. 4.10: Comparison of the von-Mises stress along the thickness of the module by using solid and shell models

## 4.4 CONCLUSIONS

Different models for EVA were used in FE simulation and the experimental results in the literature were compared. The similarity between the results of shell and solid model is also assessed to draw out the following conclusions:

- Viscoelastic model for EVA is a close estimate of its constitutive behavior, unlike the linear elastic model which gave a large deviation from actual behavior.

- Shell model is able to capture the response of PV module over loads as the solid model. It also provides conservative estimates, useful for design purposes.

# **CHAPTER 5**

## **QUALIFICATION TEST USING FINITE-ELEMENT**

### **MODEL**

A lot about this temperature cycle has been discussed in Chapter 1 and shown by Fig. 1.8. The cycle has a maximum temperature of 85°C and a minimum temperature of -40°C. Qualification standards such as ASTM E1171-09 are useful in predicting a module's failure. A single temperature cycle has been simulated in this chapter in order to get an insight on the behavior of the components of the PV module. For accurate prediction, the simulated temperature cycle starts from the lamination process of the module following 24 hour storage. At the end a parametric study has been performed to find the effect of the thickness of the encapsulant.

#### **5.1 LOADS AND BOUNDARY CONDITIONS**

The module is constraint at one corner to allow free deformation. This will help to study the pure dependence of materials on one another. The stress-free temperature is taken as the lamination temperature (150°C) because at that temperature every

component of the module is independent to one another thus allowing a stress-free expansion. Fig. 5.1 provides the simulated temperature profile. As seen, the simulation starts from the lamination temperature to room temperature of 21°C. Then the panel is assumed to be under 24 hour storage in order to incorporate the time effect of the viscoelastic model of EVA. Then a single temperature cycle of the ASTM E1171-09 standard is simulated and results are viewed at -40°C.

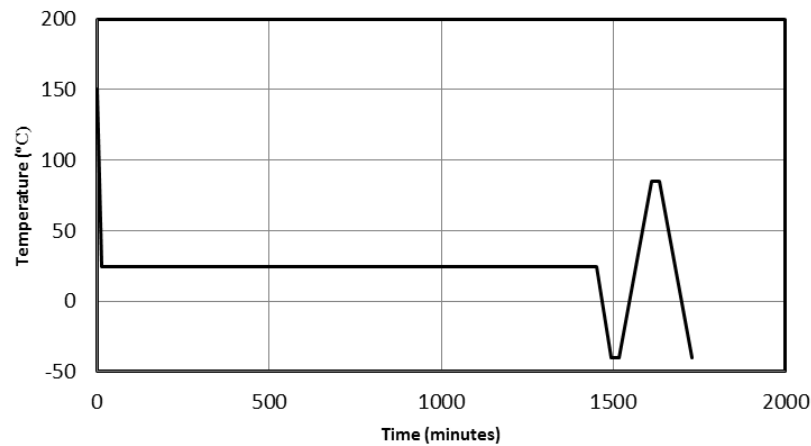


Fig. 5.1: Simulated temperature profile

## 5.2 GLASS

It is seen at the end of temperature profile, at -40°C, the first principle stress on the edges of the glass cover is tensile and is about 14.2 MPa. In the center of the glass where cells are present, the stresses are compressive with the first principle stress almost negligible. The same results for glass have also been reported in [37]. The presence of encapsulant and back-sheet only at the module edges has a higher CTE as compared to

that of glass. Thus, they compress more ultimately producing tensile stress on the edges of glass. The third principle stress is around -11.6 MPa over the region where cells are present. The compressive stress over the center of glass is regarded as the same reason that cells have a lesser CTE and therefore restricts glass to undergo compression. The stress contours are given in Fig. 5.2.

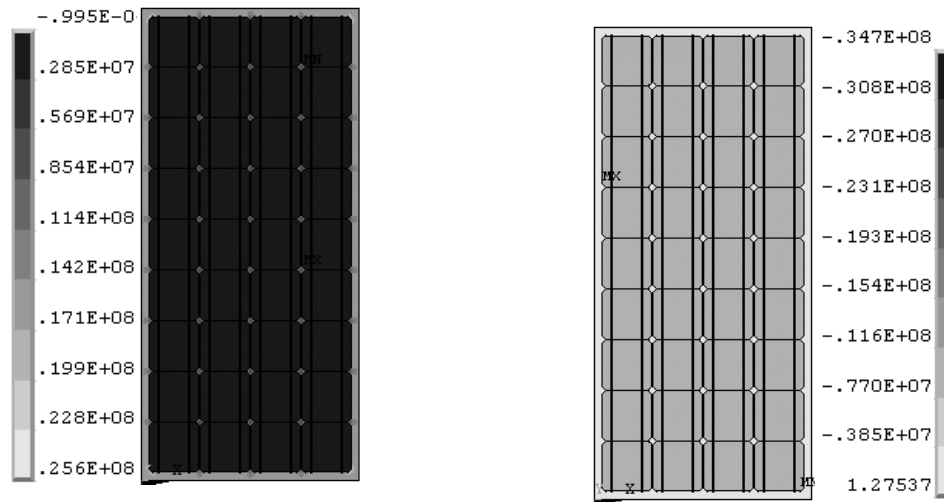


Fig. 5.2: First principle stress on glass (left) and third principle stress on glass (right)  
at -40°C

### 5.3 BACKSHEET

The back-sheet is under high tension. The values of the first principle stress range from 39.2 MPa to 40.9 MPa with the highest stress being generated over the region where cells are present. The results for backsheet also correspond to those which have been mentioned in [37]. The high tensile stresses are the result of low CTE of glass as compared to that of back-sheet. Glass exhibits a dominant character in the contraction of

the module as it has the largest thickness. All other components are forced to follow the thermal contraction of glass. The stresses generated decrease at the edges of the module as encapsulant is only present between the glass and back-sheet. Here, the contraction is accommodated by the encapsulant because of its low stiffness. Cells do not contract much and thus produce tensile stresses within the back-sheet. Contour of the first principle stress is shown in Fig. 5.3.

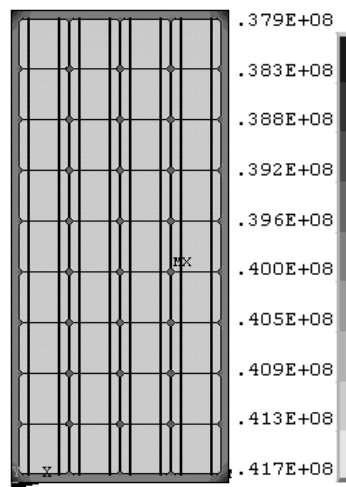


Fig. 5.3: First principle stress within the back-sheet at -40°C

## 5.4 CELLS

It can be seen in Fig. 5.4 that cells are under high compressive stresses. The maximum third principle stress is along the interconnectors and reach -217 MPa. The major region of cells has almost a uniform stress of -170 MPa. It is clear that silicon does not undergo plastic deformation as its yield stress is around 7 GPa as mentioned in [60]. The areas where the copper interconnects are present constitute the least thickness of

encapsulant within the laminate. Furthermore, copper is the stiffest component in the module and thus high stresses originate. Results of lamination process in [53] also show that high stresses in cells originate along the interconnect region. Shear stress values are dominant in the nodal plane as compared to the planes along the thickness of the laminate. The compressive stress in the two directions of the nodal plane is almost equal for the areas covering the major region of cells. The stresses near the region of the connections between interconnects are lower due to the presence of a thick compliant layer of encapsulant there.

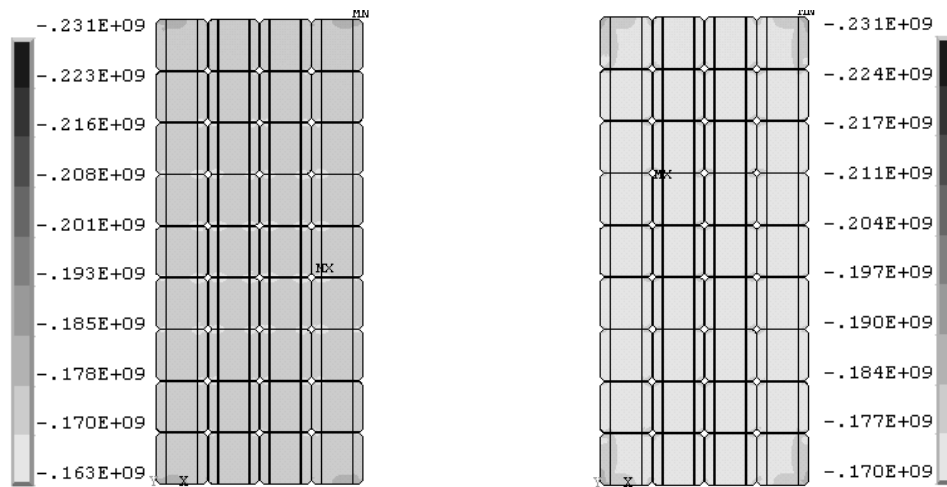


Fig. 5.4:  $x$ -direction stress in cells (left) and third principle stress in cells (right) at -  
40°C

## 5.5 INTERCONNECTS

First principle stress is almost 121 MPa along the interconnect strip. They undergo plastic deformation just after the curing process and this phenomenon has also been discussed in [53]. From the experiments performed in [11], it found that the yield stress



of copper is around 94 MPa at room temperature. The von-Mises stress, after the lamination process, reaches up to 96 MPa. As the temperature cycle is run, the interconnects yield further. Thus, it hardens producing high stress in cells along the region they are present. The nature of stress in copper is tensile as glass restricts its contraction. The contours are provided in Fig. 5.5.



Fig. 5.5: First principle stress on the interconnects (left) and on the connection between the interconnects (right)

## 5.6 PARAMETRIC STUDY

The encapsulant thickness was varied from 1.0 mm to 1.6 mm to see the effect on cells and interconnects. The variation in thickness did not show any difference on the stress value of interconnects. Although there were minor variations in the maximum third principle stress in cells (plotted in Fig. 5.6). It can be seen that the stress is least in the case of 1.2 mm thick encapsulant, whereas it is higher as the thickness is increased or decreased. It can be said that, at a lower thickness, copper follows the contraction of glass

due to its dominance and less encapsulant material. On the other hand, stresses are increased on increasing the encapsulant thickness; copper and silicon can gain room for their contraction thereby increasing the stress within the cell as both of them are directly tied to one another.

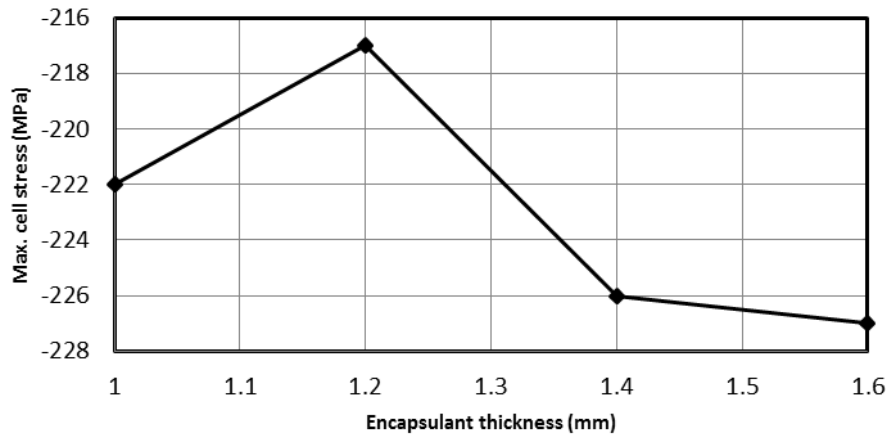


Fig. 5.6: Parametric study showing max. third principle stress on cells by varying encapsulant thickness

## 5.7 Conclusion

Simulation of the ASTM temperature cycle has been performed. The model of the PV module developed in the previous chapters is used and results are viewed at the worst condition to give the following conclusions:

- Glass exhibits a dominant character towards the contraction of the module. It forces all components to follow its pure thermal contraction.

- Stress in cells is higher along the interconnect region as:
  - They are both directly tied to one another.
  - Interconnect hardens as it undergoes plasticity.
- Parametric study shows that 1.2 mm is the optimum encapsulation thickness
- Interconnects undergo plasticity just after curing of the laminate. This hardens it thereby increasing the risk of breakage owing to fatigue induced during thermo cycles of day and night.

## **CHAPTER 6**

### **LIFE PREDICTION OF PV MODULES**

The main focus of this chapter is to develop a numerical life predicting model for PV modules capable of incorporating environmental conditions during its operation. For this purpose, firstly, a structural FE model was developed (Chapter 3) in which EVA encapsulant and silicon cells were modeled as viscoelastic and orthotropic respectively. The lamination procedure was simulated (Chapter 5) and it was found that the copper interconnects showed plastic deformation during cooling after curing of the encapsulant. This led to low-cycle fatigue as the cause of interconnect breakage. A thermal model was numerically developed and sequentially coupled to the structural model to include the effect of operating environment over PV modules. Finally, average life of a PV module (operating under the environment of Jeddah, Saudi Arabia) was estimated by using the thermal-structural coupled model. The results of the simulation were used within the Basquin-Coffin-Manson model to predict PV module life. The whole modeling procedure has been summarized by Fig. 6.1.

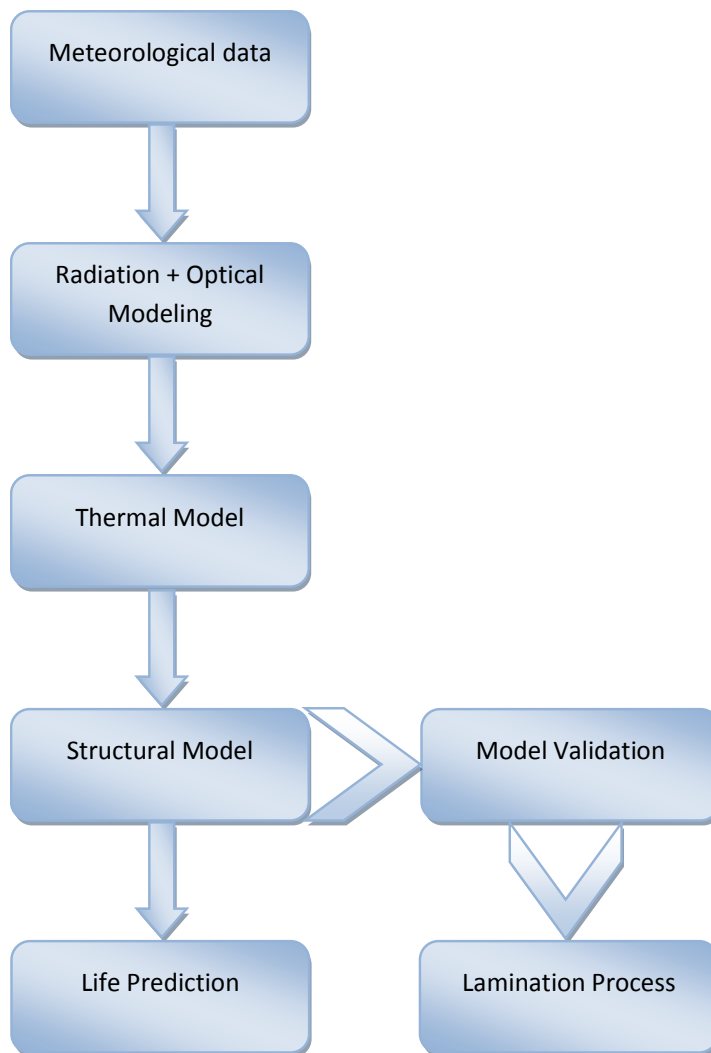


Fig. 6.1: Flow chart of the modeling process

## 6.1 FATIGUE

Fatigue is one of the failure mechanisms and is defined as a progressive and localized structural change that happens in a material which is subjected to cyclic loading. Such

loading may induce crack initiation which becomes unstable over time to propagate to complete failure. Fatigue may be distributed into four stages [61,62]:

- (I) Initiation of micro-cracks due to cyclic stress. The micro-cracks are of the order of 0.1  $\mu\text{m}$  to 1  $\mu\text{m}$ .
- (II) With time or by the increment of load, such cracks may propagate to larger length. These may range from 0.5 mm to 1 mm in length.
- (III) The cracks propagate rapidly due to instable crack growth.
- (IV) This stage relates to final instability leading to complete failure.

It has been already been discussed earlier about the cycling loading in photovoltaic modules due to successive temperature changes attributed to day and night. In the present study, time to crack initiation (stage-I) has been predicted. Or in other words, effort has been done to foresee the life of a standard photovoltaic module and to explain the 25 year guarantee given by most of the PV module manufacturers.

Fatigue may be divided into two categories, high-cycle and low-cycle fatigue [12]. When loads are of such magnitude that more than about 10,000 cycles are required to produce failure, then fatigue causing failure is termed as high-cycle fatigue. In such a case, deformation is principally elastic. On the contrary, when deformation is generally plastic due to cyclic loading or when failure occurs in less than 10,000 cycles, in such a case, fatigue is termed as low-cycle fatigue. It was seen that copper interconnects undergo plastic deformation just after lamination, so this makes it a case of low-cycle fatigue.

## 6.2 Basquin-Coffin-Manson Model

Certain life predicting models have been developed in order to evaluate life (time to failure) which make use of strain in the case of low-cycle fatigue. Strain based models are widely used at present. Among them, the Basquin-Coffin-Manson relationship is a renowned model to find out fatigue life as given by Eq. (6.1).

$$\frac{\Delta \varepsilon}{2} = \frac{\sigma'_f}{E} (2N_f)^b + \varepsilon'_f (2N_f)^c \quad (6.1)$$

where,

$$\frac{\Delta \varepsilon}{2} = \text{total strain amplitude}$$

$$\sigma'_f = \text{fatigue strength coefficient}$$

$$\varepsilon'_f = \text{fatigue ductility coefficient}$$

$$E = \text{Young's modulus of elasticity}$$

$$N_f = \text{no. of cycles to crack initiation}$$

$$b = \text{fatigue strength exponent}$$

$$c = \text{fatigue ductility exponent}$$

The total strain amplitude ( $\frac{\Delta\epsilon}{2}$ ) is actually the half of the strain range within the loading cycle. For evaluating parameters required for life estimation, there are certain testing methods discussed in [63]. Eq. (6.1) is actually the combination of the Coffin-Manson relationship and Basquin's equation given by Eq. (6.2) and Eq. (6.3) respectively. Coffin-Manson model was proposed independently by Coffin and Manson in 1954. This model accounts for such low-cycle fatigue conditions where plasticity is involved. To deal with intermediate fatigue problems having the effect of both elastic and plastic deformation, Eq. (6.3) was added onto Eq. (6.2) by dividing by the Young's modulus of elasticity ( $E$ ) to give Eq. (6.1).

$$\frac{\Delta\epsilon_p}{2} = \epsilon'_f (2N_f)^c \quad (6.2)$$

$$\frac{\Delta\sigma}{2} = \sigma'_f (2N_f)^b \quad (6.3)$$

where,

$$\frac{\Delta\epsilon_p}{2} = \text{plastic strain amplitude}$$

$$\frac{\Delta\sigma}{2} = \text{stress amplitude}$$

For the case of PV module, fatigue properties of copper were taken from [64], given in Table 6.1.



Table 6.1: Fatigue parameters for copper [64]

Fatigue parameter	Value
$\sigma'_f$	345.08 MPa
$\varepsilon'_f$	0.3
$b$	-0.05
$c$	-0.6

### 6.3 REPRESENTATIVE DAYS

In this work, life of PV module was predicted operating under the atmospheric conditions of Jeddah, Saudi Arabia. From the irradiance and ambient temperature data for one year, four representative days were chosen to represent varying condition of irradiance throughout the year. The chosen days are displayed in Fig. 6.2 and had the following characteristics:

- Day 1: First day was chosen out of January which represented low irradiance and low temperature.
- Day 2: The second one was out of July representing hot weather and smooth irradiance/not cloudy.
- Day 3: The third one was from October and was partially clouded with average ambient temperatures.
- Day 4: The fourth one was chosen from December and had an extremely overcast sky with low temperatures.

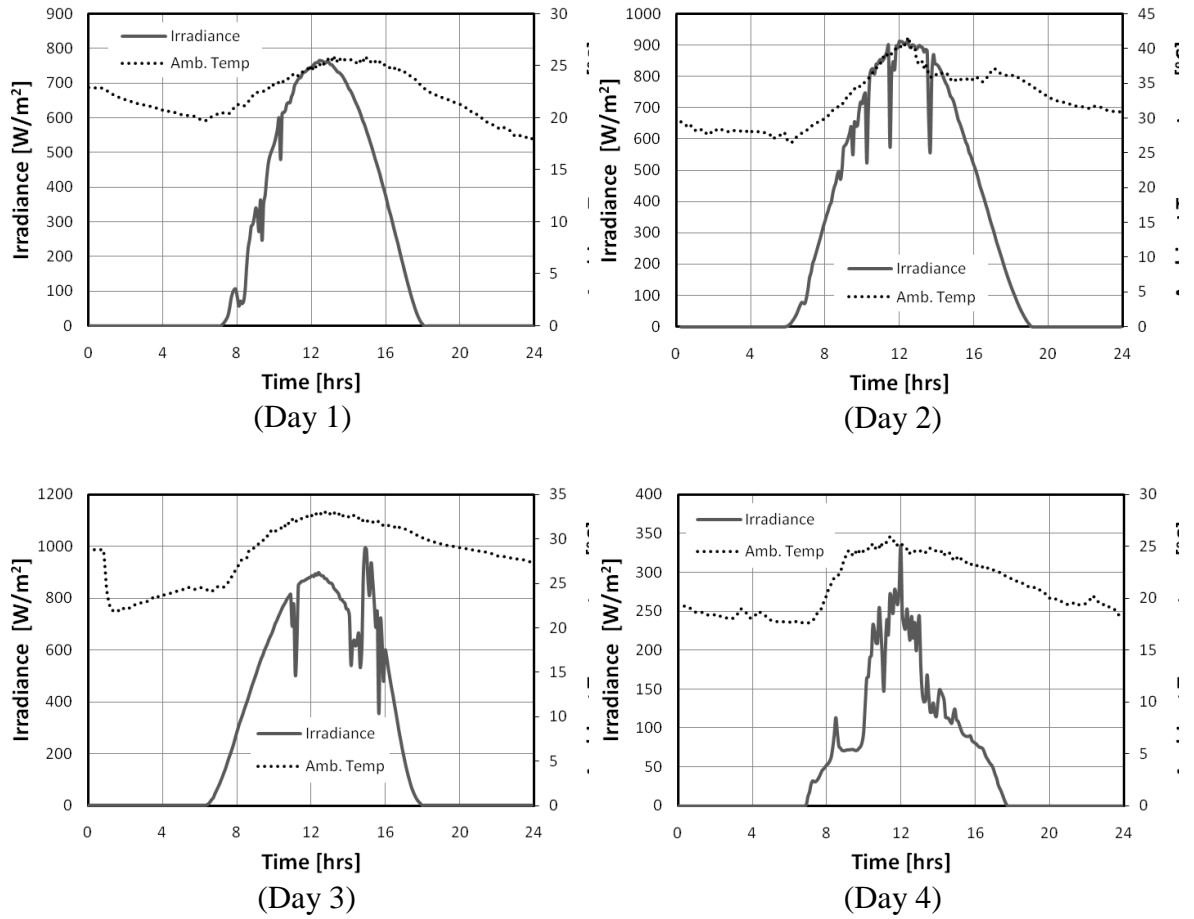


Fig. 6.2: The irradiation and ambient temperatures of the four representative days.

The maximum and minimum principle strains for each of the four days were evaluated from FE simulation. Then time (no. of loading cycles) to crack initiation of copper interconnects was calculated through Eq. (6.1) under the assumption that the PV module continues to function under the same load cycle of the day.

## 6.4 LOADS AND BOUNDARY CONDITIONS

Pre-stress due to the lamination process is studied followed by the actual temperature cycle of the representative days by coupling the thermal model to the structural model.

### 6.4.1 Lamination Process

For the first part of analysis (lamination process), the module is constraint at one corner to allow free deformation. This will help to study the pure dependence of materials on one another. The stress-free temperature is taken as the lamination temperature (150°C) because at that temperature every component of the module is independent to one another thus allowing a stress-free expansion. The room temperature is assumed to be 21°C which is the final temperature after cooling. Steady state simulation is performed.

### 6.4.2 Sequentially Coupled Analysis

#### (a) Thermal Model

For the second part of analysis, thermal boundary conditions and loads are applied first. The Hay-Davies-Reindl-Klutcher (HDKR) model is used to evaluate the absorbed radiation from irradiance and optical parameters. Eq. (3.27) represents the HDKR model [39,65], where  $S$  is the absorbed solar radiation,  $G$  is the horizontal plane solar radiation,  $R_{beam}$  is the ratio of beam radiation on tilted plane to that on horizontal plane,  $\rho$  is the ground reflectivity,  $\beta$  is the tilt angle of PV module,  $A_i$  is the anisotropy index and  $M$  is the air mass modifier. The subscripts  $b$ ,  $d$ ,  $g$  and  $ref$  are for the beam, diffuse, ground reflected and reference solar radiations respectively.

Some of the absorbed radiation is converted into electricity while the rest is converted into heat. The internal heat generation ( $Q$ ) can then be given by Eq. (6.4) [39].

$$Q = \frac{(1 - \eta_{pv}) \times S \times A_{panel}}{V_{pvcell}} \quad (6.4)$$

where  $\eta_{pv}$  is the electrical efficiency of the cells,  $A_{panel}$  is the front area of PV module and  $V_{pvcell}$  is the volume of cells.

Constant convection was applied with heat loss coefficients of  $15.4 \text{ W/m}^2 \text{ K}$  and  $2.8 \text{ W/m}^2 \text{ K}$  to the top and bottom surfaces of the module. The boundary condition applied to heat transfer equations of the top and bottom layers of the module is given by Eq. (6.5).

$$-\mathbf{n} \cdot \mathbf{q} = h(T_{amb} - T_s) \quad (6.5)$$

where  $\mathbf{n}$  is the surface normal,  $T_{amb}$  is the ambient temperature and  $T_s$  is the surface temperature.

#### (b) Structural Model

For the structural part, all the four edges of the module were fixed in all directions to approximate the presence of mounted frame. The reference temperature was the same as in lamination process. Thermal loads were applied from the solution of the coupled thermal model at each hour and steady state solution was performed.

## 6.5 RESULTS AND DISCUSSION

### 6.5.1 Lamination Process

Lamination in PV modules is basically done to cure the encapsulant so that it holds the whole structure to form a single unit. Encapsulant (EVA) sheets are placed in between each layer and then are kept at 150°C under vacuum for about 12 minutes. Then the structure is cooled down to room temperature. Cooling causes the encapsulant to solidify and adhere to all components. This process was simulated and the module was cooled to 21°C. At this temperature, von- Mises stress is almost constant over the whole interconnector strip and is about 95.8 MPa. They undergo plastic deformation just after the curing process and this phenomenon has also been discussed in [53]. From the experiments performed in [11], it is found that the yield stress of copper is around 94 MPa at room temperature. The contours for von-Mises stress and von-Mises plastic strain in the interconnects between two adjacent cells are given in Fig. 6.3 at 21°C. It should be noted that the shaded 3D view of the region for which the contours are displayed are merely for understanding. The results displayed are from the solution of the 2D shell model.

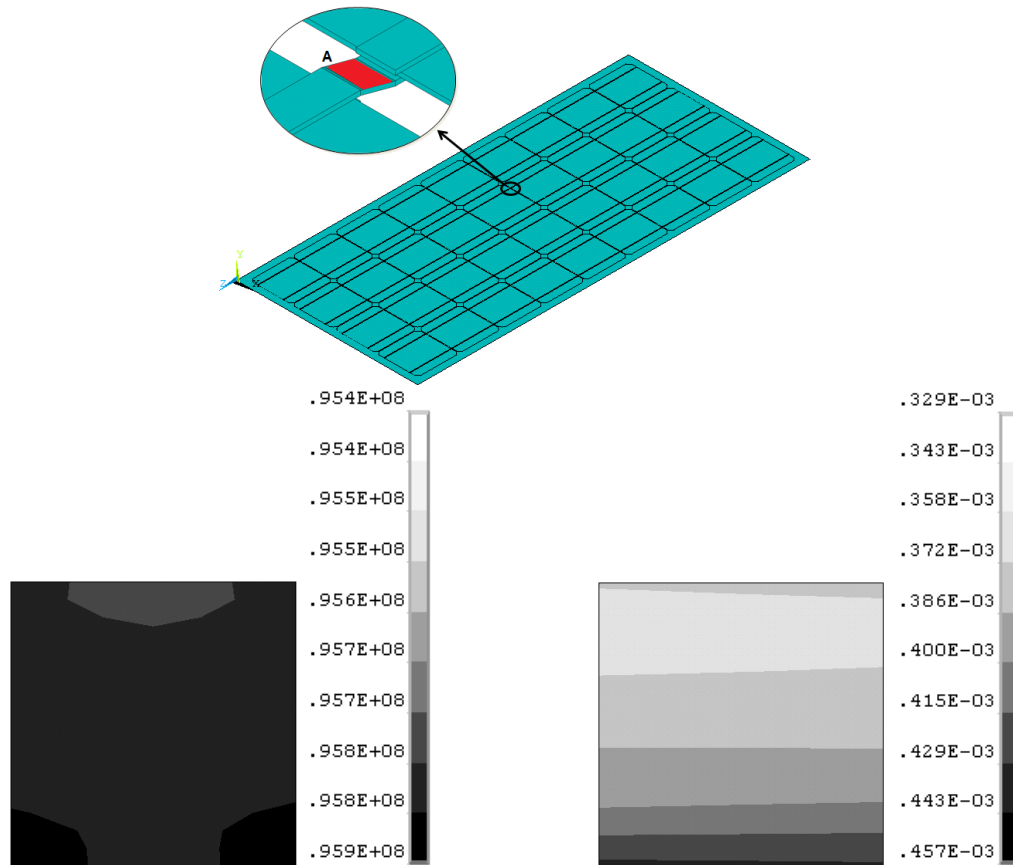


Fig. 6.3: Von-Mises stress (left) and von-Mises plastic strain (right) at room temperature after the lamination process at the interconnect region between two adjacent cells (shaded region A)

### 6.5.2 Stress Analysis on the Basis of Worst Day Conditions

Out of the four days simulated, it was found that stresses are the highest during December. This month constitutes the lowest temperatures of the whole year which makes the PV module to operate farthest from its stress-free state. As the edges of the module are fixed to simulate the presence of frame, the nature of stresses is tensile over the whole laminate and hence, the third principal stress is almost zero.

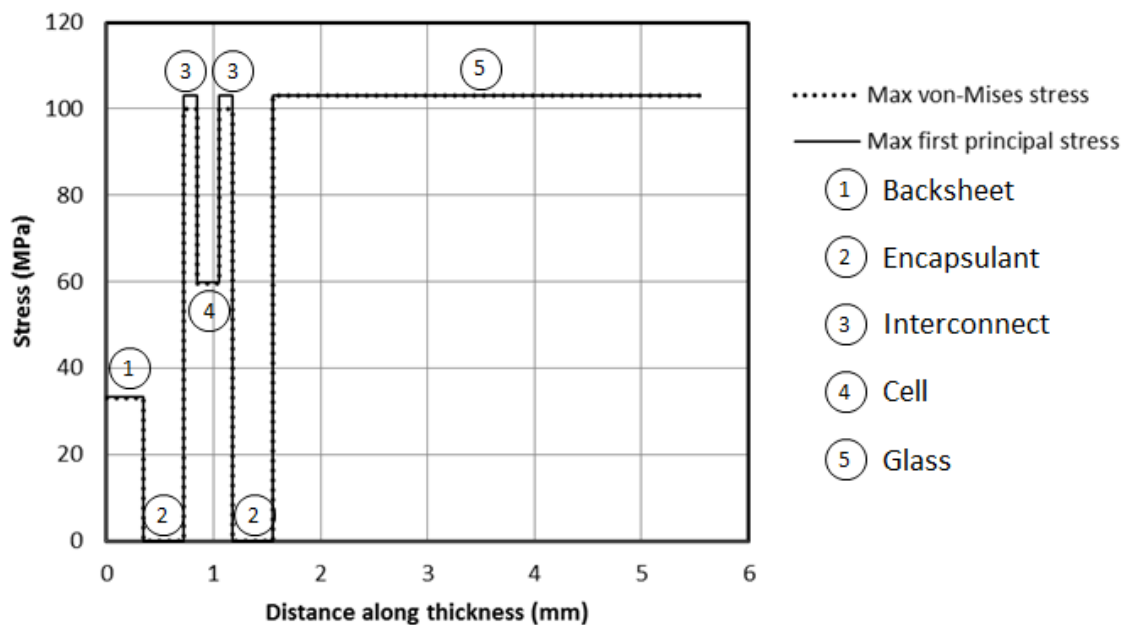


Fig. 6.4: Max von-Mises stress and first principal stress through the thickness of the module at lowest temperature on Day 4

Fig. 6.4 shows the variation of maximum von-Mises stress through the thickness of the laminate on worst day condition at lowest temperature. It is seen that the maximum

von-Mises stress and the maximum first principal stress are almost equal. Thus, the overall nature of stresses on all the components is almost tensile. Glass and the interconnects have almost the same first principal stress (around 103 MPa) which is highest amongst all. The high stress in glass is due to fixed boundary condition applied at its edges. Whereas, cells have a lower stress of 60 MPa as they are not directly constrained and their thermo-mechanical movements are aided due to the compliancy of the encapsulant material. The interconnects, on the other hand, are directly bound to cells causing them to yield. Lowest stress can be seen in the encapsulant as it is the least stiff material as compared to others.

Fig. 6.5 shows the first principal stress contours of all the components of the laminate. Highest stress on glass can be seen along the interconnect region and is around 112 MPa. It is because of less encapsulant material present at that portion along the thickness. The segment of glass which is void of cells and interconnects beneath it has the least stress of 99.4 MPa. The majority area of glass is over the cells and has an intermediate value of stress around 106 MPa. There is no significant variation of stresses in backsheet but is less than that of glass (about 33 MPa) for having a lesser value Young's modulus of elasticity. The corners of the cells have the highest stress value of 63.4 MPa. This can be attributed to the lack of material present at the location due to rounded corners. Stress of 60 MPa covers the major region of cells and is higher than the portions representing the interconnect areas. The reason being that the effect of high CTE of glass as compared to cells is barred by the presence of the interconnect material in between. In the case of interconnects (shown in Fig. 6.6), it can be seen that the



maximum stress is on the corner of the connection between adjacent cells and is 106 MPa as the contraction of copper is restricted by silicon. Rest of the portion of interconnects has almost a constant stress of 103 MPa.

Fig. 6.7 gives the stress variation along the longitudinal and transverse paths neglecting the backsheet and glass cover. Path *AB* displays maximum stresses is over the interconnect regions between adjoining cells. Lowest stress suggests the area of the encapsulant material which is the same in path *CD*. It is also seen that the stresses in cells are 13 MPa higher in the transverse direction than in the longitudinal direction, with almost the same principal and von-Mises stress. Stresses in interconnects in both direction are the same around 103 MPa.

Fig. 6.8 shows the relation of von-Mises and first principle stress with time and temperature for Day 3. Location A represents maximum stress at minimum temperature whereas location B represents minimum stress at maximum temperature. Thus, by seeing the figure, it can be said that the temperature change of 10°C causes a stress change of 85 MPa.

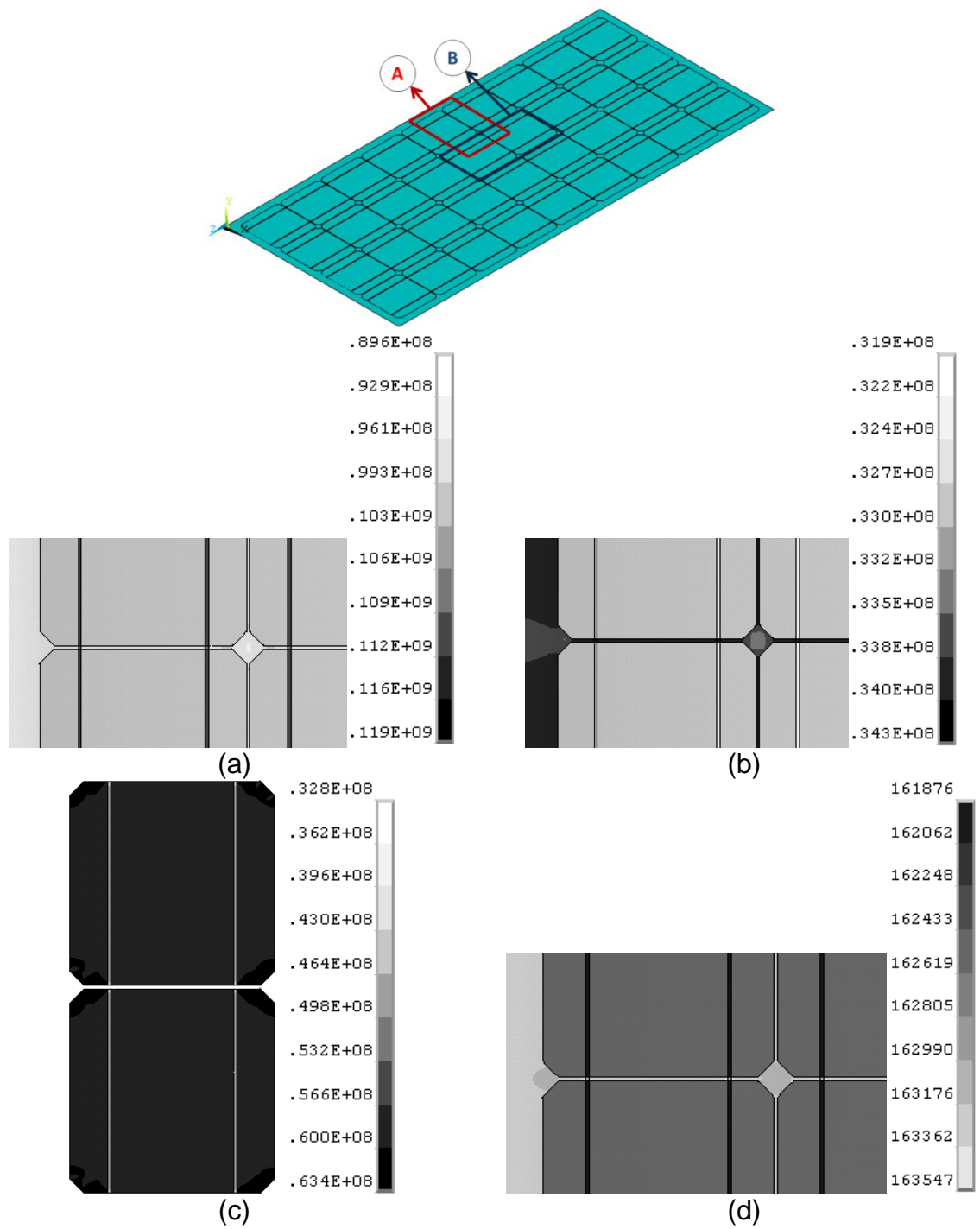


Fig. 6.5: First principal stress contours of (a): Glass at region A, (b): Backsheet at region A, (c): Cells at region B and (d): Encapsulant at region A at lowest temperature on

Day 4

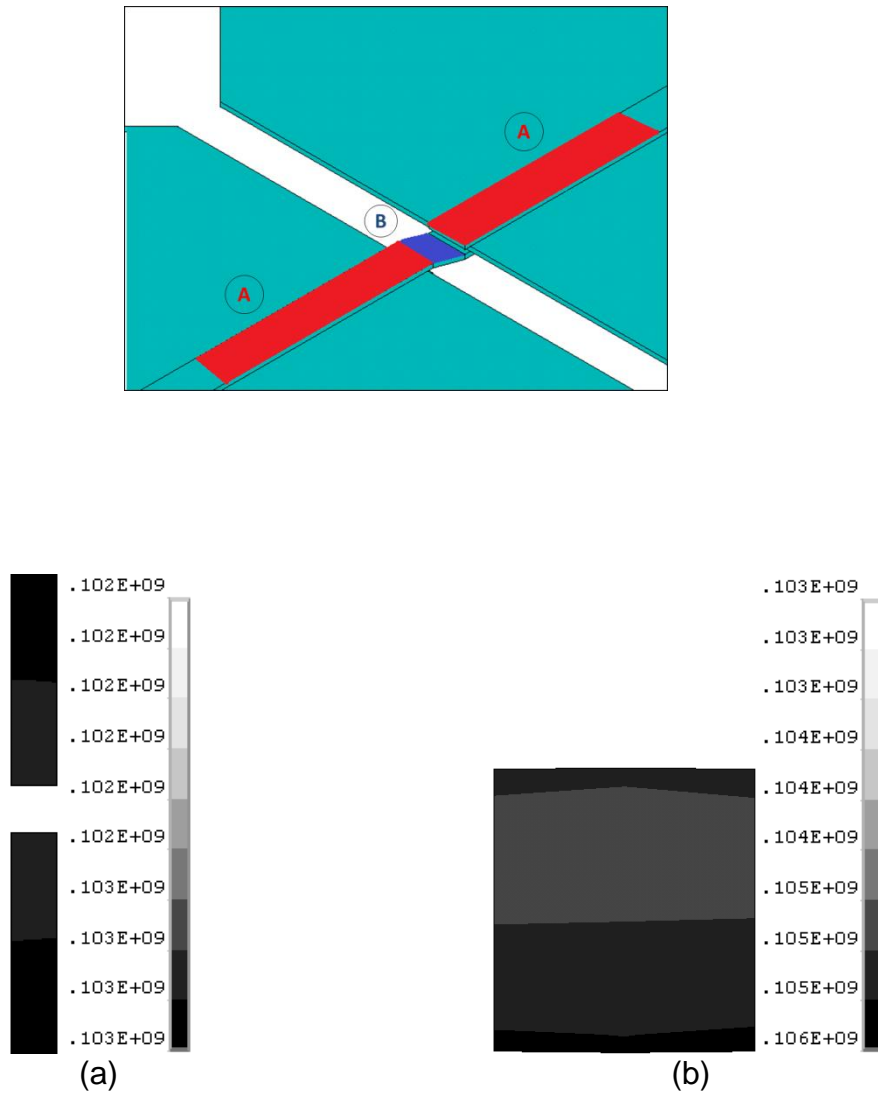
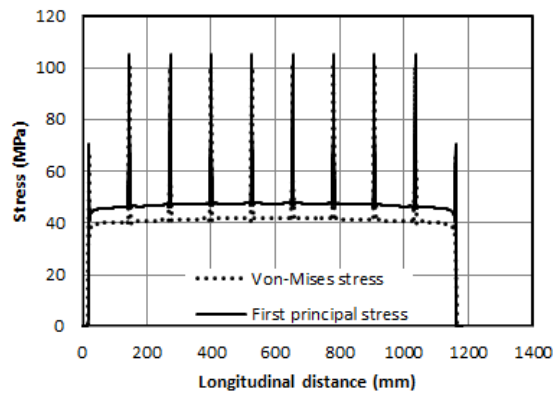
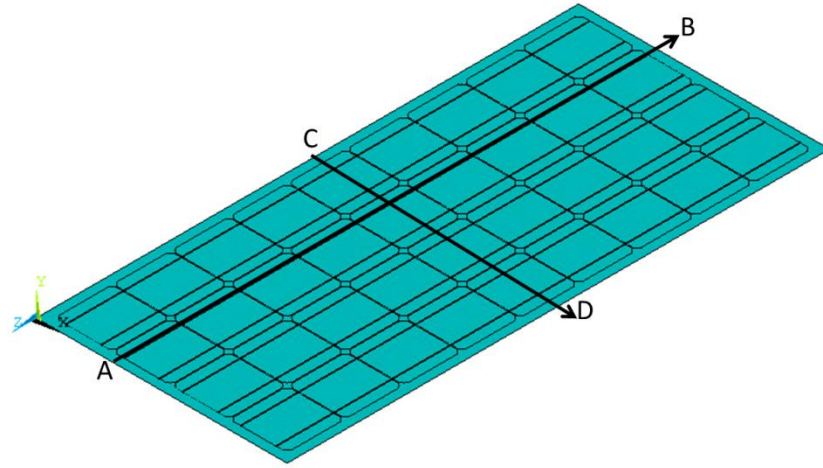
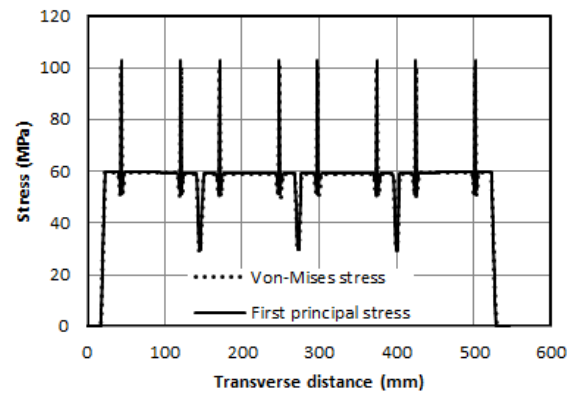


Fig. 6.6: First principal stress contours of (a): Interconnect region over the cells at region A and (b): Interconnect region between two adjacent cells at region B at lowest temperature on Day 4



(a)



(b)

Fig. 6.7: Stress variation on module along (A): Longitudinal path  $AB$  and (B):

Transverse path  $CD$  at lowest temperature on Day 4

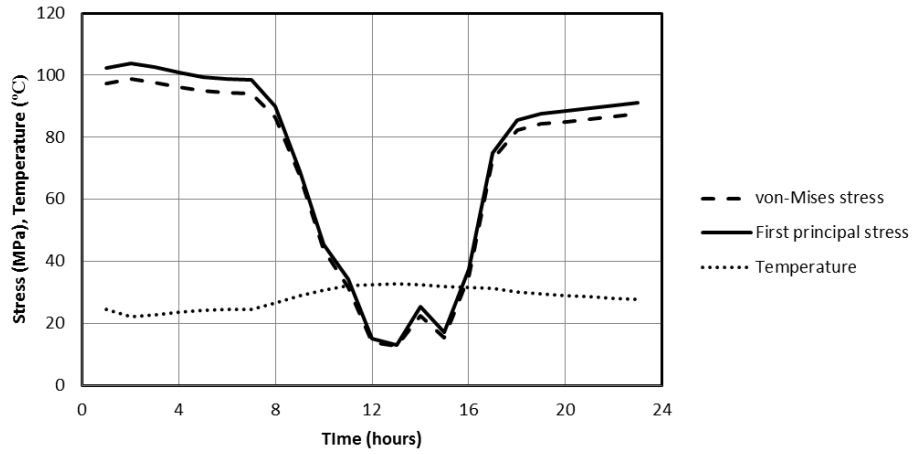


Fig. 6.8: Transient change in von-Mises stress and first principal stress for Day 3 on copper interconnect. A represents the time of max. stress, min. temperature and B represents the time of min. stress, max. temperature

Table 6.2 shows the difference of the stress amplitudes for all the four days at a point over the interconnect region. As already mentioned that the maximum stress is during Day 4 but later in Table 6.3, it will be seen that the conditions of Day 4 also give maximum life for PV module. The reason is attributed to its least stress amplitude which can be seen according to Eq. (6.3). Thus, life or time to crack initiation is majorly affected by stress amplitude rather than its magnitude.

Table 6.2: Maximum and minimum principal stress and stress amplitude at a point on the interconnect for all four representative days

<b>Day <i>i</i></b>	<b>Max Principal Stress <math>\sigma_{\max}</math> (MPa)</b>	<b>Min Principal Stress <math>\sigma_{\min}</math> (MPa)</b>	<b>Stress Amplitude <math>\Delta\sigma/2</math> (MPa)</b>
1	104	27	38.5
2	99.8	16.6	41.6
3	102	6.8	47.6
4	105	59.2	22.9

## 6.6 LIFE PREDICTION

The maximum and minimum total strain is given in Table 6.3. From them, their respective strain amplitudes (half of the difference between maximum and minimum strain) is evaluated. By using the material properties of copper in Table 6.1 and Eq. (6.1), the number of cycles is calculated for the four days and the number of years to crack initiation was determined.

Now, to get the average life of a PV module operating in Jeddah, weights were assigned to each representative day according to their impact over the whole year. It was seen that at the maximum and minimum temperature of the day, the total strain was vice-versa of temperature. It is because the zero strain temperature of the simulation was set to 150°C as mentioned in the previous sections. Thus, it can be said that the temperature

range of a day has a direct impact on the life of a PV module. To assign weights, total of average ambient temperatures ( $T_{total}$ ) was calculated according to Eq. (6.6).

$$T_{total} = \sum_{i=1}^n \frac{T_{\max\_i} + T_{\min\_i}}{2} \quad (6.6)$$

where  $T_{\max\_i}$  is the maximum ambient temperature of day  $i$ ,  $T_{\min\_i}$  is the minimum ambient temperature of day  $i$  and  $n$  is the total no. of days in a year. Weight for each representative day is simply estimated by dividing the average temperature of the day by  $T_{total}$ . The average life ( $L_{avg}$ ) can then be given by Eq. (6.7) and is calculated to be 26.63 years.

$$L_{avg} = \sum_{i=1}^4 \frac{W_{day\_i} \times L_{day\_i}}{W_{total}} \quad (6.7)$$

where  $W_{day\_i}$  is the weight of the  $i^{th}$  representative day,  $L_{day\_i}$  is the life of the  $i^{th}$  representative day and  $W_{total}$  is the sum of the weights of four representative days. The weights for the four days are given in Table 6.3.

Table 6.3: Maximum and minimum total strain, no. of cycles to crack initiation, life and respective weights for the four representative days

<b>Day</b> <i>i</i>	<b>Max Total Strain</b> $\epsilon_{\max}$	<b>Min Total Strain</b> $\epsilon_{\min}$	<b>Cycles</b> $N_f$	<b>Weight</b> $W_{day\_i}$	<b>Life</b> $L_{day\_i}$ (years)
1	0.025931	0.019471	10563	10563	28.94
2	0.024076	0.017513	9685	9685	26.53
3	0.024237	0.017319	7366	7366	20.18
4	0.026252	0.01993	11937	11937	32.71

## 6.7 CONCLUSIONS

Within the presented work, FE analysis was used to determine the behavior of the components of PV module under operating conditions. A comprehensive structural model was formed and which was coupled to a thermal model. Starting from the lamination procedure and then by using meteorological data, the following conclusions have been drawn out of this work.

- The high failure percentage of copper interconnects, given in the literature, is justified through its yielding during cooling after the lamination process. The plastic deformation makes it a subject of low-cycle fatigue.
- Glass exhibits a dominating behavior due to its large thickness. As the contraction of glass was fixed, all the components of PV module had induced



tensile stress within them. Glass also bears the maximum stress amongst all of the components as it bears most of the area of constraint, which was made to provide frame effect.

- Silicon cells operate within a safe temperature range as the stresses induced in it are way below their yield stress; unless they are pre-cracked due to their manufacturing and soldering procedure.
- Average life of a PV module has been estimated through its dominant mode of failure which is the breakage of copper interconnects and is found out to be 26.63 years. It is quite close to the 25-year warranty given by most of the PV module manufacturers.
- The numerical procedure developed is inclusive of operating conditions and has the ability to predict proper behavior and life under operation with reasonable accuracy. It can be used as a tool to anticipate the effects of design changes in the form of stress distribution and life.

## **CHAPTER 7**

### **COMPARATIVE STUDY OF PV MODULE ENCAPSULANTS**

The Photovoltaic (PV) industry has shown rapid growth in the last few decades. Its expansion has led to the selection of such materials in its construction, which enable it to meet its requirements efficiently. Now-a-days, research on PV modules is mainly focused on the encapsulant material, due to significant involvement of its properties over PV module performance. The structural performance of PV modules is enhanced due to the protective covering it provides to isolate silicon cells from the influence of the environment. At the same time, it also has to be transparent to light so as not to hinder the electrical performance of PV modules. In fact, it could also provide a medium to extract heat from the cells to increase their efficiency. So, the fulfillment of these requirements and others (discussed later) are important for an optimal performance of PV modules.

During the 1960s and 1970s, Polydimethylsiloxane (PDMS)/Silicone was used as an encapsulant for PV modules [66]. But from the 1980s till today, the PV industry is dominated with Ethylene Vinyl Acetate (EVA) [67]. EVA was chosen over PDMS

mainly due to its low cost. In the late 1990s, it was found that EVA turned yellow/brown due to UV radiation from the sun thus decreased its transmittance. It has also been reported to lose adhesion under UV light [23]. Furthermore, EVA has the ability to concentrate water due to diffusion which makes it to react with moisture to form acetic acid. The acetic acid speeds up the corrosion process of the inner components of the PV module [68]. This raises a question of its operation under humid climates. The glass transition temperature ( $T_g$ ) of EVA is  $-15^{\circ}\text{C}$  [27] and it comes in between the operating range of a PV module in cold regions. Thus, compliancy of EVA is an issue for modules operating in such regions. The mentioned concerns have recently revitalized the interest to study different polymers for PV module encapsulation. Such polymers include Polyvinyl butyral (PVB), Ionomer, PDMS and Thermoplastic polyurethane (TPU). The mentioned encapsulants have their merits and demerits over one another, but the best compromise amongst them needs to be chosen with respect to PV module performance and life.

In the current work, modeling and life prediction technique developed in the previous chapters is used including different encapsulants. Viscoelastic modeling of was utilized for EVA, PDMS, PVB and TPU to determine the proper effect of time-temperature dependence over stress distribution with in the components of the PV module. Ionomer was modeled as a hyperelastic. The electrical performance and life of the module is compared for all encapsulants. At the end, the remaining properties and cost for each encapsulant is seen to ultimately pick out an optimum one.

## **7.1 SIGNIFICANCE AND DESIRABLE PROPERTIES**

The main function of an encapsulant material is to protect the components of a PV module from foreign impurities and moisture along with the fortification from mechanical damage. An encapsulant also acts as an electrical insulator between cells and interconnects to prevent leakage current and binds all of the components together. Along with these, there are certain desirable properties of an encapsulant such as easy processability, good transmittance of light, high thermal conduction, long operating temperature range, UV radiation resistant, compliant, low cost and long life.

These characteristics of an encapsulant have a direct impact on the performance and life of PV modules. A more compliant encapsulant will have a less modulus of elasticity. In [69], FE simulations were carried out to find out stresses in solder bond by changing the modulus of elasticity of EVA. It was found that the increase in modulus caused more stress transfer to solder bond. Same phenomenon is reported by [67] by performing accelerated aging tests on EVA and Silicones. The thermal conductivity of encapsulants is also an important factor. An increase in PV module electrical efficiency has been reported in [70] by increasing the thermal conductivity of EVA using thermally conductive fillers. More thermal conductivity of the encapsulant material will help to dissipate heat. It is also reported that PV modules face a 0.4% to 0.5% loss in efficiency per rise in temperature [71]. Thermally conductive polymers also help the cells to prevent mismatch losses which is usually generated due to temperature change (leading to change in power output) in cells. Cells may be burnt out due to heat generation, especially in the

case of CPV. Encapsulants with larger thermal conductivity may assist the removal of heat to heat sink. Similarly, good light transmittance of encapsulants helps in more power generation. Long-term exposure to UV light may cause the breakage of bonds of encapsulant material [66]. Encapsulants such as EVA don't remain transparent to light with such changes. Such chemical changes also cause the encapsulant to lose its adhesiveness and allows foreign impurities to enter and corrode the inner components of the module. Encapsulants with high water vapor transmission rate (WVTR) are the reason for corrosion of the solder bonds and copper interconnects. High water concentration ability in encapsulants such as EVA react with moisture to form acetic acid which speeds up the corrosion process [72].

## **7.2 MATERIAL MODELING OF ENCAPSULANTS**

A comprehensive theoretical background of viscoelastic modeling has been discussed in Chapter 2. Chapter 3 deals with the implementation of the viscoelastic model for EVA which is based on [37]. Dillard et al. [73] have developed a numerical model to estimate stresses in PDMS sealants due to thermally-driven deformations. To model viscoelasticity of PDMS, experiments were performed on a controlled-strain rheometer at different temperatures over a frequency range. The recorded data was used to construct a master-curve by using the William-Landel-Ferry (WLF) shift function (Eq. (3.17)), to incorporate the effect of Time-Temperature-Superposition (TTS) as shown in Fig. 7.1. This master curve was then fit by Prony series (Eq. (3.15)) of the Maxwell's model (Fig. 3.3); of which the shear moduli have been stated in [73]. In the current work,

instantaneous modulus ( $G_o$ ) was found out by Eq. (3.22). Next, the relative moduli ( $\phi_i$ ) were found out using Eq. (3.21) which were incorporated into the Finite-Element (FE) model.

PVB is mostly used in structural laminated glass or glass/PVB/glass configuration. Sanz-Ablanedo et al. [74] have performed the viscoelastic characterization of PVB, used in the stated application to incorporate the model into a numerical simulation. For this purpose, stress relaxation tests on PVB samples. The measured data was processed in a similar fashion to get the master-curve (Fig. 7.2). This master-curve was used to get  $\phi_i$ , which is provided to ANSYS in the current work.  $G_o$  was calculated for discrete data points of the curve in Fig. 7.2, using Eq. (3.23) and Eq. (3.24).

Multi-layered structures are bonded together through a binding polymer such as TPU. Thermo-mechanical displacements of such structures lead to CTE mismatch and ultimately stressing of the laminate. To model such process, MacAloney et al. [75] have characterized TPU to study its viscoelasticity. DMA was used to find the storage and loss moduli at different temperatures. The frequency domain of the results was shifted to time domain using inverse Fourier transform. By going through a similar procedure, as described above, a single master-curve (Fig. 7.3) was constructed whose Prony coefficients were evaluated. The mentioned equilibrium modulus ( $G_\infty$ ) in [75] is used to find  $G_o$  in the present work by Eq. (3.22).  $G_o$  is then utilized to get  $\phi_i$  as mentioned for PDMS.

Ionomer is used as a cover in golf balls. To simulate the collision of golf balls, Tanaka et al. [76] have modeled Ionomer as a hyperelastic material which is also used in the current work. Although, it may not model the temperature and rate dependency of the material but it is a good assumption of the non-hookean properties of the polymer. The hyperelastic material properties are defined through the Mooney-Rivlin model given by Eq. (7.1) [59],

$$W = c_{10}(\bar{I}_1 - 3) + c_{01}(\bar{I}_2 - 3) + \frac{1}{d}(J - 1)^2 \quad (7.1)$$

where,

$W$  = Strain energy potential

$\bar{I}_1$  = First deviatoric strain invariant

$\bar{I}_2$  = Second deviatoric strain invariant

$c_{10}, c_{01}$  = Material constants defining deviatoric deformation

$d$  = Material incompressibility parameter

The initial shear modulus ( $\mu_i$ ) and the initial bulk modulus ( $K_i$ ) are given by Eq. (7.2) and Eq. (7.3) respectively,

$$\mu_i = 2(c_{10} + c_{01}) \quad (7.2)$$

$$K = \frac{2}{d} \quad (7.3)$$

$d$  can be defined as,

$$d = \frac{1 - 2\nu}{c_{10} + c_{01}}$$

The rest of the parameters used in simulation for all the polymers are given in Table 7.1.

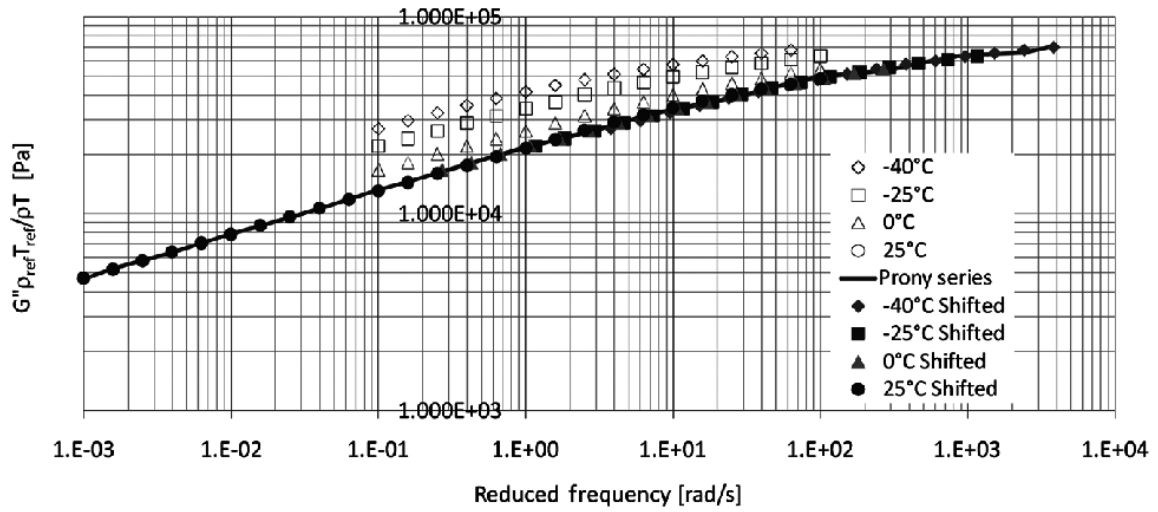


Fig. 7.1: Measured data and master-curve for the loss modulus of PDMS in reduced frequency domain [73]



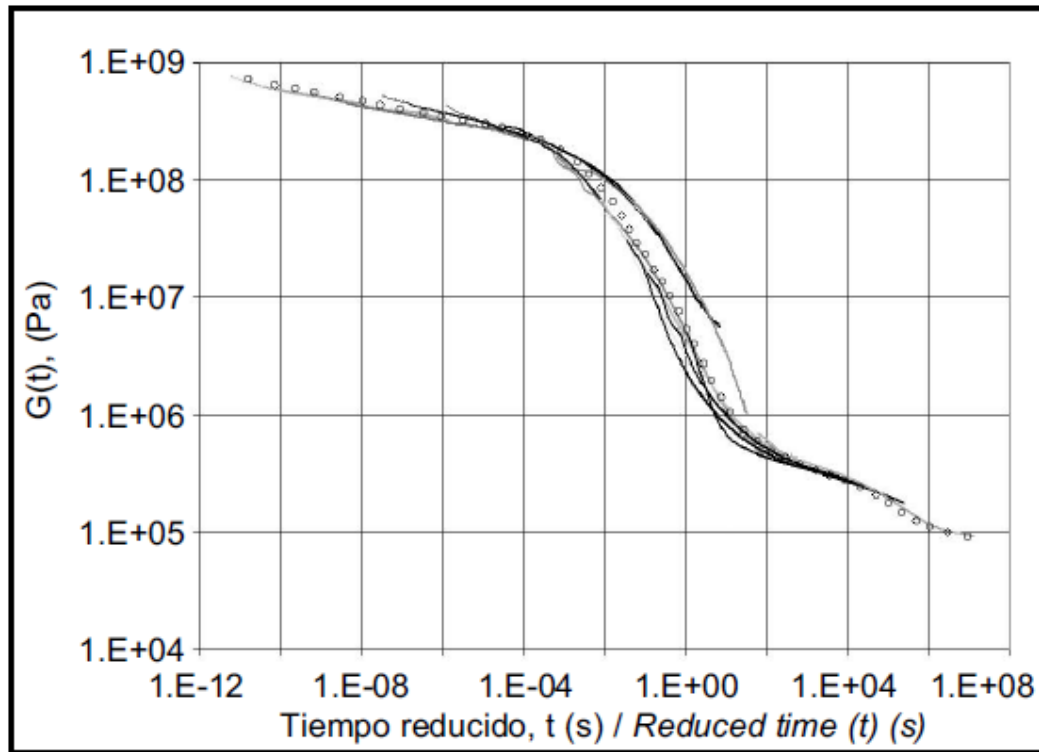


Fig. 7.2: Master-curve for PVB obtained by shifting the results of stress relaxation experiments at different temperatures in time domain [74]

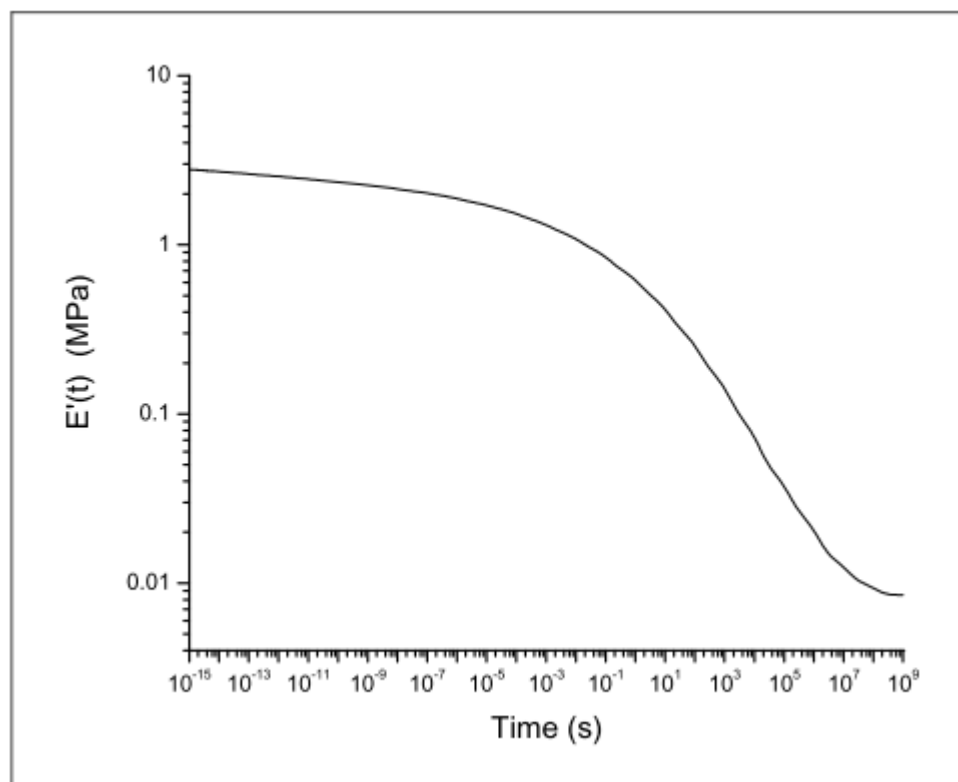


Fig. 7.3: Prony fit of the master-curve of TPU in time domain [75]

Table 7.1: Thermo-mechanical properties of polymers used in simulation

<div>Properties</div> <div>Polymers</div>	$E_o$ (MPa)	$C_1$	$C_2$ (K)	$T_r$ (K)	$E_i$ (MPa)	$c_{10}$ (MPa)	$c_{01}$ (MPa)	$\nu$	$\rho$ (kg/m <sup>3</sup> )	$\alpha$ x 10 <sup>-6</sup> (K <sup>-1</sup> )	$k$ (W/mK)	$C$ (J/kgK)
<b>EVA</b> [37,39,41]	1300	48.44	172.55	253	-	-	-	0.40	960	270	0.311	2090
<b>PDMS</b> [73,77]	2.832	2.33	158	298	-	-	-	0.49	965	200	0.15	1460
<b>PVB</b> [74,78]	2981	20.7	91.1	293	-	-	-	0.33	1030	412	0.2	1973
<b>TPU</b> [75,79,80]	3.126	23.1	69.3	212	-	-	-	0.45	1230	220	0.176	1550
<b>Ionomer</b> [76,81,82]	-	-	-	-	400	6.25	43.8	0.45	950	130	0.24	1200

### 7.3 RESULTS AND DISCUSSION

By sequentially coupling a transient thermal model and by using the meteorological data of Jeddah, Saudi Arabia, the simulation of four representative days was carried out. The details of the thermal model and the life prediction model have already been described in Chapter 3 and Chapter 6 respectively.

Fig. 7.4 gives four points selected on the PV module for which results are provided in Table 7.2. These points represent the critical points for each material and are at the interface of one another such that, A represents the point on cell at the interface of the interconnect, B represents the point on interconnect at the interface of the cell, C represents the point on glass at the interface of the encapsulant over the interconnect between cells and D represents point on back-sheet at the interface of the encapsulant below the interconnect region. Table 7.2 gives the von-Mises stress ( $\sigma_{\text{von}}$ ), the first principal stress ( $\sigma_1$ ) and the first principal total mechanical and thermal strain ( $\epsilon_{\text{total}}$ ) which is later used in the life-prediction model. These results are provided at the minimum ( $T_{\text{min}}$ ) and maximum temperature ( $T_{\text{max}}$ ) for the month of October (Day 3). For cells, glass and backsheet, there is almost no difference within the operating stresses of the PV module. It can be inferred that the stress in cells is way below its yield stress, thus operate within a safe range. The highest stress is found in glass as it bears most of its part to frame which restricts its thermo-mechanical motion. An interesting set of results is seen over the interconnect region (point B). Although the changes in stress using all of

the five encapsulants is insignificant, but the variation of first principal total mechanical and thermal strain ( $\epsilon_{\text{total}}$ ) can be seen. The interconnects operate over its yield stress of 94 MPa [11]. The Bilinear Isotropic hardening (BISO) model applied to copper causes large change in strains for small changes in stress. That is why a strain based life prediction model is used to find time to crack initiation in the copper interconnects. Copper faces a first principal stress difference of 97.5 MPa during the cycle of the day which is highest amongst all materials and makes it most vulnerable to fatigue. The strain it faces is almost more than 50 times higher than the other materials. An irregular pattern of strains is seen corresponding to their respective stresses as some of the encapsulants are so compliant that they help to provide free thermal motion.

Table 7.3 shows the von-Mises stress, first principal stress and the elastic strain in the encapsulant material for Day 3 at maximum and minimum temperature. Maximum stress can be seen in Ionomer whereas, TPU has the lowest stress. In the case of strains, PVB has induced maximum strain amongst all and on the contrary, Ionomer has the lowest strain.

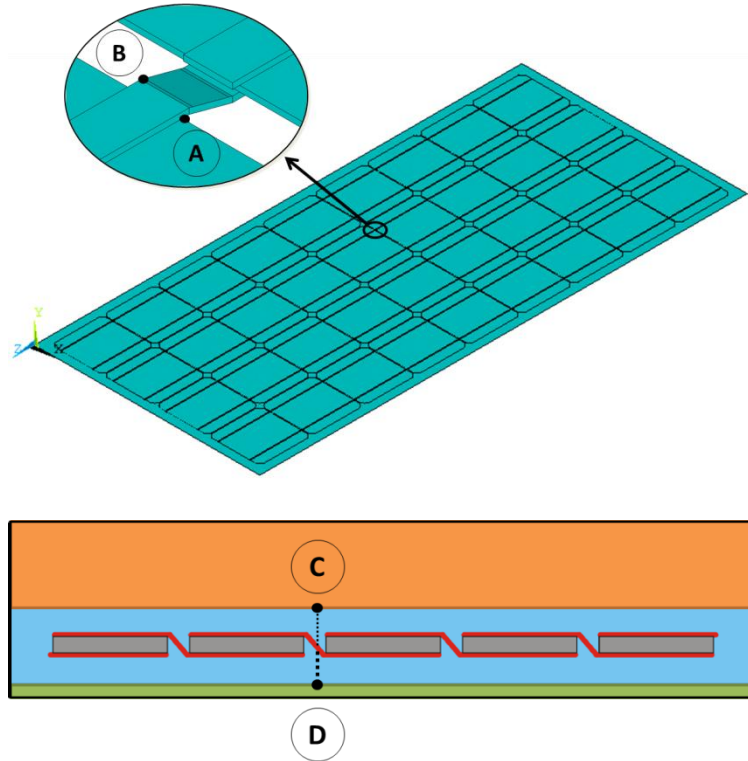


Fig. 7.4: Location of points A, B, C and D for Table 7.2

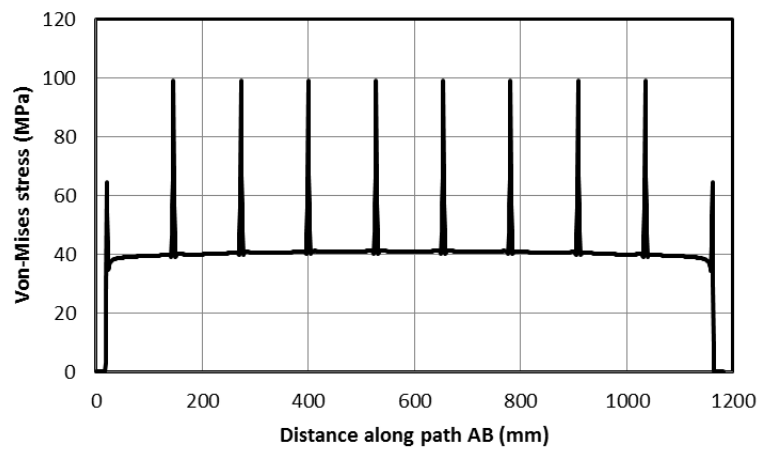
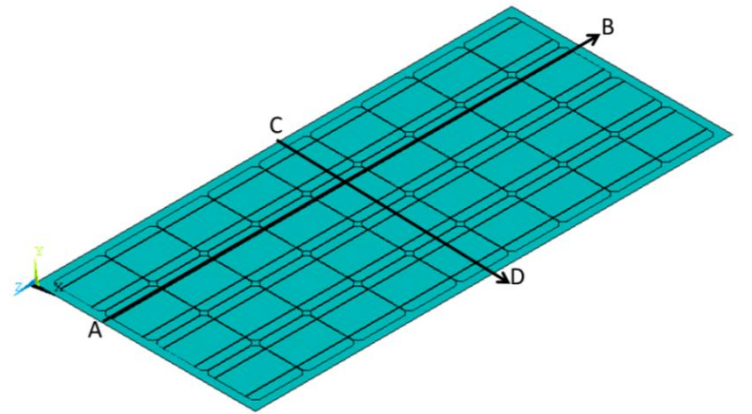
Table 7.2: Table comparing von-Mises stress, first principal stress and total first principal strain at four points (Fig. 7.4) using different encapsulants

	<b>A (Cell)</b>						<b>B (Interconnect)</b>					
	$T_{min}$			$T_{max}$			$T_{min}$			$T_{max}$		
	$\sigma_{von}$ (MPa)	$\sigma_1$ (MPa)	$\epsilon_{total}$	$\sigma_{von}$ (MPa)	$\sigma_1$ (MPa)	$\epsilon_{total}$	$\sigma_{von}$ (MPa)	$\sigma_1$ (MPa)	$\epsilon_{total}$	$\sigma_{von}$ (MPa)	$\sigma_1$ (MPa)	$\epsilon_{total}$
<b>EVA</b>	39.1	44.6	0.0004	37.9	39	0.00032	98.8	103.4	0.024237	14.6	5.9	0.017319
<b>PDMS</b>	38.9	44.4	0.0004	37.9	38.7	0.00032	98.8	103.4	0.018438	15.9	4.1	0.010273
<b>PVB</b>	39.2	44.6	0.0004	37.9	38.9	0.00032	98.8	103.5	0.036816	14.9	4.8	0.027388
<b>TPU</b>	37.8	38.8	0.0004	37.8	38.8	0.00032	98.8	103.4	0.020884	15.1	5.3	0.01117
<b>Ionomer</b>	39.9	46.1	0.0004	40.1	40.2	0.00033	98.8	102.9	0.012874	13.7	6.6	0.007092
	<b>C (Glass)</b>						<b>D (Backsheet)</b>					
	$T_{min}$			$T_{max}$			$T_{min}$			$T_{max}$		
	$\sigma_{von}$ (MPa)	$\sigma_1$ (MPa)	$\epsilon_{total}$	$\sigma_{von}$ (MPa)	$\sigma_1$ (MPa)	$\epsilon_{total}$	$\sigma_{von}$ (MPa)	$\sigma_1$ (MPa)	$\epsilon_{total}$	$\sigma_{von}$ (MPa)	$\sigma_1$ (MPa)	$\epsilon_{total}$
<b>EVA</b>	101	112.4	0.00027	64.5	69	0.00013	31.9	32.8	0.00038	19.2	19.5	0.0002
<b>PDMS</b>	101.1	112.6	0.00026	64.4	68	0.00013	31.9	32.8	0.00038	19	19.3	0.0002
<b>PVB</b>	101	112.4	0.00026	64.5	68.2	0.00013	31.9	32.8	0.00038	19.2	19.4	0.0002
<b>TPU</b>	101.1	112.6	0.00026	64.7	68.5	0.00013	31.9	32.8	0.00038	19.2	19.5	0.0002
<b>Ionomer</b>	100.1	110.4	0.00022	64.7	67.5	0.00011	31.9	32.7	0.00033	19.4	19.6	0.00017

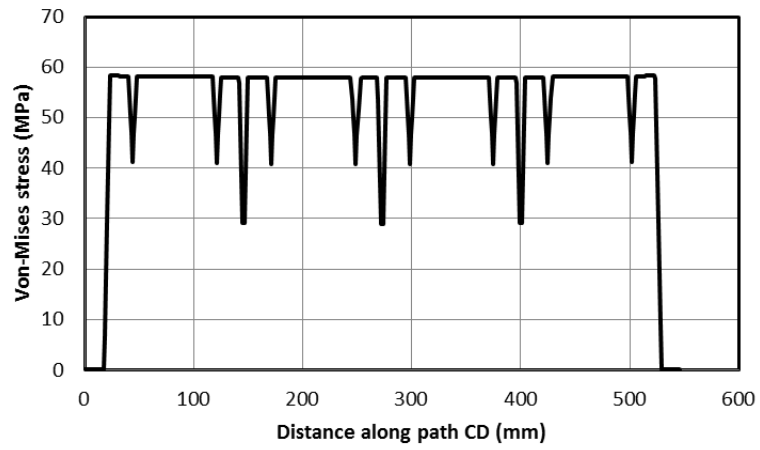
Table 7.3: Von-Mises stress, first principal stress and total strain comparison of encapsulants at minimum and maximum temperature for the month of October

	$\sigma_{\text{von}}$ (Pa)		$\sigma_1$ (Pa)		$\epsilon_{\text{el}}$		
	$T_{\min}$	$T_{\max}$	$T_{\min}$	$T_{\max}$	$T_{\min}$	$T_{\max}$	$T_{\min} - T_{\max}$
<b>EVA</b>	126320	29412	126180	29370	0.025565	0.014698	0.010867
<b>PDMS</b>	27588	16109	27270	16092	0.02812	0.016341	0.011779
<b>PVB</b>	91508	49578	91113	49559	0.052668	0.030548	0.02212
<b>TPU</b>	7465.6	1527.5	4554.9	1026.5	0.034514	0.020144	0.01437
<b>Ionomer</b>	8859800	5359700	8900800	5369900	0.016845	0.010135	0.00671





(a)



(b)

Fig. 7.5: Stress variation on module along (a): Longitudinal path *AB* and (b): Transverse path *CD* at lowest temperature on Day 3 using EVA as encapsulant

Fig. 7.5 provides the von-Mises stress variation over the PV module along longitudinal and transverse paths for Day 3. The shell layer considered includes the encapsulant, cells and interconnects between cells only. The same profile was seen for all of the five encapsulants with no significant stress difference which can be seen in Table 7.2. Thus, results are shown using EVA as the encapsulant material. The start of path *AB* includes the encapsulant material which is under minimum von-Mises stress of 0.12619 MPa. The maximum stress is found in the interconnect region and is around 100 MPa. Neglecting, the boundary edges of cells, they face an almost constant stress of 40 MPa. The same situation can be seen for the transverse path *CD* except that the stress in cells is increased to almost 58 MPa and it drops to 41 MPa below the interconnect region over it. In Chapter 5, the dominant effect of glass over other materials of the laminate has been discussed. The interconnects over the cells help to block the effect of glass thereby, reducing stress. The encapsulant between cells is stressed to 28 MPa as it has a little room of relief.

Fig. 7.6 gives the stress variation along the thickness of the module starting from backsheet to glass using EVA for Day 3. No variation of stress is seen along the thickness direction for each material except for glass. Glass also covers the maximum thickness within the laminate with a stress difference of almost 19 MPa. The interconnects above and below the cell share the same stress of 99 MPa. Cells constitute a lesser stress as its thermo-mechanical movement is aided through the compliancy of the encapsulant material.

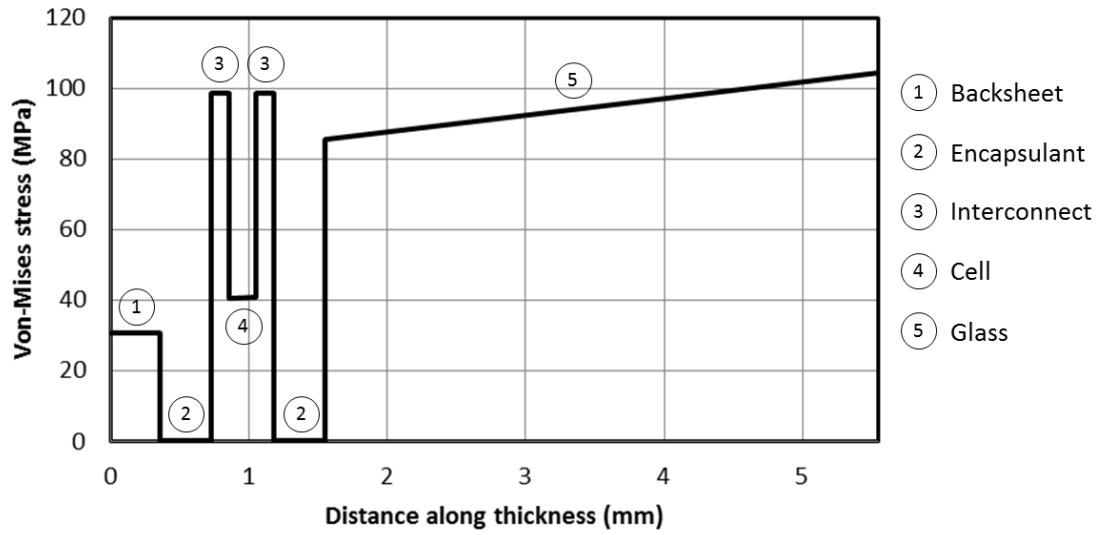


Fig. 7.6: Von-Mises stress variation along thickness of the PV module using EVA for  
Day 3

Table 7.4: Maximum power point voltage and current along with the efficiency of the  
PV module at 10:30 am during Day 3 for all encapsulants

Encapsulant	Max. Power Voltage $V_{mp}$ (V)	Max. Power Current $I_{mp}$ (A)	Efficiency $\eta_{pv}$ (%)
EVA	13.1105	4.5326	6.0867
PDMS	13.0198	4.5305	6.0752
PVB	13.08	4.5319	6.0716
TPU	13.0822	4.5322	6.0731
Ionomer	13.1186	4.5337	6.092

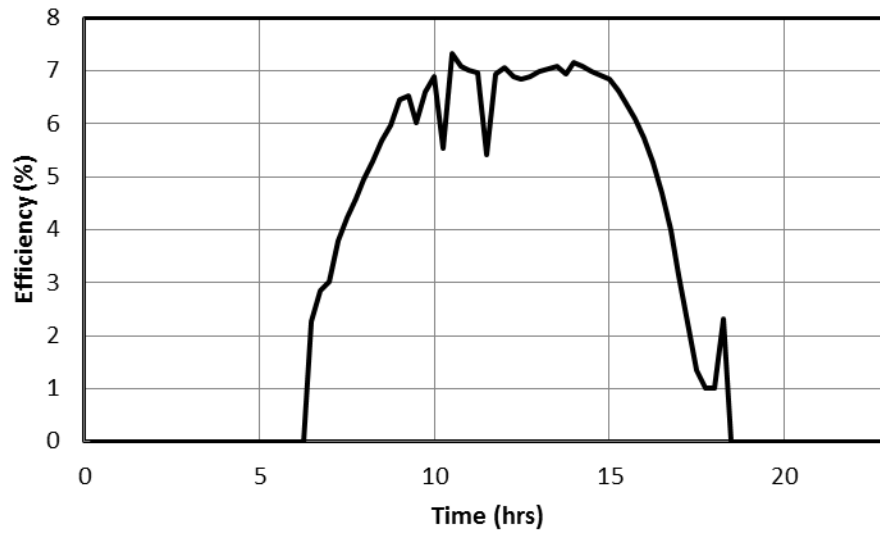


Fig. 7.7: Cell efficiency for the five encapsulants during Day 3

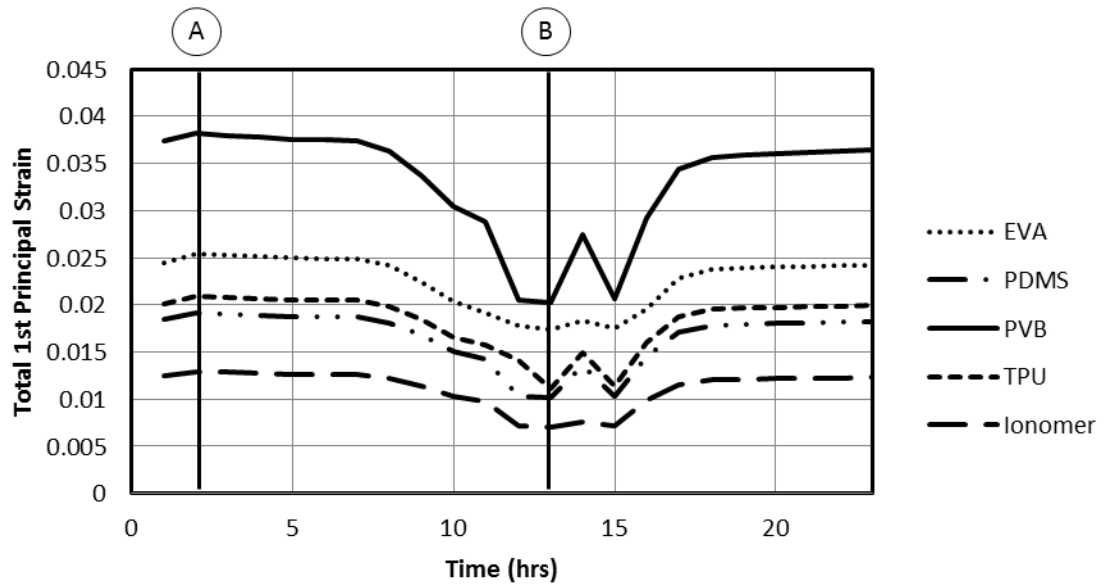


Fig. 7.8: Total first principal strain variation for all encapsulants during Day 3. Line A shows the time of maximum strain and Line B shows the time of minimum strain

Fig. 7.7 shows the cell efficiency during Day 3. No change in cell efficiency was found by changing the encapsulants in the model as shown in Table 7.4. The power output of cells depends on cell temperature. There was a very little cell temperature difference for all five encapsulants and was around 0.5 to 1 K which led to almost no change.

For life prediction, as in Chapter 6, the Basquin-Coffin-Manson relationship (Eq. (6.1)) was used to predict life or time to crack initiation in copper interconnects. The total strain amplitude was found out as the half the difference between the maximum and the minimum first principal strains. Fig. 7.8 gives the first principal strain variation of copper during Day 3. As it can be seen PVB provides the most strain change during the day, and thus provides the minimum time to crack initiation, shown by Table 7.5. Similarly, the strain change in Ionomer is the least providing it the maximum life. Line *A* and *B* provide the times of maximum and minimum strain respectively using all encapsulants. The factor providing a great variation in life of PV module (Table 7.5), is the compliancy of the encapsulant material. The instantaneous moduli of the encapsulants ( $E_o$ ) in Table 7.1 can be seen for comparison. PVB provides the highest value of modulus and thus resulting in minimum life. Similarly, the rest can be ranked by viewing it. The simulation, on the other hand, gives an idea of the extent to which the life has decreased or increased.

Table 7.5 also shows that the module is under worst condition during October, as suggested by the evaluated module lives. On the contrary, module life is best during December. It was found that the temperature range of the day during October and

December was maximum and minimum respectively. In order to find average life ( $L_{avg}$ ) for all encapsulants, weights were assigned according to Eq. (6.6). Eq. (6.7) was then used to get average life, as in Chapter 6. PDMS, TPU and Ionomer are seen to better than the commonly used EVA with respect to structural performance. Life, by using Ionomer, may be overestimated due to the usage of hyperelastic model although the material is viscoelastic.

If the life outcomes of Table 7.5 are compared with the strains of Table 7.3, it can be seen that the strain range of the encapsulant material has an impact on the life of PV module such that the larger the strain range, the lesser is the life. It is also seen that the life trend of encapsulants in October is different to that of the other months. For instance, EVA has a better life in October as compared to PDMS, but by comparing their results during other months, PDMS excels EVA in life. According to Fig. 7.8, the minimum temperature (and the maximum strain of copper interconnects) is reached at 2 am. As each encapsulant relaxes at a different rate, at this time, some of the encapsulants are harder as compared to other, and might be softer at a later time. For January, July and December, the minimum temperature is reached at 11 pm, 6 am and 7 am respectively during which each encapsulant is much relaxed than October's case. It was seen in Chapter 5, that glass dominates the thermo-mechanical movements and forces all components of the PV module to follow its contraction during cooling due to its large thickness as compared to others. The encapsulant material restricts glass's motion by binding it and thus, it can be seen that the encapsulants having a larger strain range provide minimum life of PV module.

Table 7.5: Life of PV module using different encapsulants

Month	Weights	Maximum Total 1 <sup>st</sup> Principal Strain $\epsilon_{\max}$	Minimum Total 1 <sup>st</sup> Principal Strain $\epsilon_{\min}$	Half of Strain Amplitude $\Delta\epsilon/2$	Life $L_{day\_i}$ (years)	Average Life $L_{avg}$
EVA						
January	0.003147	0.025931	0.019471	0.00323	28.94	26.63
July	0.004786	0.024076	0.017513	0.0032815	26.53	
October	0.003932	0.024237	0.017319	0.003459	20.18	
December	0.003065	0.026252	0.01993	0.003161	32.71	
PDMS						
January	0.003147	0.019624	0.01443	0.002597	130.7	80.09
July	0.004786	0.018343	0.012668	0.0028375	65.23	
October	0.003932	0.018438	0.010273	0.0040825	9.631	
December	0.003065	0.019753	0.014607	0.002573	141.7	
PVB						
January	0.003147	0.038653	0.031931	0.003361	23.4	14.37
July	0.004786	0.036376	0.027268	0.004554	6.361	
October	0.003932	0.036816	0.027388	0.004714	5.361	
December	0.003065	0.039481	0.03303	0.003226	29.17	
TPU						
January	0.003147	0.021466	0.015898	0.002784	74.88	46.98
July	0.004786	0.020064	0.013949	0.003058	39.86	
October	0.003932	0.020884	0.01217	0.004357	7.482	
December	0.003065	0.02161	0.01609	0.00276	80.1	
Ionomer						
January	0.003147	0.01323	0.008077	0.002577	140	111.6
July	0.004786	0.012371	0.007099	0.002636	115.1	
October	0.003932	0.012874	0.007092	0.002891	57.27	
December	0.003065	0.013317	0.008191	0.002563	146.7	

## 7.4 COMPARISON OF OTHER PROPERTIES

EVA, PDMS, PVB, Ionomer and TPU represent the currently used and prospective encapsulants for flat plate PV module. Michael Kempe [83] deals with the comparison of these encapsulants with respect to light transmittance, UV durability and electrical insulation. The stability of encapsulants in humid climates can be described by experiments performed in [68]. The properties have been summarized in Table 7.6. The glass transition temperature of an encapsulant is the reversible transition in polymer materials from a hard and relatively brittle state into a molten or rubber-like state. As the definition suggests, a polymer becomes stiff and its  $E$  is increased to one or two orders of magnitude when the operating temperature is lower than the  $T_g$ . The sudden change in modulus would make the encapsulant brittle and reduce its compliancy. Therefore, it is preferred that the  $T_g$  of the encapsulant should not lie within the operating range. As seen in Table 7.6, PDMS has a very low  $T_g$  and it is impossible for it to lie within the operating range of PV module thus makes it the best option. It can also be seen that the conventional EVA has its  $T_g$  in the operating range for PV modules working in cold regions.

As discussed earlier, encapsulant polymers must have the ability to be transparent to light so in order to achieve maximum power generation from cells. Table 7.6 provides the percentage transparency of encapsulants to light. Again, PDMS provides the highest transmittance with respect to other encapsulants. But as the difference is quite less



between one another, changing an encapsulant would not have a major effect on the efficiency of PV module which is also discussed in the previous section. The transmittance of an encapsulant is majorly affected by UV radiation in light as it may cause destruction of bonds within the encapsulant material thereby changing its color. It is well known that EVA turns to yellow/brown after few years of operation due to the same reason. Kempe [83] has performed accelerating aging tests on these encapsulants by exposing them to 42 UV suns at a temperature between 80 to 95°C. As shown in Table 7.6, PDMS samples showed no significant loss for up to 6000 hours of exposure. EVA and TPU showed loss of transmittance between 750 to 6000 hours. Ionomer was better than EVA and PVB had lost the most transmittance.

High electrical resistivity of a polymer prevents leakage current as well as electrochemical corrosion. It was also measured in [83] for all the five polymers in both dry and wet conditions. PVB was mostly affected by water due to its ability to absorb it. It showed least resistance in both dry and wet conditions. Other polymers were slightly affected by saturation. Ionomer and PDMS had almost the same resistance which was highest amongst all of the polymers.

The impact of moisture over PV module encapsulants can be measured by Water Vapor Transmission Rate (WVTR) and water concentration. WVTR is actually the measure of rate of moisture ingress into the PV module through encapsulant. Encapsulants with high WVTR are the reason for corrosion of the solder bonds and copper interconnects. Water concentration is actually the measure of the water absorption

ability of a polymer. Swonke and Auer [68] measured WVTR and water concentration of the encapsulants and found that Ionomer was the most suitable polymer with respect to moisture stability. PDMS has a high value of WVTR but did not absorb water. PVB had the highest water absorption with high WVTR and thus provides highest vulnerability. Rest of the polymers also had the ability to absorb and transmit water with their values mentioned in Table 7.6.

## 7.5 WEIGHTING AND RATING OF ENCAPSULANTS

In order to provide the best encapsulant for PV module with respect to its properties and outcomes of life, weighting and rating decision matrices have been utilized. The weighting matrix (Table 7.7) is used to determine the relative importance of the properties of encapsulants. Each property and outcome of encapsulant usage was compared over one another (by comparing the columns with the rows of Table 7.7). If a column property was considered to be more important than the row one, a “+” sign was entered in the relative position. Similarly, if it is considered less important, then a “-” was entered. The mentioned actions helped to assign weights to each property.

Table 7.8 gives the rating matrix for encapsulants. Each property was first ranked in a scale of 1 to 5. To do so, the property was first normalized by its respective maximum value and then multiplied by five. For example, life of PV module using PDMS encapsulant is mentioned to be 80.09 years. It is divided by the life of Ionomer (111.6 years) which is the maximum amongst all to give 3.59 out of 5. The computed ranking is

then multiplied by the respective weight of the property from Table 7.7, to give the final rank. At the end, sum of all the results for each encapsulant gives its final score which helps to decide the best encapsulant for PV module.

One of the most important properties of encapsulant is that it should be transparent to light. High transmittance leads to better power output and without it, a PV module is nonoperational. Thus, it was given the highest importance within Table 7.7. Next, the UV durability of the encapsulant is assessed. The importance of UV durability was set in accordance with experiments performed in [84]. The experiment involved the testing procedure prescribed by IEC 61215. The module was kept under illumination of not less than 1000 Lx and was then examined visually. Next, the DC dielectric insulation test was carried out under humidity. The yellowing/browning of the encapsulant was seen along with the corrosion along the busbar of the cells. This suggested delamination which is a cause of moisture. When the electric insulation properties were tested, it was found that there was no dielectric breakdown and the insulation was still within the limits of IEC 61215. With the introduction of Ce-doped glass, as given in [85], much of the UV light is filtered and hence structural life of copper interconnects is preferred over it. In addition to this the breakage of copper interconnects has been attributed to thermal cycling and has a percentage of being a reason of warranty returns as reported in [24]. High WVTR is responsible for allowing water to enter the PV module which causes corrosion of the inner components and ultimately failure. But a more hazardous property of the encapsulant is water absorption or concentration as it enables the encapsulant, such as EVA, to react with moisture to form acetic acid [20] which speeds up the corrosion

process. The properties related to the resistance to moisture ingress thus have a high weightage. Finally, it is seen that the cost has also been given a large importance as now-a-days technology changes are fast, and it is preferred to have cheap technology with satisfactory results, so that it can be replaced easily with a more advance alternative within a few years.

Table 7.8 gives the rating matrix and it suggests Ionomer to be the best option amongst all encapsulants. Although its life is somewhat over-estimated due to the usage of hyperelastic properties but it is good in other properties as well. EVA is next as it gains a heavy score due to its low cost. EVA is also better than TPU and PVB in other aspects. PDMS provides the best properties, but its high cost lags it behind.

## **7.6 CONCLUSIONS**

In this work, thermal, structural and life prediction models were coupled to find life of the PV module using five different encapsulants. EVA, PDMS, TPU, PVB and Ionomer were modeled and their respective results were discussed and compared. The comparison also includes the findings in the literature to draw out the following conclusions:

- Changing the encapsulant material of PV module has an insignificant effect on the stresses of its components.
- More PV module life is observed by using encapsulants with least strain range.
- The efficiency of the PV module is not affected by changing its encapsulant.

- Maximum life of PV module is predicted when Ionomer is used as an encapsulant. On the contrary, the usage of PVB gives minimum life.
- Ionomer is seen to be as the best encapsulant for PV modules as it provides a mix of good properties at a reasonable cost as rated by the decision matrix.

Table 7.6: Comparison of encapsulant properties in the literature

<div>Properties</div> <div>Encapsulants</div>	Glass Transition Temperature ( $T_g$ ) [86]	Light Transmittance [83]	UV Durability (under 42 suns at 6000 hours of exposure) [66]	Volume Resistivity/Electrical Insulation (Ohm-cm) [66]	Moisture Ingression [68]
<b>EVA</b>	-16°C	93.9 %	Significant degradation during 750 to 6000 hours	$10^{14}$	WVTR = $115 \text{ g}^{-1}\text{d}^{-1}$ ; Water concentration is lower than TPU
<b>PDMS/Silicone</b>	<-100°C	94.5 %	No significant loss	$10^{16}$	WVTR = $310 \text{ g}^{-1}\text{d}^{-1}$ ; No water concentration
<b>PVB</b>	35°C	93.9 %	Poor performance	$10^{12}$ (Dry) $10^{10}$ (Wet)	WVTR = $310 \text{ g}^{-1}\text{d}^{-1}$ ; Highest water concentration
<b>TPU</b>	21°C	93.3 %	Significant degradation during 750 to 6000 hours	$10^{14}$	WVTR = $510 \text{ g}^{-1}\text{d}^{-1}$ ; Water concentration is lower than PVB
<b>Ionomer</b>	69°C	92.3 %	Better than EVA with some loss	$10^{16}$	WVTR = $55 \text{ g}^{-1}\text{d}^{-1}$ ; No water concentration

Table 7.7: Weighting matrix for properties of encapsulant and its outcomes

<b>ID</b>	<b>CRITERIA</b>	<b>A</b>	<b>B</b>	<b>C</b>	<b>D</b>	<b>E</b>	<b>F</b>	<b>G</b>	<b>TOTAL</b>	<b>WEIGHT</b>
<b>A</b>	Light Transmittance		+	+	+	+	+	+	6	0.286
<b>B</b>	UV Durability	-		+	-	-	-	-	1	0.048
<b>C</b>	Electrical Insulation	-	-		-	-	+	-	1	0.048
<b>D</b>	WVTR	-	+	+		-	+	-	3	0.143
<b>E</b>	Water Absorbance	-	+	+	+		+	-	4	0.190
<b>F</b>	Structural Life	-	+	-	-	-		-	1	0.048
<b>G</b>	Cost	-	+	+	+	+	+		5	0.238
<b>TOTAL</b>									21	1

Table 7.8: Rating matrix for encapsulants by scaling properties on a scale of 1 to 5 and the weights from Table 7.7

CRITERIA	WEIGHT	RATING					WEIGHTED RATING				
		EVA	PDMS	PVB	TPU	Ionomer	EVA	PDMS	PVB	TPU	Ionomer
Light Transmittance	0.286	4.97	5	4.97	4.93	4.88	1.421	1.430	1.421	1.410	1.396
UV Durability	0.048	3	5	1	3	4	0.144	0.240	0.048	0.144	0.192
Electrical Insulation	0.048	0.05	5	0	0.05	5	0.002	0.240	0.000	0.002	0.240
WVTR	0.143	2.42	0.9	0.9	0.54	5	0.346	0.129	0.129	0.077	0.715
Water Absorbance	0.190	3	5	1	2	5	0.570	0.950	0.190	0.380	0.950
Structural Life	0.048	1.19	3.59	0.64	2.07	5	0.057	0.172	0.031	0.099	0.240
Cost	0.238	5	0.75	3.33	2.58	2.58	1.190	0.179	0.793	0.614	0.614
<b>TOTAL</b>							3.731	3.340	2.611	2.727	4.347



## **CHAPTER 8**

### **SENSITIVITY ANALYSIS**

Sensitivity analysis is used in a wide range of problems whether they are statistical or engineering in nature. It actually deals with the influence of certain input parameters over a model. For engineering applications, it has been greatly used in design optimization, system identification and statistical structural analysis [87]. The purpose of this analysis is to determine the sensitivity coefficients that are estimated through partial derivatives of the considered function related to a certain input parameter, where the derivatives are obtained from algebraic equations or numerical procedures such as Finite-Element (FE) Analysis [88].

In this chapter, sensitivity analysis is used to detect the influence of the encapsulant material over the structural, thermal and electrical performance of PV module. The constitutive, thermal and geometrical properties of the encapsulant are slightly varied and results are reported by performing FE analysis by taking Ethylene-Vinyl Acetate (EVA) as the nominal case. The measurement of the results is done through electrical and

structural performance of the PV module. The models used have been explained in the previous chapters.

## 8.1 BASIC PRINCIPLES OF SENSITIVITY ANALYSIS

Much of the principles of sensitivity analysis have been discussed in [89]. To find the influence of an input to the model over its outcomes, let us consider the independent input variables as  $X_i$ . The vector  $X$  denotes the set of these variables.

$$X = \bar{X} \pm U_X \quad (8.1)$$

In Eq. (8.1),  $\bar{X}$  denotes the nominal value of the independent variable and the  $U_X$  is the small change about the nominal value. The range of  $U_X$  is selected such that the nominal value, if changed with  $U_X$ , lies within the real domains of the problem. Or it can be said that the change may be quite less i.e. from 5% to 10% of the nominal value.

$$Y = Y(X_1, X_2, \dots, X_N) \quad (8.2)$$

$Y$  defines the output variable in Eq. (8.2), such that it is a function of all the input variables  $X_i$ . Therefore the uncertainty in the output variables is linked with the uncertainty of the input variables by Eq. (8.3).

$$U_Y = \frac{dY}{dX} U_X \quad (8.3)$$

To include the effect of the uncertainties of all the input parameters, the uncertainty in Y can be expressed in terms of the root sum square values given in Eq. (8.4). The uncertainties are normalized as provided by Eq. (8.5).

$$U_Y = \left[ \sum_{i=1}^N \left( \frac{\partial Y}{\partial X_i} U_{X_i} \right)^2 \right]^{1/2} \quad (8.4)$$

$$\left( \frac{U_Y}{\bar{Y}} \right) = \left\{ \sum_{i=1}^N \left[ \left( \frac{\partial Y}{\bar{Y}} \frac{\bar{X}_i}{\partial X_i} \right) \left( \frac{U_{X_i}}{\bar{X}_i} \right) \right]^2 \right\}^{1/2} \quad (8.5)$$

One of the terms in Eq. (8.5) is the Normalized Sensitivity Coefficient (NSC) which is used to compare the impact of an input over a models output. This term is given in Eq. (8.6).

$$NSC_{X_i} = \left( \frac{\partial Y}{\bar{Y}} \frac{\bar{X}_i}{\partial X_i} \right)^2 \quad (8.6)$$

The whole idea of sensitivity analysis is portrayed by Fig. 8.1 and Fig. 8.2. The first figure gives the nominal case where a certain set of inputs is provided to the program to compute the output. In order to find the influence of an input, it is varied to add the uncertainty in the output. This way, each input is changed to a small amount keeping others constant and the results are recorded and analyzed.

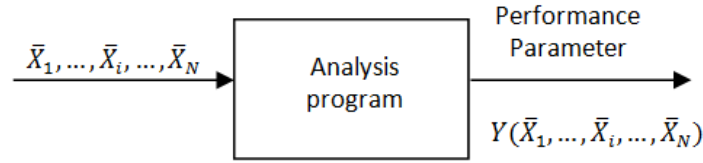


Fig. 8.1: Nominal system [90]

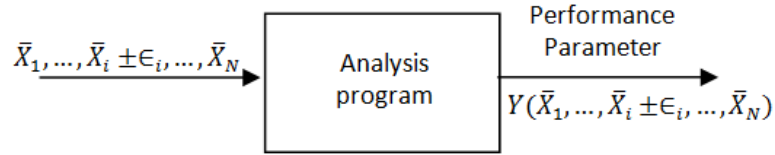


Fig. 8.2: Perturbed system [90]

## 8.2 SELECTION OF INPUT AND OUTPUT PARAMETERS

The selection of the varying input parameters are based on the constitutive, thermal and geometrical properties of the encapsulant. In Chapter 7, five encapsulants were studied and compared. It was seen that EVA is mainly used due to its cost effectiveness and satisfactory properties. Thus, EVA is selected to see what parameters are important to consider in the selection of an encapsulant for PV module. Chapters 2 and 3 deal with the viscoelastic modeling of EVA where Eq. (3.15) provided its relaxation modulus. The relaxation curve of EVA on a time scale can be divided into three portions, glassy, viscoelastic and rubbery as given by Fig. 8.3. The glassy region represents the rigid state of EVA where it is brittle. The viscoelastic region represents the degradation of the

viscosity of the material. The decrease in viscosity is seen in the form of decrease in modulus of the material. It is the region where the glass transition temperature of the polymer lies. After the glass transition, the material goes on to its rubbery state where the polymer is flexible and soft. By seeing the time scale it can be inferred that the PV module operates when the encapsulant is in viscoelastic mode. After all of these, the polymer comes in the flow region (not shown) which represents its degradation by showing its ability to flow.

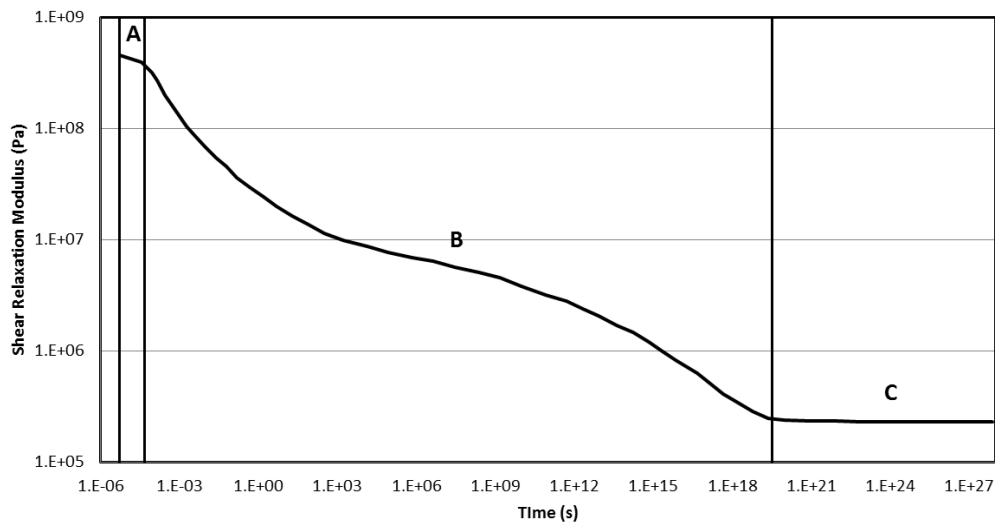


Fig. 8.3: Division of relaxation curve of EVA on time scale where A represents the glassy region, B represents the viscoelastic region and C is the rubbery region.

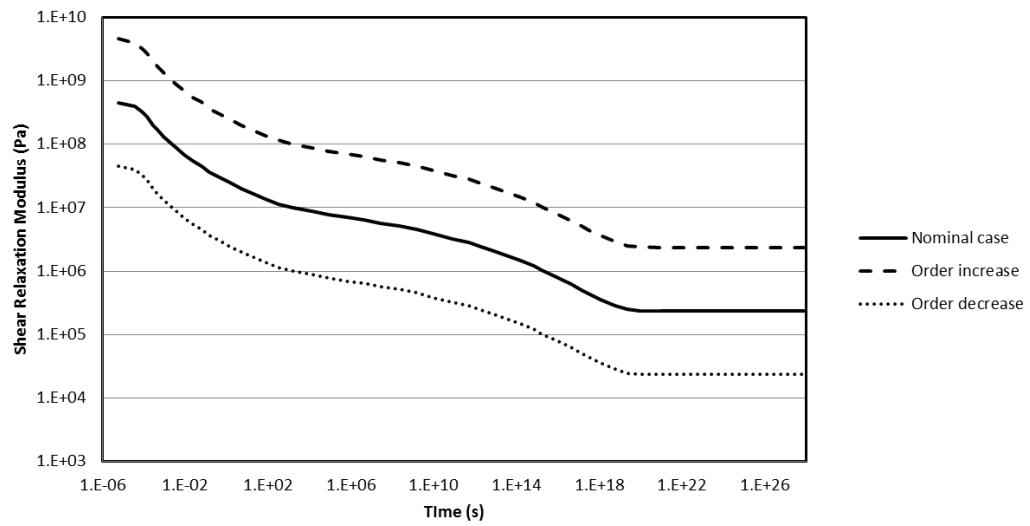


Fig. 8.4: Relaxation modulus of EVA at -20°C varying by one order

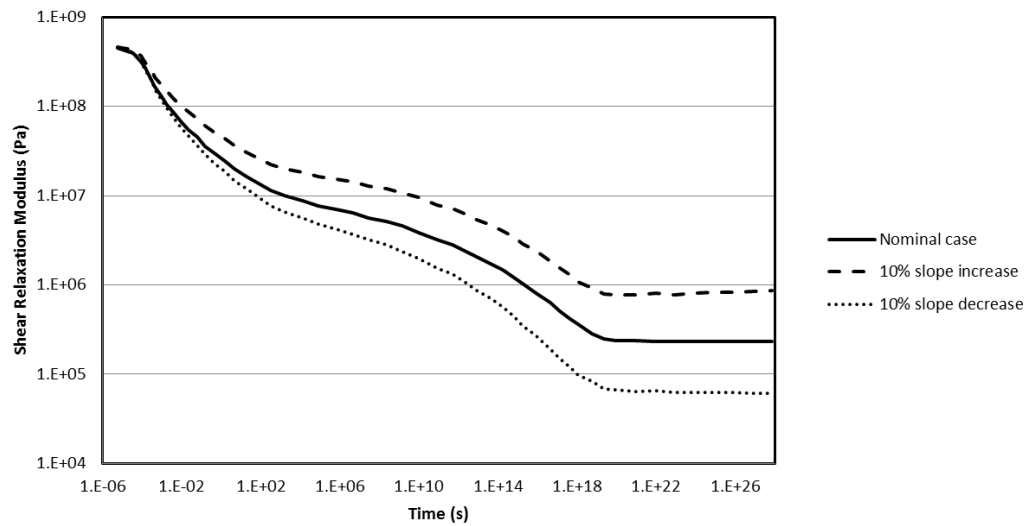


Fig. 8.5: Relaxation modulus of EVA at -20°C with varying slope by 10%

The term  $G_o$  in Eq. (3.15) is the instantaneous modulus of EVA. Or it can be said that it gives the start of the relaxation curve in time-temperature domain while keeping its shape constant. This is explained by Fig. 8.4, where  $G_o$  was varied by an order from the nominal case. Another property ought to change is the overall slope of this curve. The slope change provides the rate at which EVA degrades its relaxation modulus. It is done by varying the modulus at each time with the variation increasing from its previous value as advancing over the time scale. The overall percentage increase or decrease is measured by the sum of nominal and changed moduli and was kept up to 10%.

Next focus is towards the thermal properties of an encapsulant material. The thermal conductivity is an important factor as it will define the ability of a polymer to dissipate heat from cells to the environment. It is well known that the heating of cells is a cause of its decrease in efficiency. Similarly, by increasing the density and specific heat of the encapsulant, the conductive heat loss of the encapsulant increases thereby barring heat from reaching the cells. The effect of CTE has a direct impact on the structural life of PV module as it will define the inter-structural movements in the module where each component is constrained onto one another. The only geometric parameter which was varied was the thickness of the encapsulant.

The outputs are measured in the form of structural and electrical performance. Structural performance is measured through the life prediction formula given by Eq. (6.1) in Chapter 6 which is actually the time to crack initiation in the copper interconnects. Power output is measured by the procedure described in Chapter 7 by the electrical

model. The selection list is summarized by Table 8.1 with their nominal values. The life mentioned is for the month of July. This month is chosen as it has the most weightage over others. The overall life is not calculated in order to save simulation time. There is no value for the slope of the relaxation curve as it directly gives the fractional percentage  $\left(\frac{\partial X}{X}\right)$  through the procedure described before and is equal to 0.1. This fractional percentage is directly used in Eq. (8.6).

Table 8.1: List of variables with their nominal values selected for sensitivity analysis

<b>Inputs</b>	
<b>Variable</b>	<b>Nominal value</b>
Instantaneous shear modulus ( $G_o$ )	464 MPa
Slope of relaxation curve	-
Coefficient of thermal expansion (CTE)	$270 \times 10^{-6} \text{ K}^{-1}$
Thermal conductivity ( $k$ )	0.311 W/m K
Specific heat ( $C$ )	2090 J/kg K
Density ( $\rho$ )	$960 \text{ kg/m}^3$
Thickness ( $t$ )	1.2 mm
<b>Outputs</b>	
Life in July	26.53 years
Power output	53.7095 W



### 8.3 RESULTS AND DISCUSSION

To calculate power output, thermal model for PV module developed in Chapter 6 was utilized to find the cell temperatures. The cell temperatures along with the absorbed solar radiation were used in the electrical model in Chapter 7 for power output. Similarly, for life estimation, maximum and minimum total principal strains were evaluated for copper interconnects and were deployed in Eq. (6.1). For the month of July, it was seen that the maximum efficiency is at 10:30 am, so the results reported in Table 8.2 are at this time. By using the nominal values in Table 8.1 and input and output variables from Table 8.2, NSC was evaluated in terms of power output and life.

The instantaneous shear modulus was changed by one order (Fig. 8.4) and its exponents were utilized in the Eq. (8.6). In the case of life, it is seen that it provides the minimum impact amongst all parameters. It is clear that the life prediction formula deals with the half of strain amplitude, and the amplitude remains almost same in all the three cases. The change in instantaneous shear modulus, definitely, will not have an impact on cell temperature. Therefore, no change in power output is observed to give NSC a value of zero.

Table 8.2: Outputs and NSCs with respect to varying input properties of EVA encapsulant

Property	Variation	Input $X_i \pm \Delta X_i$	Total Strain		Life $Y_1 \pm \Delta Y_1$ (years)	Cell Temperature $T_c$ (K)	Power $Y_2 \pm \Delta Y_2$ (W)	NSC (Eq. (8.6))	
			$\epsilon_{\max}$	$\epsilon_{\min}$				wrt Life	wrt Power
$G_o$	+ Order change	4640 MPa	0.024192	0.017512	26.51	329.265	53.7095	$9.09 \times 10^{-6}$	0
	- Order change	46.4 MPa	0.02419	0.017511	26.53	329.265	53.7095		
Slope	+10%	-	0.026919	0.01981	20.03	329.265	53.7095	19.38	0
	-10%	-	0.021896	0.015452	31.71	329.265	53.7095		
CTE	+10%	$297 \times 10^{-6} \text{ K}^{-1}$	0.026505	0.019192	45.09	329.265	53.7095	26.38	0
	-10%	$243 \times 10^{-6} \text{ K}^{-1}$	0.021882	0.015835	17.84	329.265	53.7095		
$k$	+10%	0.3421 W/m K	0.024192	0.017486	26.03	329.218	53.7270	0.031	$1.3 \times 10^{-5}$
	-10%	0.2799 W/m K	0.024192	0.017486	26.96	329.322	53.6882		
$C$	+10%	2299 J/kg K	0.024193	0.017522	26.36	329.203	53.7326	0.0039	$1.86 \times 10^{-5}$
	-10%	1881 J/kg K	0.024191	0.017503	26.69	329.327	53.6863		
$\rho$	+10%	1056 kg/m <sup>3</sup>	0.024193	0.017522	26.36	329.203	53.7326	0.0039	$1.86 \times 10^{-5}$
	-10%	864 kg/m <sup>3</sup>	0.024191	0.017503	26.69	329.327	53.6863		
$t$	+10%	1.32 mm	0.024196	0.017513	26.46	329.251	53.7147	0.0008	$9.55 \times 10^{-7}$
	-10%	1.08 mm	0.024188	0.017513	26.61	329.279	53.7042		

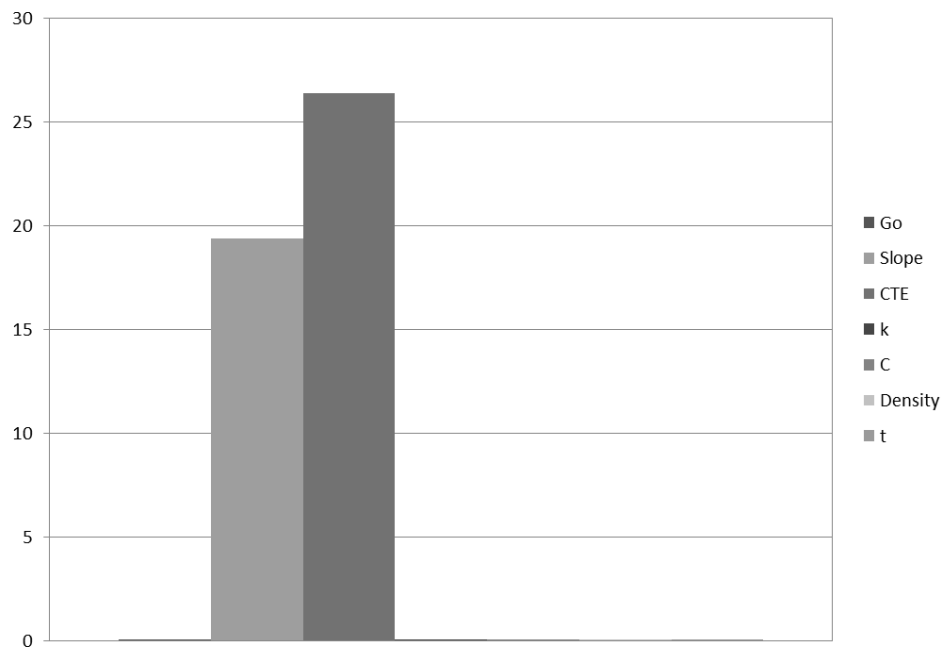


Fig. 8.6: NSC evaluated through life output varying encapsulant parameters

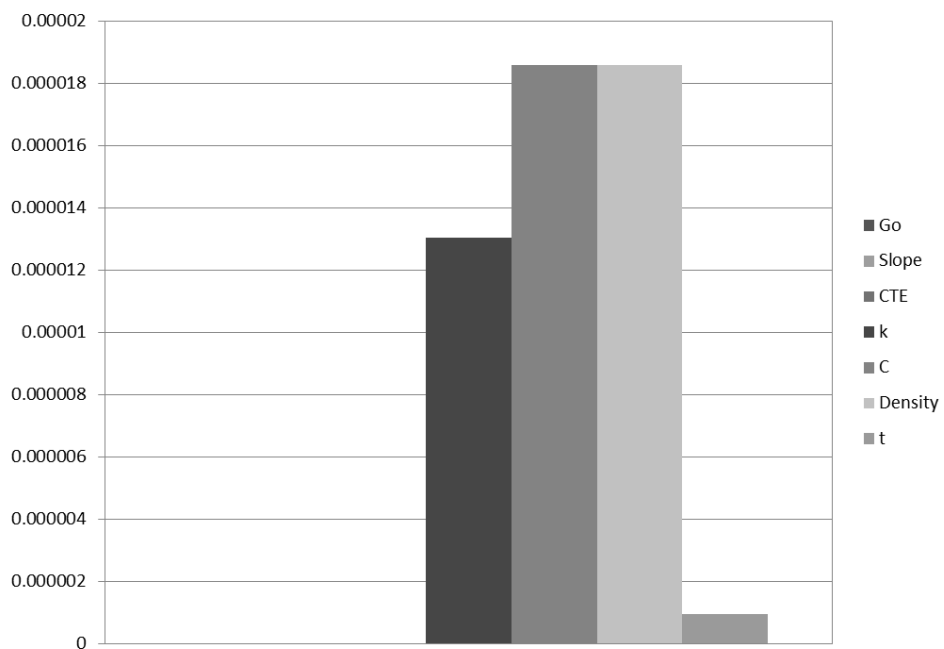


Fig. 8.7: NSC evaluated through power output varying encapsulant parameters

With the results of the instantaneous shear modulus, it was clear that the slope of the relaxation curve will have an impact on life. As in chapter 7, greater strain change of encapsulant led to lesser life of the interconnects. Thus, when the slope was increase by 10%, life reduced from 26.53 years to 20.03 years. Likewise, the decrease in slope increased life up to 31.71 years. This large impact gave it a NSC of 19.38, which is the second highest of all parameters. No change in power output is observed.

The CTE played a significant role in the case of life (Fig. 8.6). Larger the CTE, more inter-structural movement is bound to occur, leading to larger strains in all components of the PV module. The  $\pm 10\%$  change has altered life between 17.84 and 45.09 years. This is the highest change seen with respect to all the parameters thereby giving it the largest NSC of 26.38. It is evident that there no change in power output.

Thermal conductivity was altered in a similar fashion. Quite a little change in power output is seen. With a higher conductivity, the encapsulant provides a path to dissipate heat from cells. But this gives a negligible amount of improvement along with a NSC of  $1.3 \times 10^{-5}$  in terms of power output. The alteration in thermal conductivity gave an opposite impact in the case of life. Life was increased to 26.96 by decreasing 10% decrease in  $k$ . Although NSC in the case of life was as little as 0.031, but still it is better than the power output. The decrease in thermal conductivity causes lesser change in strain by trapping heat with the interconnects.

When the specific heat and density were altered by 10%, both of them gave the same impact for both life and power output cases. It gave the highest NSC with respect to

power output and equal to  $1.86 \times 10^{-5}$ , displayed in Fig. 8.7. The concept and outcomes are the same as seen in the thermal conductivity case. The increase in both of them provides a thermal barrier, thus restraining increase in cell temperature. The encapsulant can absorb greater amount of heat. On the other hand, life displays a reverse influence.

The increase in thickness decreases life and increases the power output. As the encapsulant material is increased, the effect of glass dominance decreases, which increases the inter-structural motion between glass and the interconnects. The NSC in the case of electrical power is  $9.55 \times 10^{-7}$  which is so little that the change can be neglected.

## 8.4 CONCLUSIONS

In this study sensitivity analysis was performed over certain parameters of the encapsulant material to find its effect on the life and the power output of PV module. EVA was kept as the nominal case as it is the least expensive and the most widely used encapsulant. The constitutive and thermal properties were slightly altered to draw out the following conclusions:

- The slope of the relaxation curve and the CTE of the encapsulant have a dominant influence over the life of the PV module with CTE proving the highest impact.
- There is negligible improvement or loss in the electrical efficiency of PV module. Thus, it can be inferred that the encapsulant material has an insignificant effect over the electrical performance of PV module.

## **CHAPTER 9**

### **CONCLUSIONS AND RECOMMENDATIONS**

A comprehensive finite element model for PV module was developed in this work. The model included viscoelastic modeling of the encapsulant material along with the orthotropic modeling of silicon cells. The concept of layered-shell modeling was utilized to reduce problem size, so that the effect of whole 36 cell PV module, incorporating the interconnects, could be analyzed. The model was validated by an experiment in the literature. This model was utilized to simulate ASTM E1171-09 qualification test and copper interconnects were found to go under low-cycle fatigue. In the next study, a life prediction model for the copper interconnects was used in conjunction with a thermal model. The meteorological conditions of Jeddah, Saudi Arabia were used to predict PV module life under actual operating loads. In the third study, five different encapsulants were incorporated in the model and their effects on structural and electrical performance were studied. In the last study, sensitivity analysis was performed to find the most important properties of the encapsulant material that would impact on the overall performance of PV module. The following conclusions and recommendations have been drawn out:

- Viscoelastic model for EVA is a close estimate of its constitutive behavior, unlike the linear elastic model which gave a large deviation from actual behavior.
- Shell model is able to capture the response of PV module over loads as the solid model. It also provides conservative estimates, useful for design purposes.
- Glass exhibits a dominant character towards the contraction of the module. It forces all components to follow its pure thermal contraction.
- Stress in cells is higher along the interconnect region as:
  - They are both directly tied to one another.
  - Interconnect hardens as it undergoes plasticity.
- Parametric study shows that 1.2 mm is the optimum encapsulation thickness.
- Interconnects undergo plasticity just after curing of the laminate. This hardens it thereby increasing the risk of breakage owing to fatigue induced during thermo cycles of day and night.
- The high failure percentage of copper interconnects, given in the literature, is justified through its yielding during cooling after the lamination process. The plastic deformation makes it a subject of low-cycle fatigue.
- Glass exhibits a dominating behavior due to its large thickness. As the contraction of glass was fixed, all the components of PV module had induced tensile stress within them. Glass also bears the maximum stress amongst all of

the components as it bears most of the area of constraint, which was made to provide frame effect.

- Silicon cells operate within a safe temperature range as the stresses induced in it are way below their yield stress; unless they are pre-cracked due to their manufacturing and soldering procedure.
- Average life of a PV module has been estimated through its dominant mode of failure which is the breakage of copper interconnects and is found out to be 26.63 years. It is quite close to the 25-year warranty given by most of the PV module manufacturers.
- The numerical procedure developed is inclusive of operating conditions and has the ability to predict proper behavior and life under operation with reasonable accuracy. It can be used as a tool to anticipate the effects of design changes in the form of stress distribution and life.
- Changing the encapsulant material of PV module has an insignificant effect on the stresses of its components.
- More PV module life is observed by using encapsulants with least strain range.
- The efficiency of the PV module is not affected by changing its encapsulant.
- Maximum life of PV module is predicted when Ionomer is used as an encapsulant. On the contrary, the usage of PVB gives minimum life.
- Ionomer is seen to be as the best encapsulant for PV modules as it provides a mix of good properties at a reasonable cost as rated by the decision matrix.



- The slope of the relaxation curve and the CTE of the encapsulant have a dominant influence over the life of the PV module with CTE proving the highest impact.
- There is negligible improvement or loss in the electrical efficiency of PV module. Thus, it can be inferred that the encapsulant material has an insignificant effect over the electrical performance of PV module.
- This model could be used to predict life of PV module under direct mechanical loading due to wind, hail and snow. The loads of ASTM E1830-09 can be utilized. An optimized design for frame can be proposed under such loading conditions.
- In the future, there is also a possibility to incorporate this model with dust accumulation and moisture ingress models to predict PV module overall performance.
- The model can be utilized to assess the effect of concentration over PV module.

## NOMENCLATURE

$A(T(t))$	WLF shift function
$A_i$	Anisotropy index
$A_{panel}$	Front area of the PV module ( $m^2$ )
$b$	Fatigue strength exponent
$C$	Specific heat capacity (J/kg.K)
$C_1, C_2$	Calibration constants for WLF shift function ( $C_2$ is in K)
$C_p$	Specific heat capacity (J/kg.K)
$c$	Fatigue ductility exponent.
$c_{01}, c_{10}$	Material constants for deviatoric deformation (Pa)
$[D]$	Material properties matrix (Pa)
$d$	Material incompressibility parameter ( $Pa^{-1}$ )
$E_o$	Instantaneous Young's modulus (Pa)
$E_x$	Young's modulus of elasticity in $x$ -direction (Pa)
$G$	Horizontal plane solar radiation ( $W/m^2$ )
$G_i$	Shear modulus of $i^{th}$ spring-damper in Maxwell's model (Pa)
$G_o$	Instantaneous shear modulus (Pa)

$G_{\infty}$	Long term shear modulus (Pa)
$G_{xy}$	Shear modulus in $xy$ -plane (Pa)
$h$	Heat loss coefficient (W/m <sup>2</sup> .K)
$\bar{I}_1$	First deviatoric strain invariant
$\bar{I}_2$	Second deviatoric strain invariant
$K$	Initial bulk modulus (Pa)
$K_{\tau\alpha}$	Incidence angle modifier
$k$	Thermal conductivity (W/m.K)
$L_{avg}$	Average life (years)
$L_F$	Final length (m)
$L_{day\_i}$	Life of the $i^{th}$ representative day (years)
$L_O$	Initial length (m)
$M$	Air mass modifier
$N_f$	No. of cycles to crack initiation
$Q$	Volumetric heat generation (W/m <sup>3</sup> )
$q$	Heat conduction (W)
$R$	Relaxation modulus (Pa)
$R_{beam}$	Ratio of beam radiation on tilted plane to that on horizontal plane
$[S]$	Compliance matrix (1/Pa)
$S$	Absorbed solar radiation (W/m <sup>2</sup> )

$s_{11}, s_{12}, s_{44}$	Components of compliance matrix (1/Pa)
$t$	Current time (s)
$T$	Current temperature (K)
$T_{amb}$	Ambient temperature (K)
$T_s$	Surface temperature (K)
$T_{\max\_i}$	Maximum ambient temperature of day $i$ (K)
$T_{\min\_i}$	Minimum ambient temperature of day $i$ (K)
$T_o$	Initial temperature (K)
$T_r$	Reference temperature of WLF shift function (K)
$T_{ref}$	Stress-free temperature or initial temperature (K)
$T_{total}$	Total of average ambient temperatures (K)
$U_X$	Uncertainty in $X$
$U_Y$	Uncertainty in $Y$
$V_{pvcell}$	Volume of the cells in the PV module (m <sup>3</sup> )
$W$	Strain energy potential (J/m <sup>3</sup> )
$W_{day\_i}$	Weight of the $i^{\text{th}}$ representative day
$W_{total}$	Sum of the weights of four representative days
$X$	Input variable for sensitivity analysis
$Y$	Output variable of sensitivity analysis

### ***Greek Symbols***

$\alpha_x$	Thermal expansion coefficient in $x$ -direction ( $K^{-1}$ )
$\beta$	Tilt angle
$\eta_{pv}$	Electrical efficiency of PV module
$\mu_i$	Initial shear modulus (Pa)
$\frac{\Delta \epsilon}{2}$	Total strain amplitude
$\epsilon_{\max}$	Maximum total principal strain
$\epsilon_{\min}$	Minimum total principal strain
$\epsilon_x$	Total strain in $x$ -direction
$\epsilon'_f$	Fatigue ductility coefficient
$\frac{\Delta \epsilon_p}{2}$	Plastic strain amplitude
$\{\epsilon\}$	Total strain vector
$\{\epsilon^{el}\}$	Elastic strain vector
$\{\epsilon^{th}\}$	Thermal strain vector
$\phi_i$	Relative modulus of $i^{\text{th}}$ spring-damper in Maxwell's model
$\gamma_{xy}$	Shear strain in $xy$ -plane
$\nu_{xy}$	Poisson's ratio in $xy$ -plane

$\rho$	Density (kg/m <sup>3</sup> )
$\rho$	Reflectivity of the ground
$\{\sigma\}$	Stress vector (Pa)
$\frac{\Delta\sigma}{2}$	Stress amplitude (Pa)
$\sigma'_f$	Fatigue strength coefficient (Pa)
$\sigma_{\max}$	Maximum principal stress (Pa)
$\sigma_{\min}$	Minimum principal stress (Pa)
$\sigma_x$	Stress in $x$ -direction (Pa)
$\sigma_{xy}$	Shear stress in $xy$ -plane (Pa)
$\sigma_{von}$	Von-Mises stress (Pa)
$\sigma_1$	First principal stress (Pa)
$\tau_i$	$i^{\text{th}}$ term for pseudo time (s)

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